

Calorimetric Measurement of the SOX Neutrino Source Activity

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**Genova, colonia di gabbiani blocca l'aeroporto:
stop ai voli**

Genova, une colonie de mouettes a bloqué les vols

PER APPROFONDIRE + [genova](#), [gabbiani](#), [animali](#), [aereo](#), [aeroporto](#)



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IL GIORNALE DI DOMANI
TI ARRIVA LA SERA PRIMA.

DISPONIBILE DALLA MEZZANOTTE
PROVALO 1 MESE GRATIS

ALTRI ARTICOLI

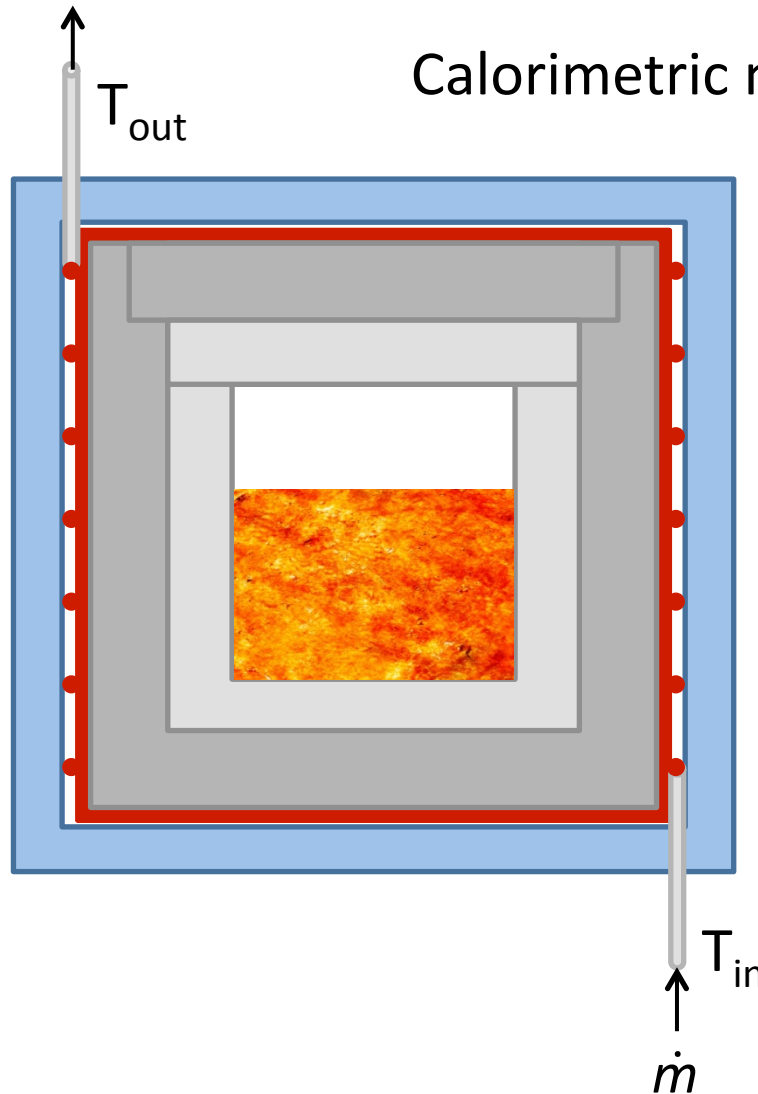
Paris, February 6 2013

Source initial power $P_0=2500$ W

$$P = P_0 e^{-t/\tau} \quad \tau=40 \text{ d}$$

Estimated power after 2 month: $P \approx 500$ W

Calorimetric measurements



W, Ni, Fe Alloy

steel

thermal insulation

copper

$$\dot{Q}_{source} = \dot{m}C(T_{in} - T_{ext}) + \dot{Q}_{cond}$$

$$\dot{Q}_{source} = 2KW; \quad \dot{m}_{water} = 0.01Kg/s \rightarrow \Delta T = 48^{\circ}C$$

Calorimetric measurements $\frac{\Delta\dot{Q}}{\dot{Q}} \leq 0.01$

Steady state equation: $\dot{Q}_{source} = \dot{m}C (T_{in} - T_{ext}) + \dot{Q}_{cond}$

Temperature T:

Calibrated Platinum Thermometers allow measurements with with accuracy $<10^{-3}$
(the fluxes in different pipes must be accurately mixed.)

Specific heat C:

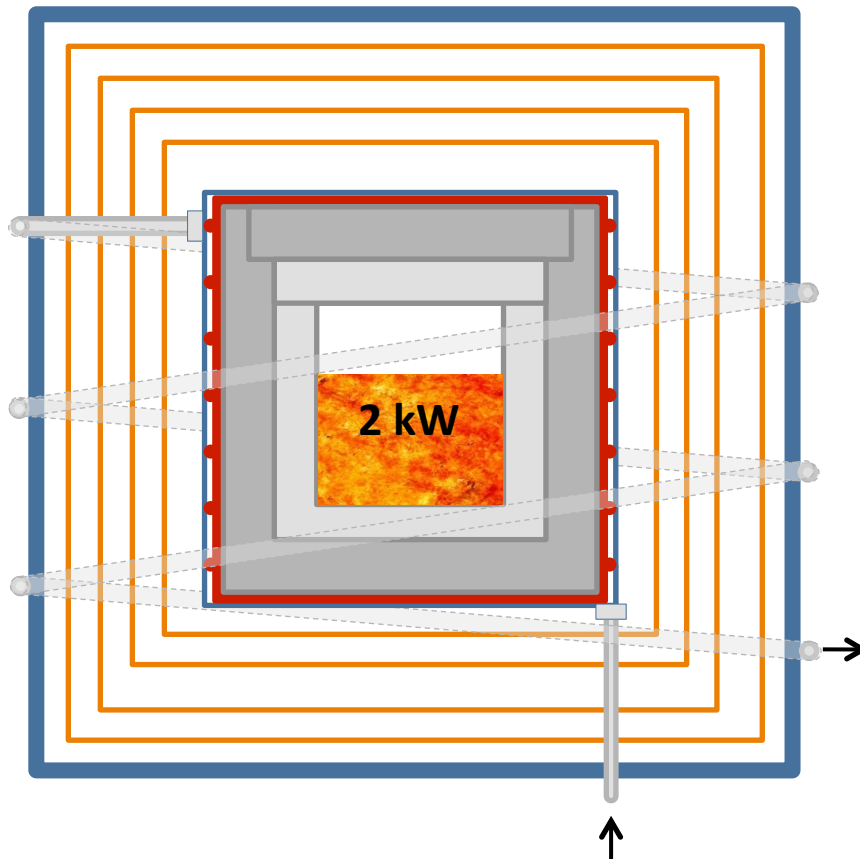
Using pure water, C is known with accuracy $<10^{-4}$

Mass flow \dot{m} :

Coriolis flowmeters allow measurements with accuracy $\pm 10^{-3}$

Heat losses \dot{Q}_{cond} due to conduction, radiation and convection must be minimized

Vacuum insulation + thermal shields



With 5 copper shields $\dot{Q}_{\text{rad}} \approx 2 \text{ W}$

To further reduce \dot{Q}_{rad} , the outgoing water flux is used to warm up the vacuum chamber.

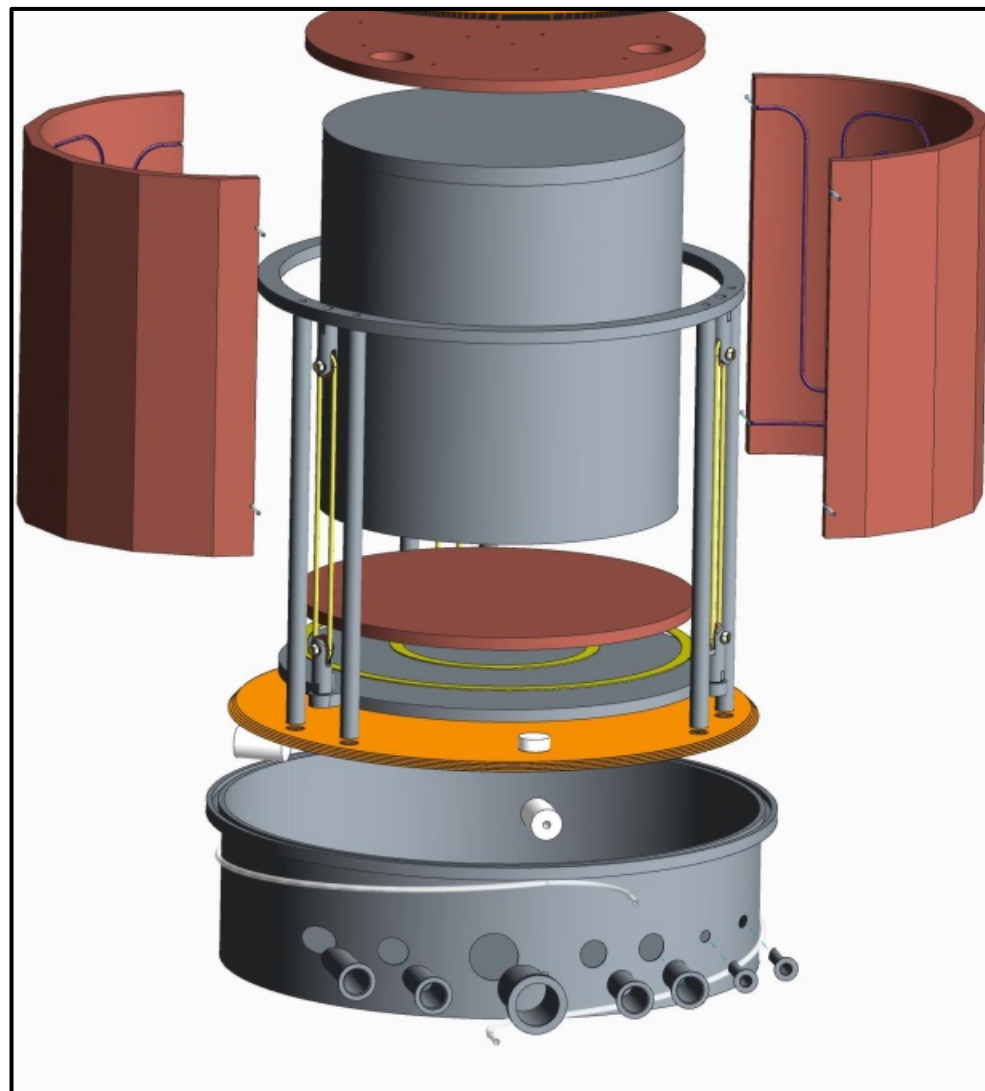
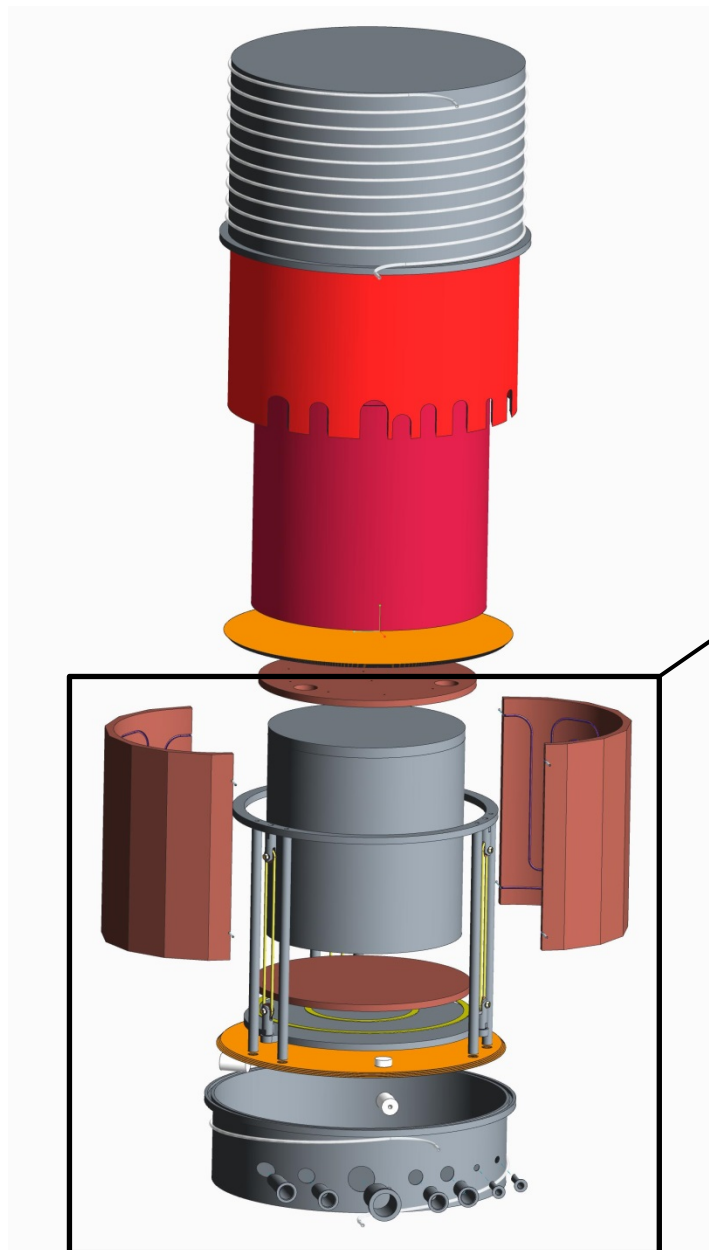
$\dot{Q}_{\text{supports}} \leq 0.1 \text{ W}$ (suspended using kevlar ropes)

$\dot{Q}_{\text{gas}} \approx 0.7 \text{ W}$ @ 10^{-4} mbar

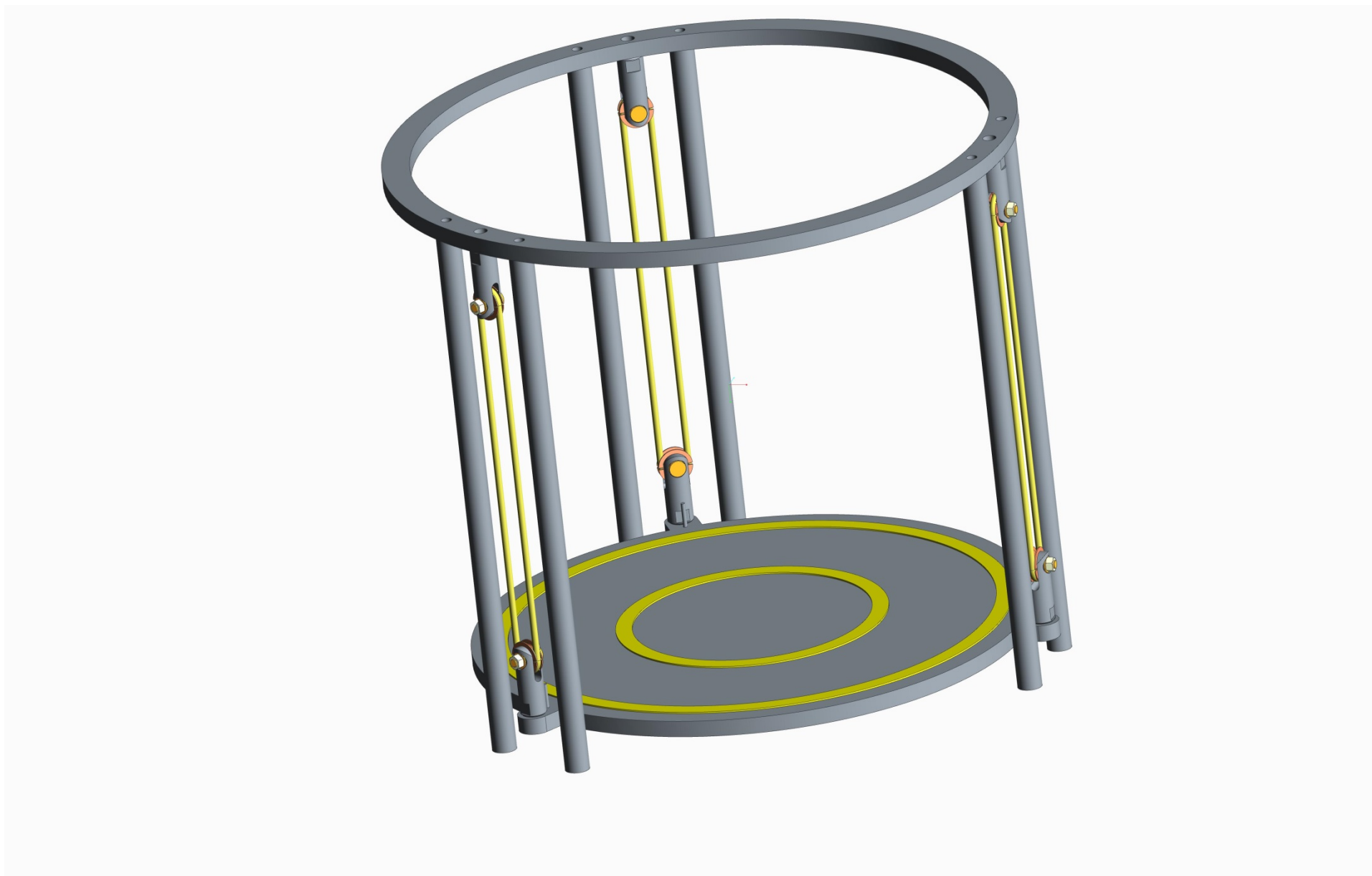
$\dot{Q}_{\text{water flow friction}} \approx 0.3 \text{ W}$ ($\dot{m} = 0.04 \frac{\text{kg}}{\text{s}}$)

Other conduction heat losses (wires, pipes, spacers): $\dot{Q}_c \approx 0.2 - 0.3 \text{ W}$

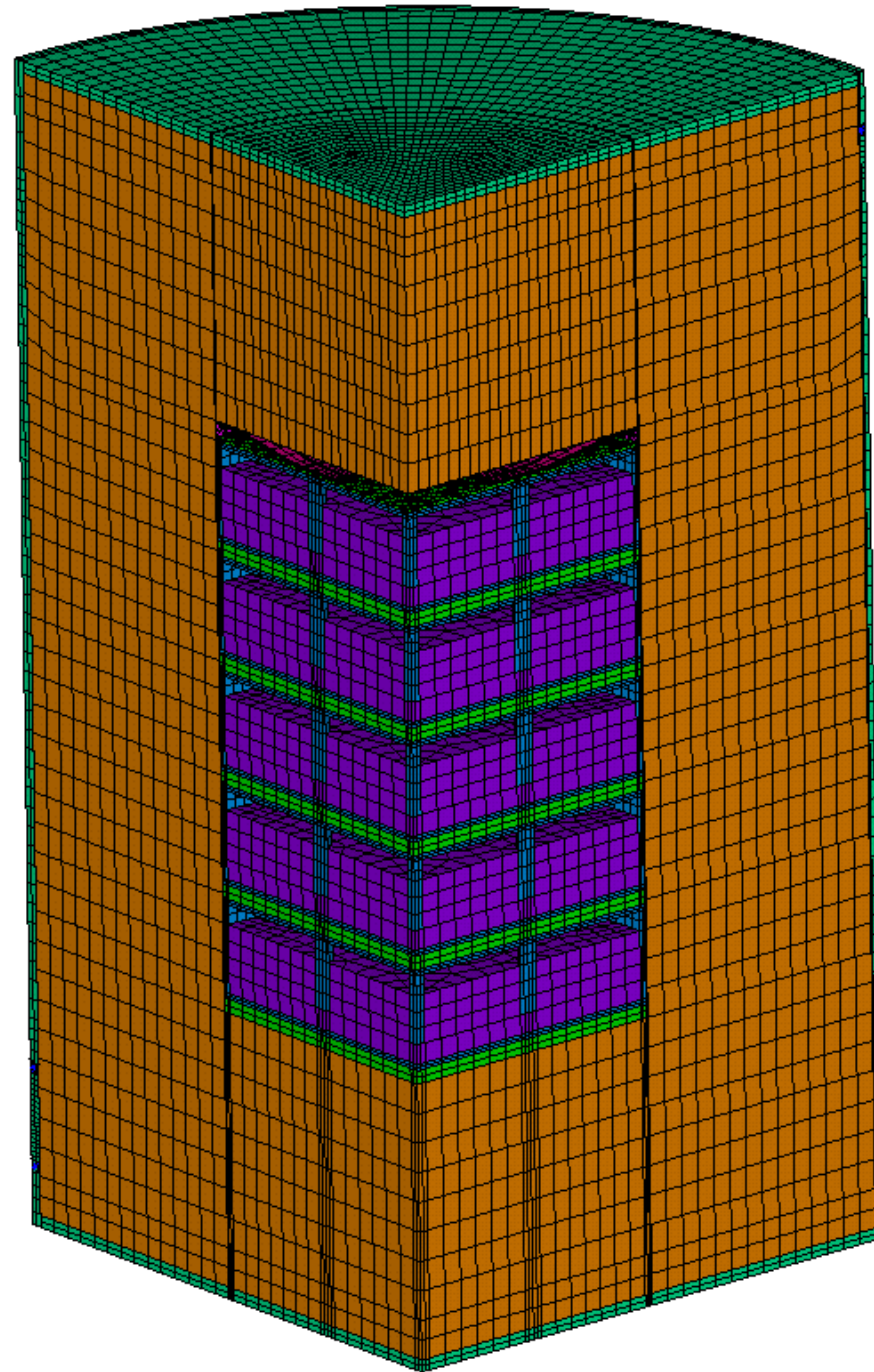
⇒ Estimated $\dot{Q}_{\text{cond}} < 4 \text{ W}$



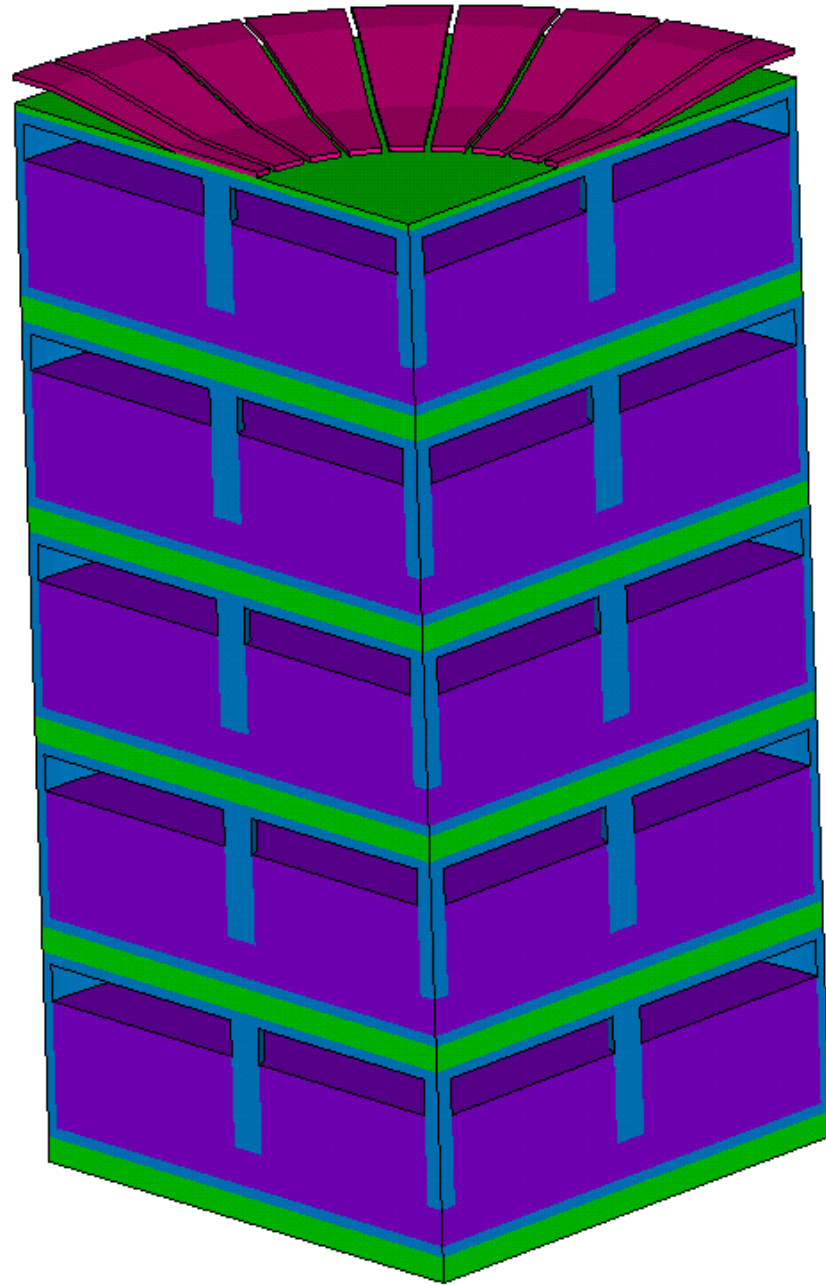
Mechanical Design
Roberto Cereseto and Fabio Bragazzi
(Servizio Progettazione Meccanica, INFN – Genova)



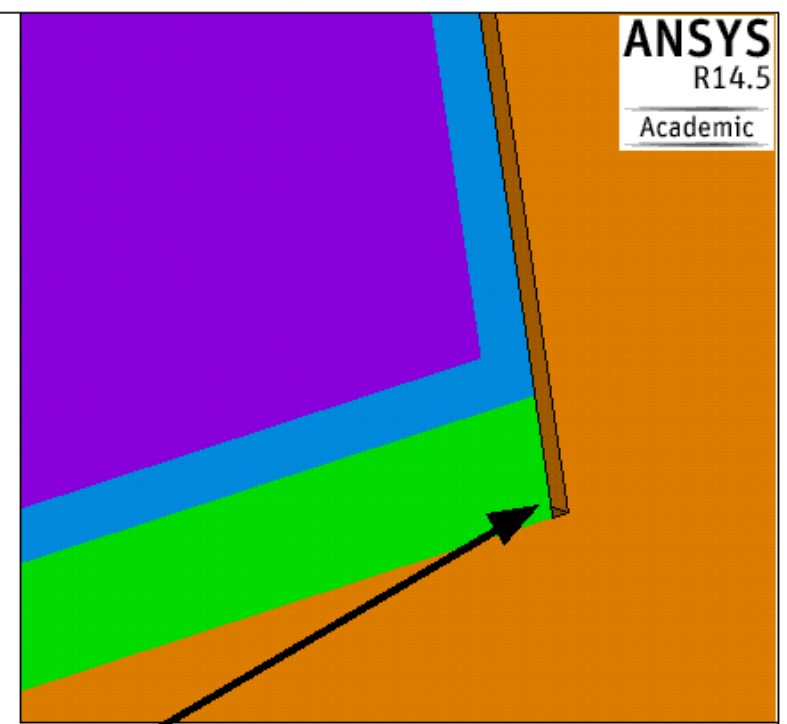
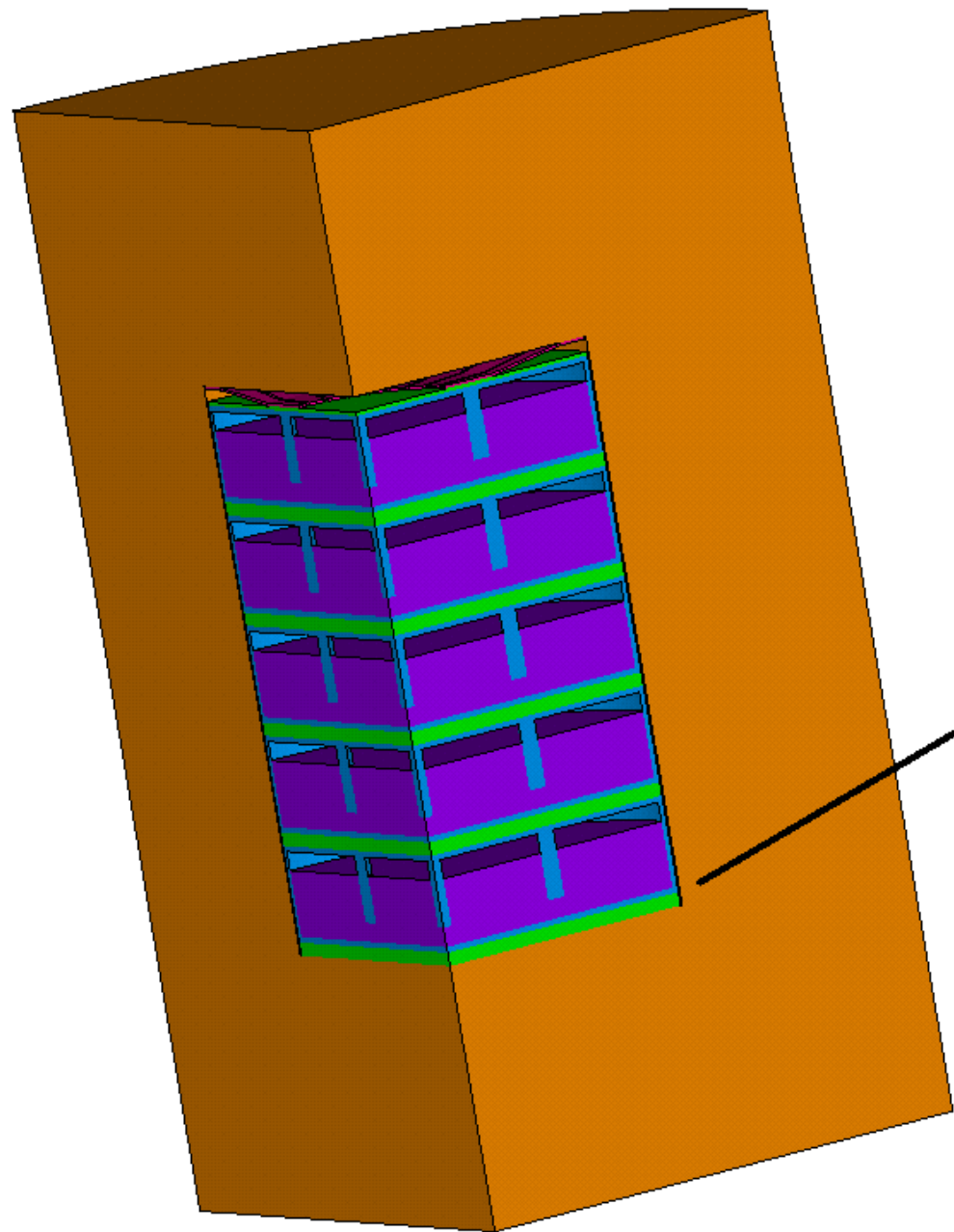
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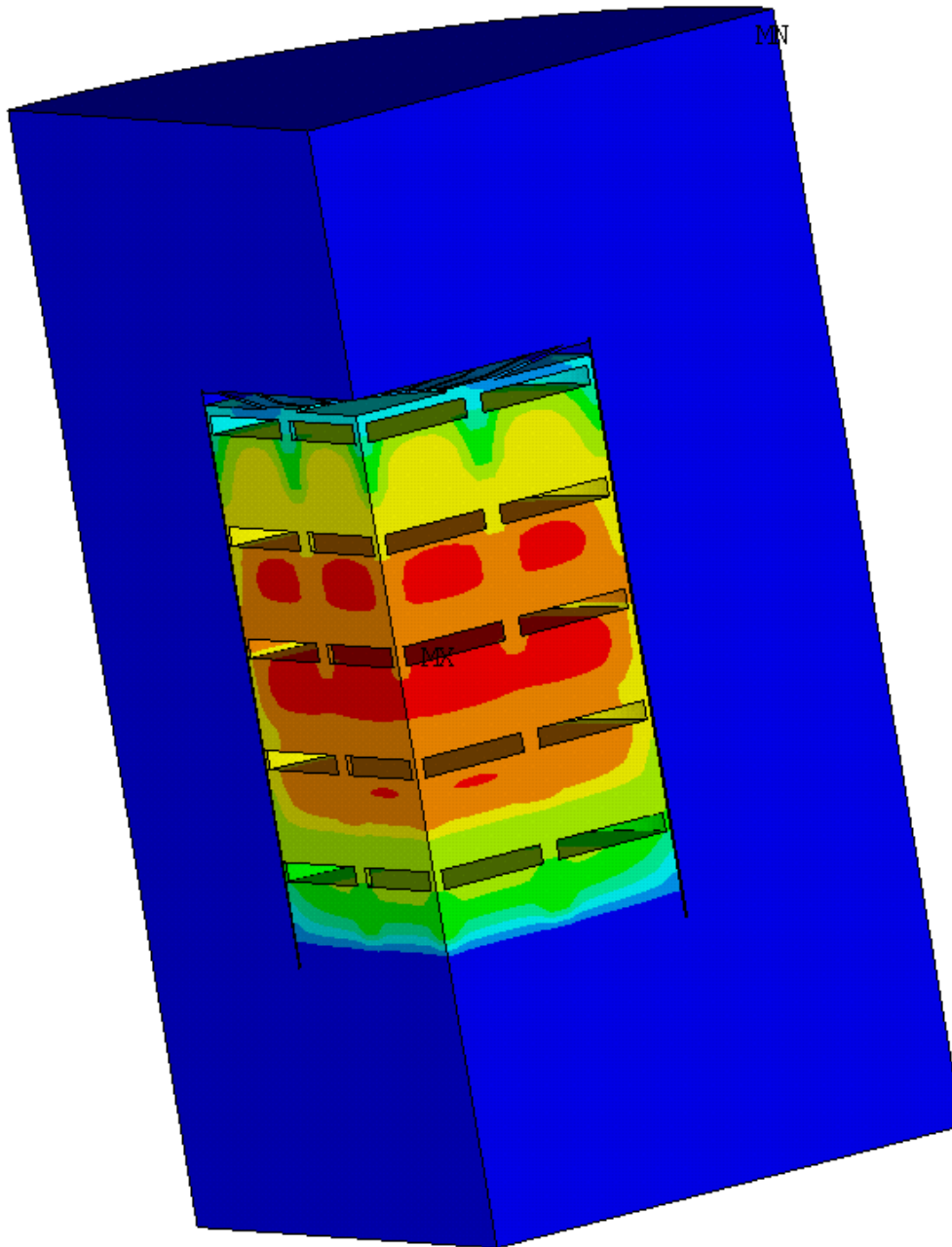
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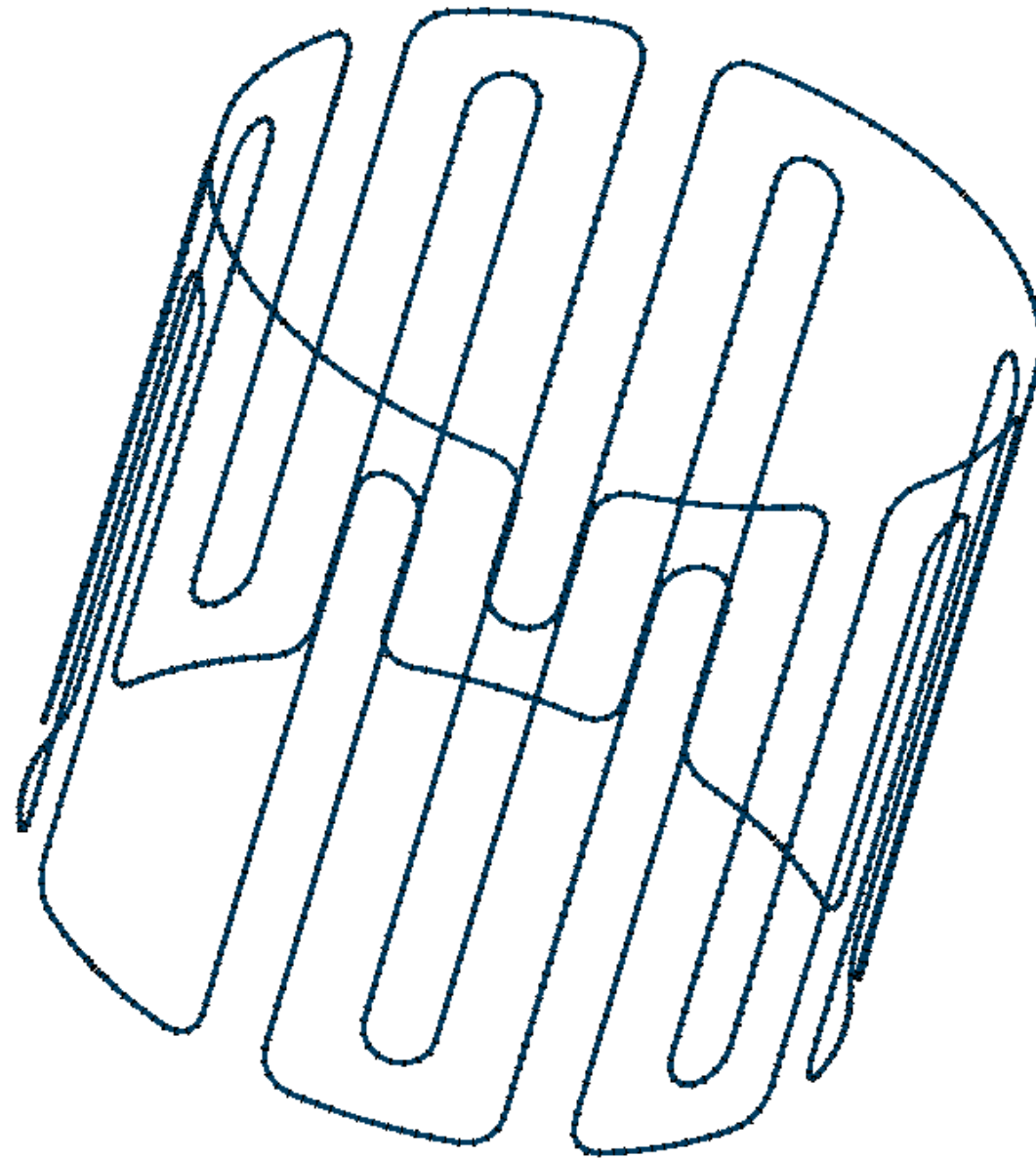


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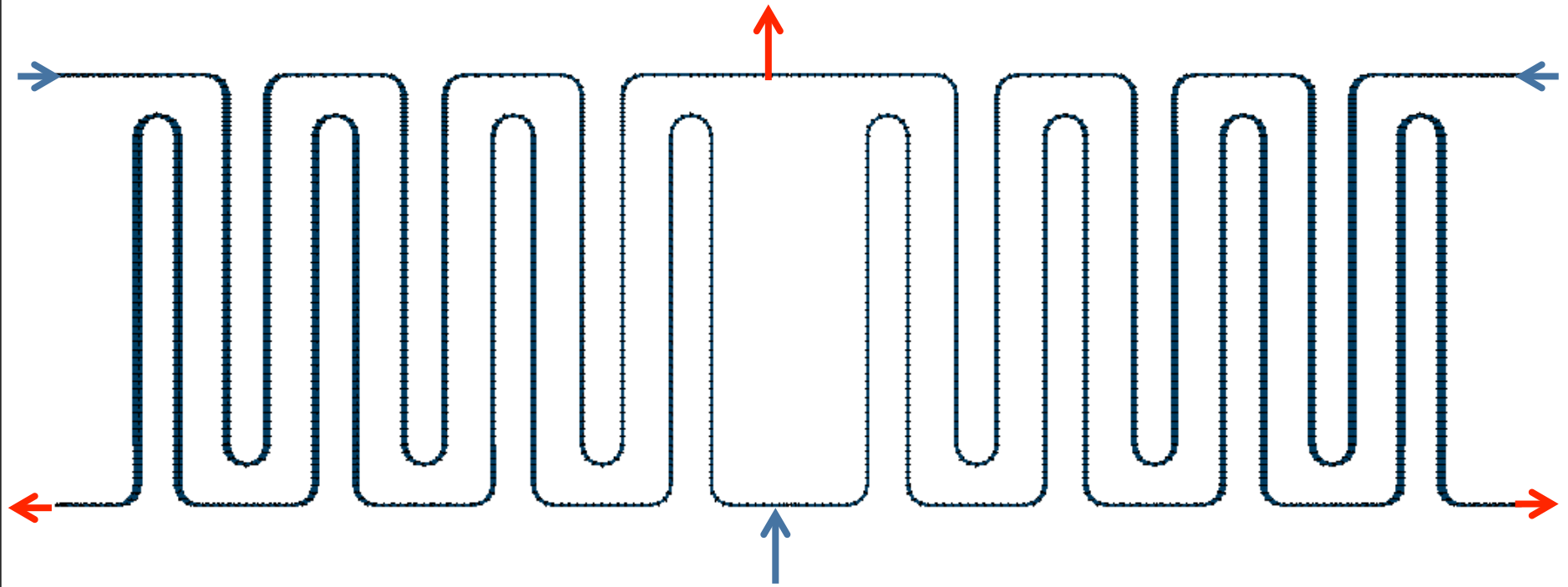
ANSYS 14.5
DEC 13 2013
11:55:26
PLOT NO. 1
NODAL SOLUTION
STEP=1
SUB =1
TIME=1
TEMP
SMN =20.5528
SMX =416.64
20.5528
64.5626
108.572
152.582
196.592
240.602
284.611
328.621
372.631
416.64

dissipated power = 2500 W



1
1

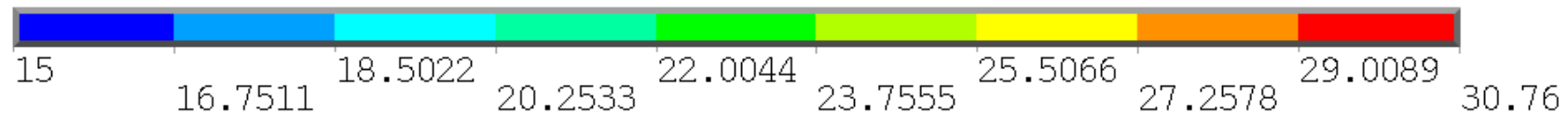
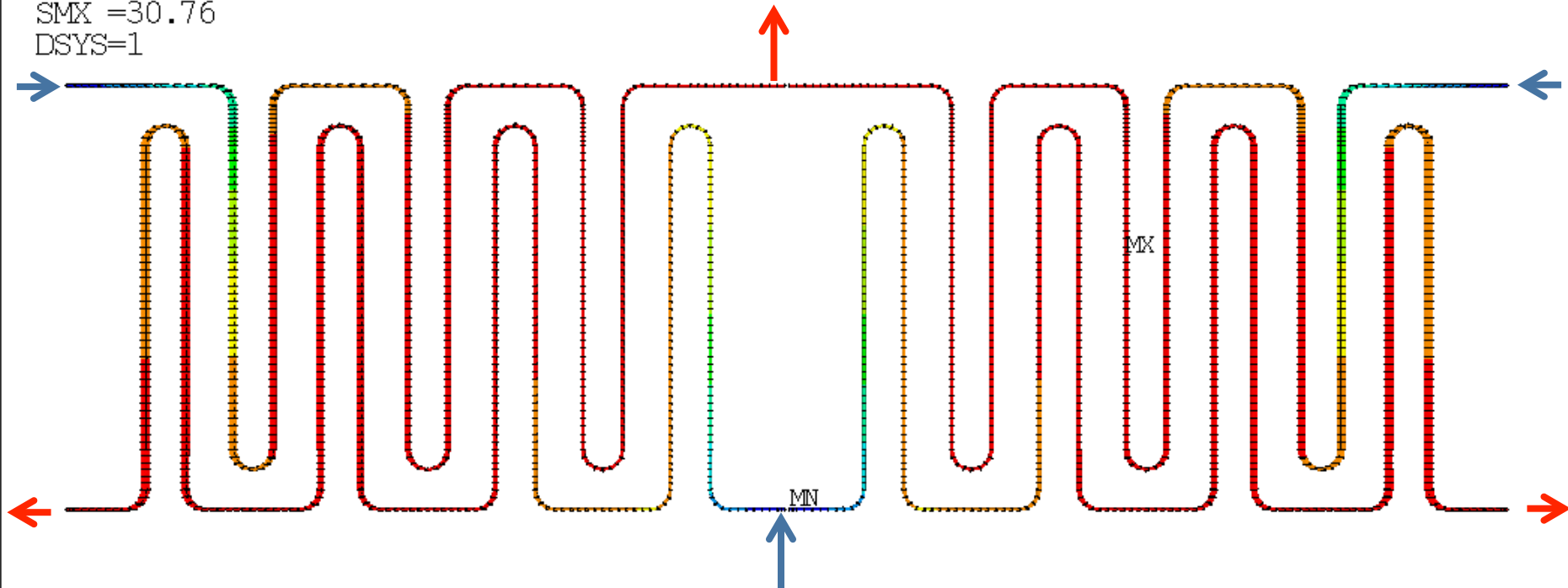
1



DEC 13 2013
13:26:57
PLOT NO. 1

1
NODAL SOLUTION
STEP=1
SUB =1
TIME=1
TEMP (AVG)
RSYS=0
SMN =15
SMX =30.76
DSYS=1

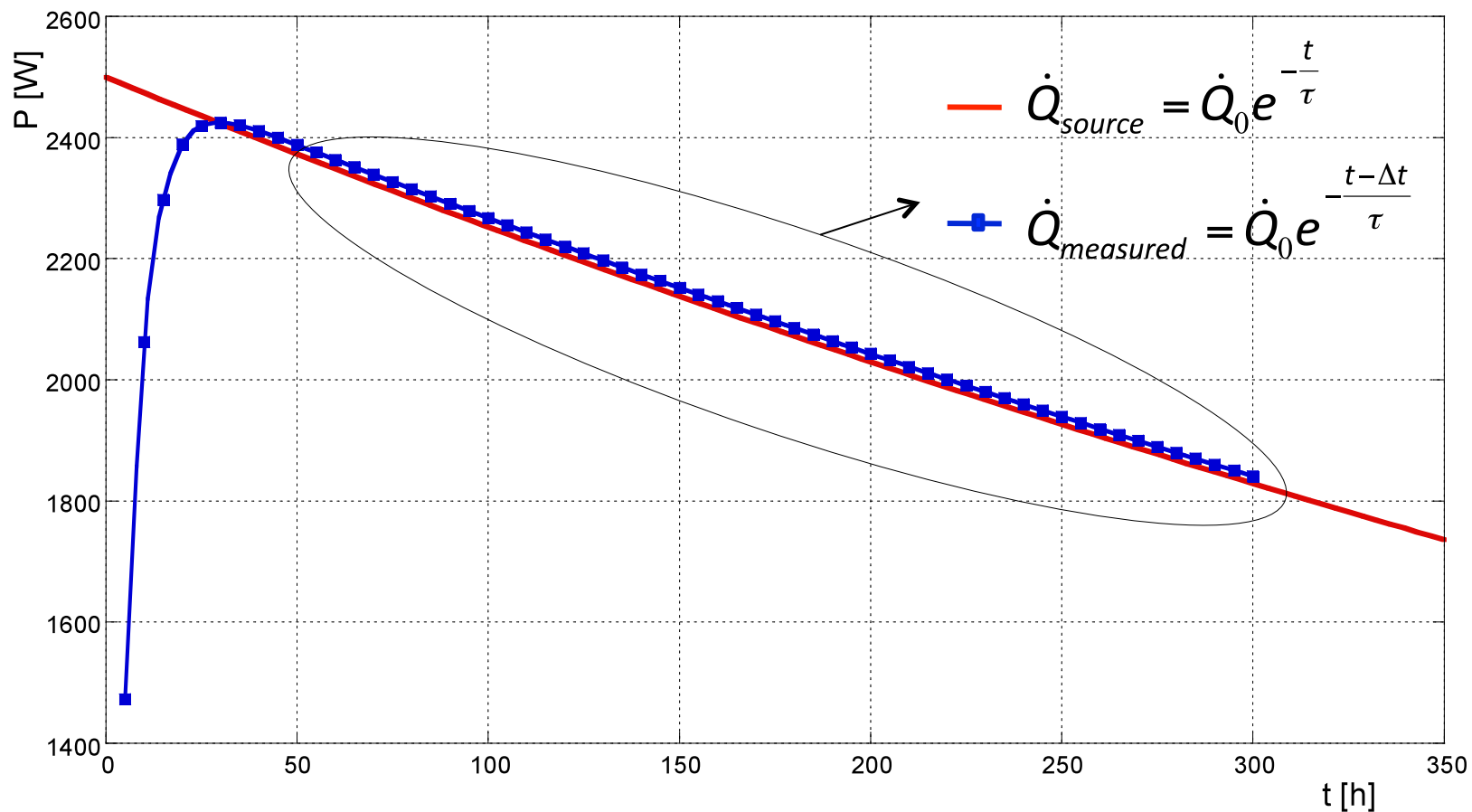
Tmin = 15 C
Tmax1 = 30.312 C Tmax = 29.917 C
Tmax2 = 29.522 C
 $P = 4 \times 0.01 \times 4190 \times (29.917 - 15) = 2500.09 \text{ W}$

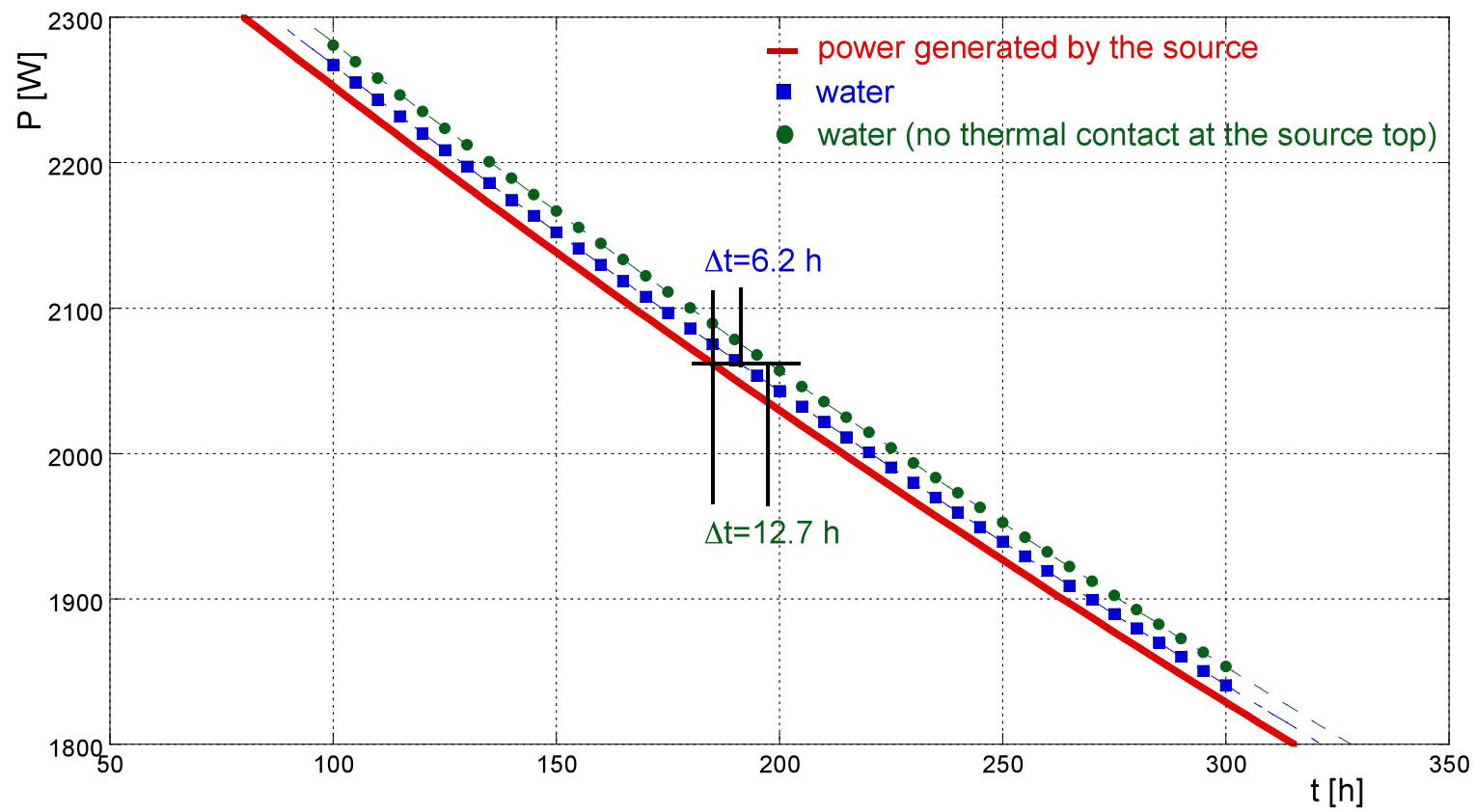
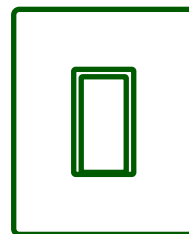
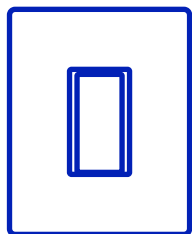


TRANSIENT ANALYSIS

$$\nabla \left(\frac{k}{\rho C} \nabla T \right) = \frac{\partial T}{\partial t}$$

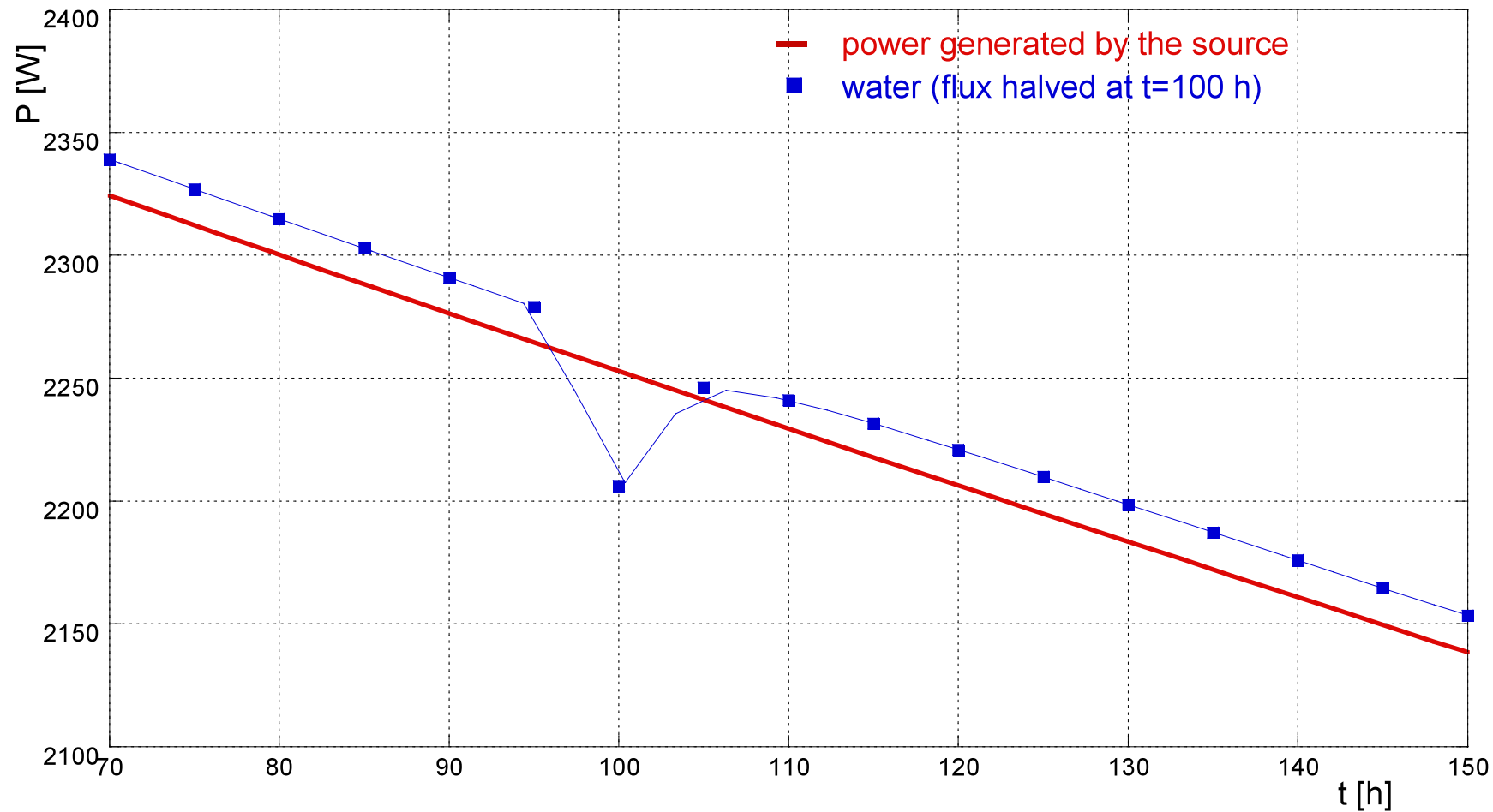
At $t=0$, the source is inserted in the tungsten cylinder (taken at uniform temperature $T=15^\circ\text{C}$).
 The external wall is cooled by a 0.04 Kg/s water flow.





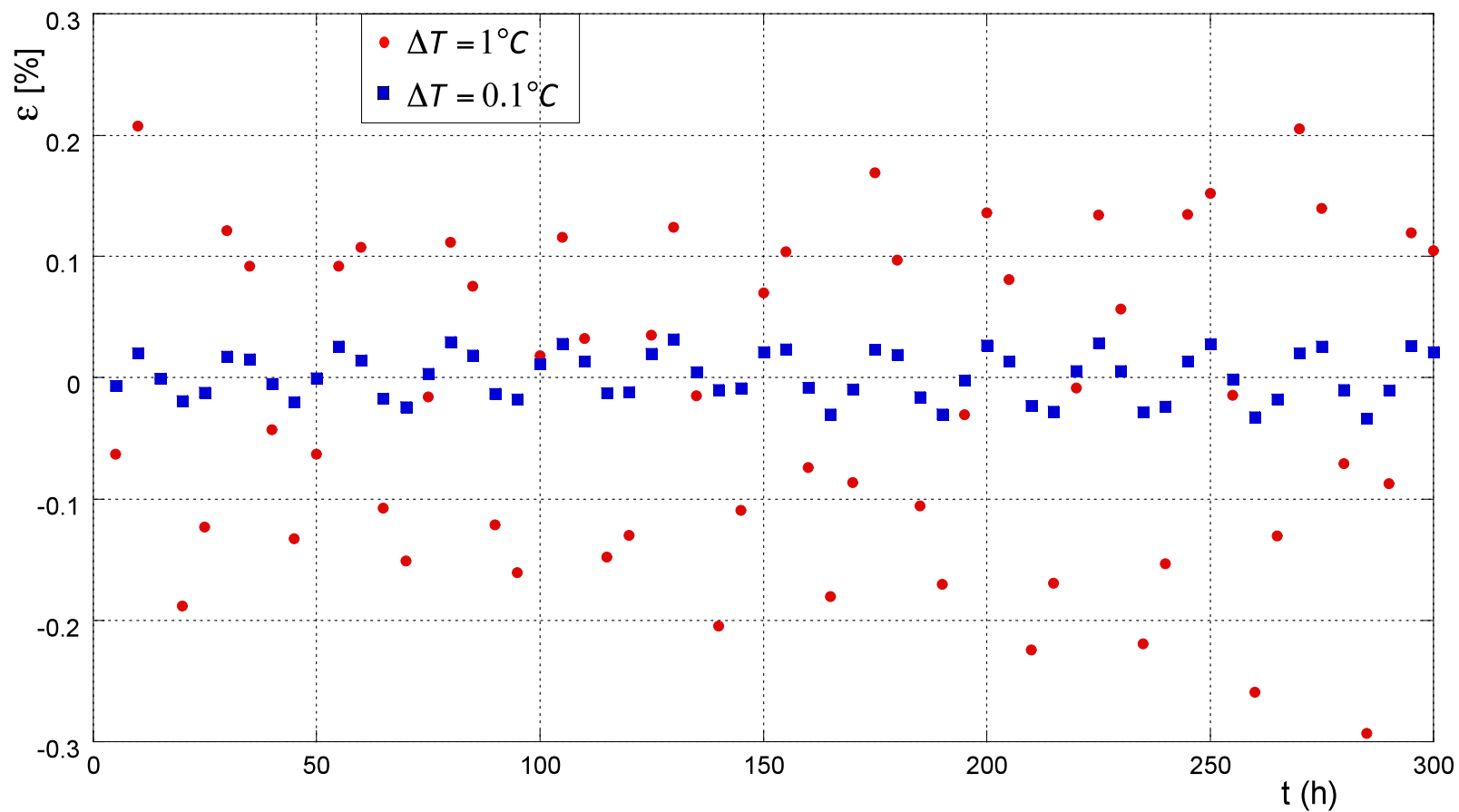
What happens if we change the cooling conditions during the measurement?

Case 1: reduction of the water flux



Case 2: daily oscillation of the water temperature

$$T = T_0 + \Delta T \cos \frac{2\pi}{24} t \quad ; \quad \varepsilon = \frac{P(T) - P(T_0)}{P(T_0)} \cdot 100$$



The cooling conditions are controlled but variations
can be done by purpose.
Small unwanted variations are possible but are monitored.

Are uncontrolled variations possible?

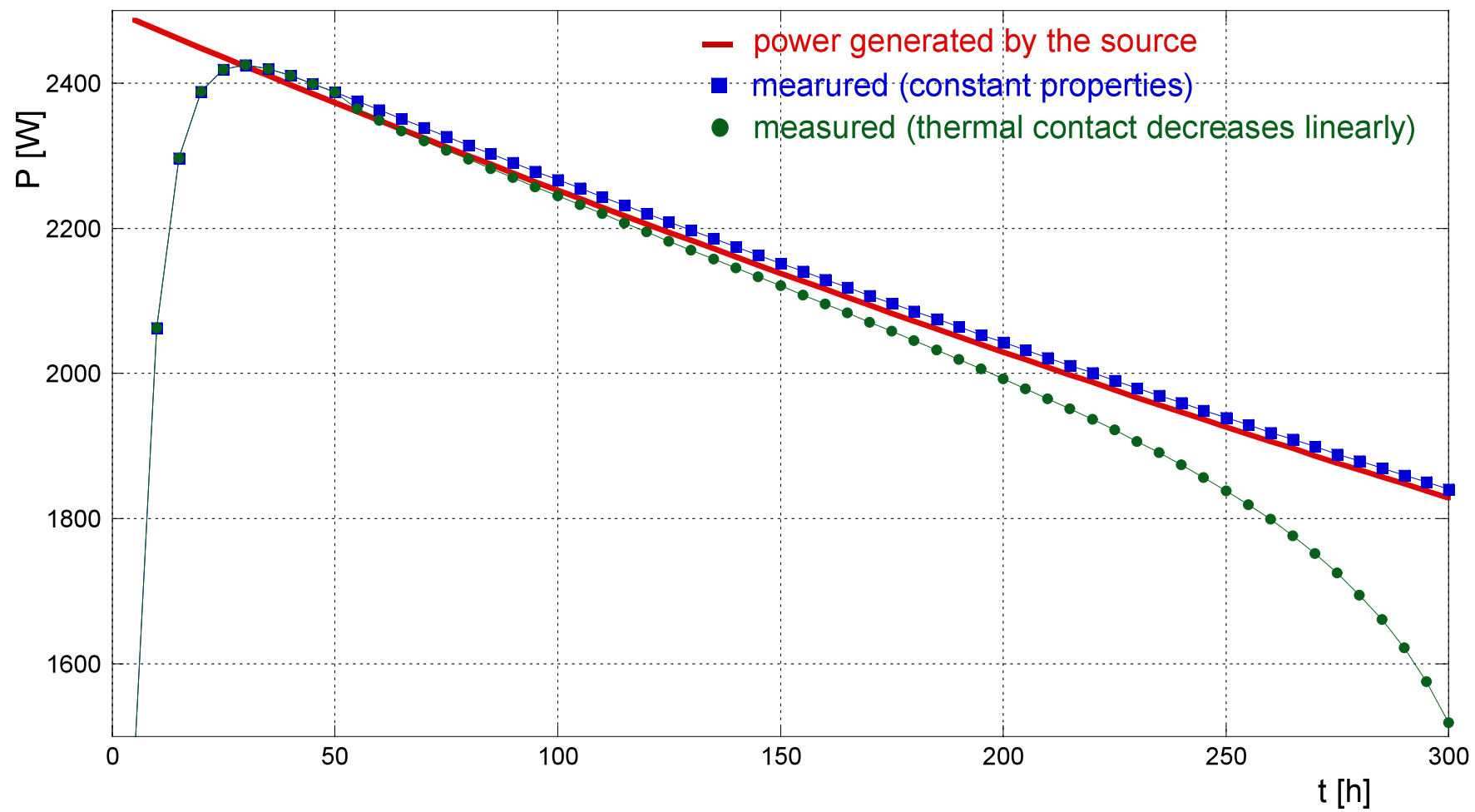
Temperature dependent material properties

In principle are known but we have to know $T(x,y,z)$ inside the source.

FEA will help us

Changing of thermal contacts

Unpredictable: FEA cannot help



**An effort must be done to make the thermal contacts
constant and reproducible.**

Not necessarily good but constant and reproducible.

Next step:

Finite Element Analysis of the new source
(with temperature dependent properties)

THE END