# Two phase CO2 cooling for Micromegas Modules

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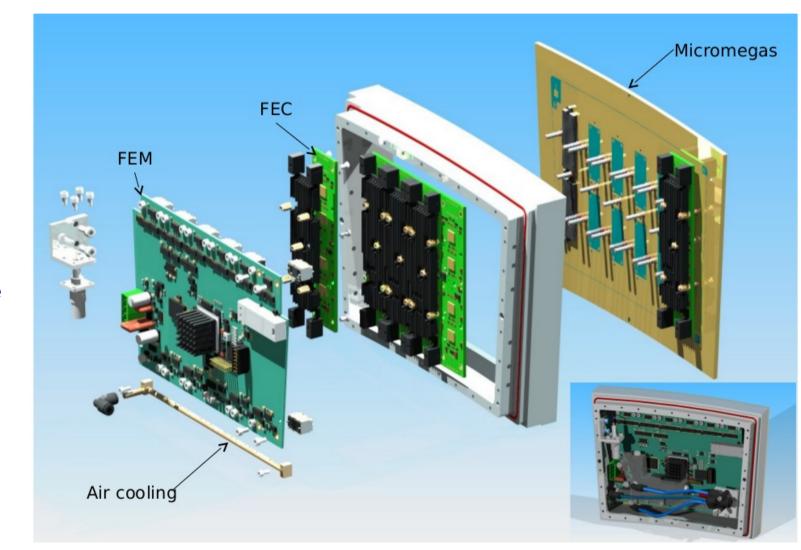
RD 51, 6 Feb 2014, CERN, Geneva

## Overview :

- The electronics
- Large Prototype TPC for ILC
- Requirement for cooling and why CO2 cooling
- How cooling is done
- Results of cooling
- Future plans

#### The AFTER electronics is applied for Micromegas readout

#### 6 FECs, 1 FEM, 24 Chips, 1728 ch



24 rows with 72 pads

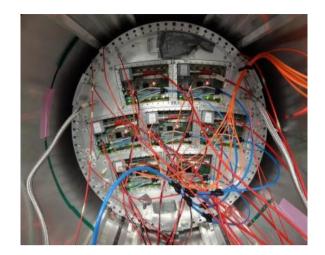
3/7 mm2 pad

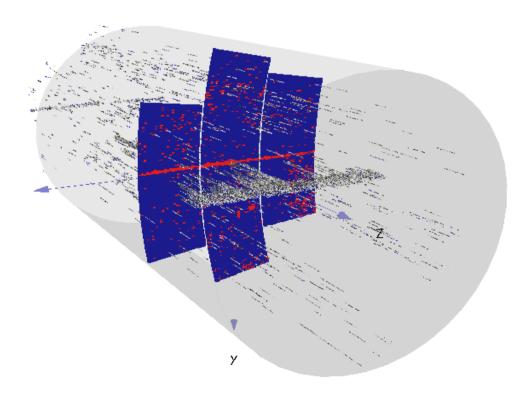
1728 pads per module

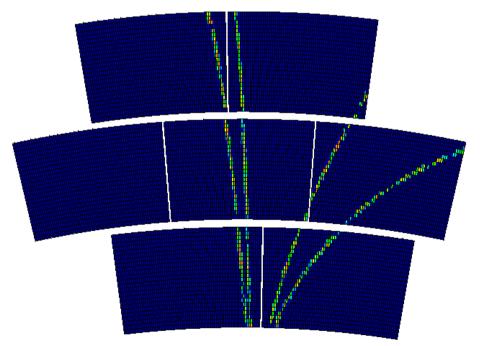
resistive foil to spread charge











## **Requirement For Cooling**

### The electronics runs at 5 Volt and consumes power nearly 26 Watts

6 FECs	ASICs = 12 Watts	19 Watts
	Power Regulators = 7 Watts	
FEM		3.5 Watts
FPGA		3.5 Watts
Total		26 Watts

## **Requirement For Cooling**

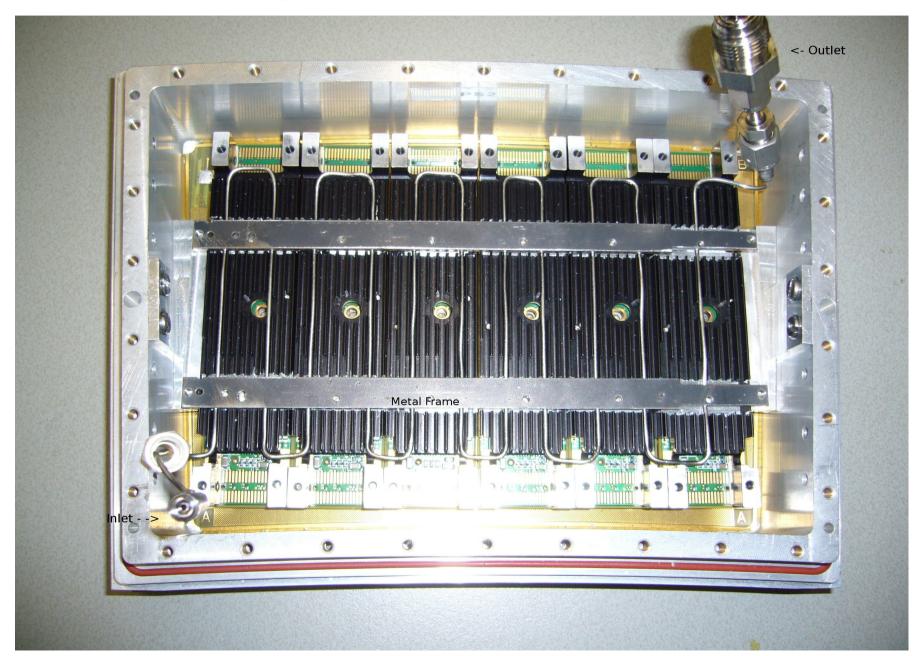
This power consumption rises up temperature of the Module up to 60 degrees

Growth of temperature results in:

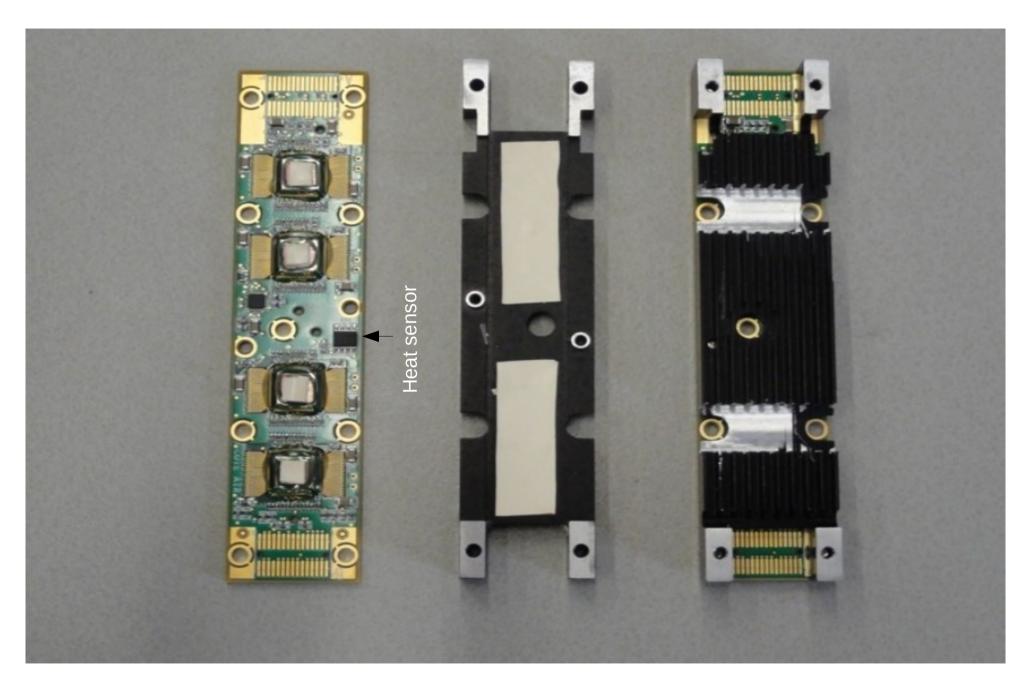
- possibility of damage in electronics, if left running for hours without cooling
- heating up of pad plan and hence convection current in TPC gas

Conclusion: 'proper cooling is necessary'

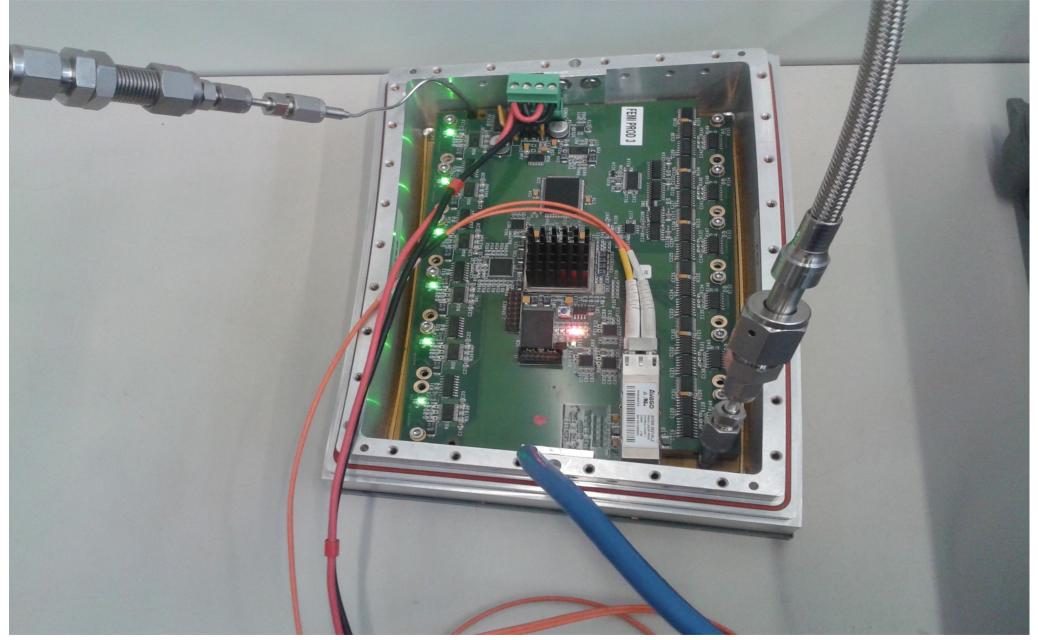
### This Cooling circuit inserted inside the Module



FEC and the Radiator. Each FEC contains one heat Sensor.



# A test of the cooling circuit has been carried out at NIKHEF during 02/Dec/2013 to 04/Dec/13



### The CO2 cooling system used at NIKHEF



#### **At NIKHEF**



### **Transportable Refrigeration Apparatus for CO2 Investigation**

or 'TRACI'

(Purchased by KEK)

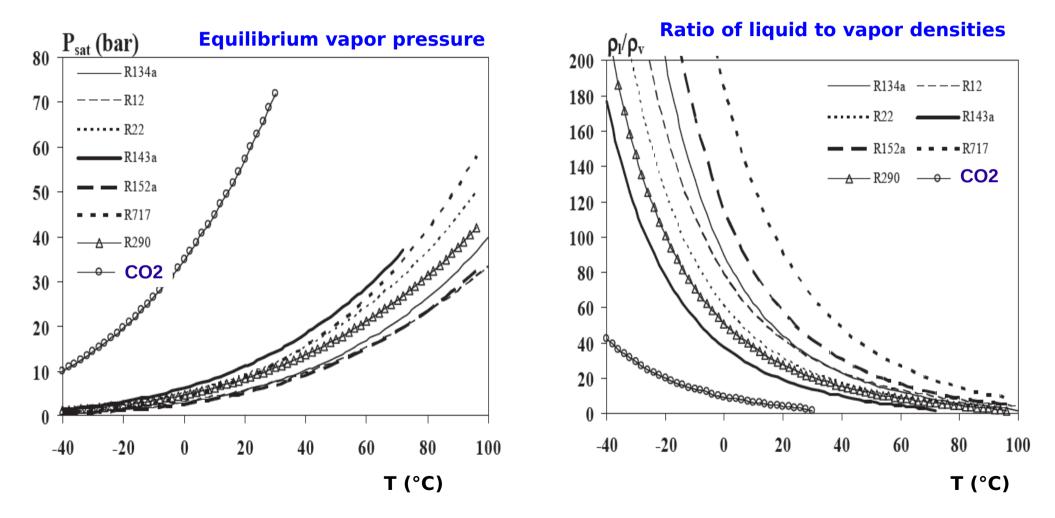
## **Can Circulate Liquid CO2**

- at different pressures, hence, at different vapor temperatures.
- and at different flow rates.
- cooling power: '100 watts 250 watts'.
- cooling ranges form " -40 °C to +20 °C.

#### Why CO2 as a refrigerant?

Higher slope of the vapor pressure curve means that  $\Delta P$  in the system is smaller.

Low ratio of liquid to gas densities results in a more homogenous two-phase flow (Pettersen 2002).

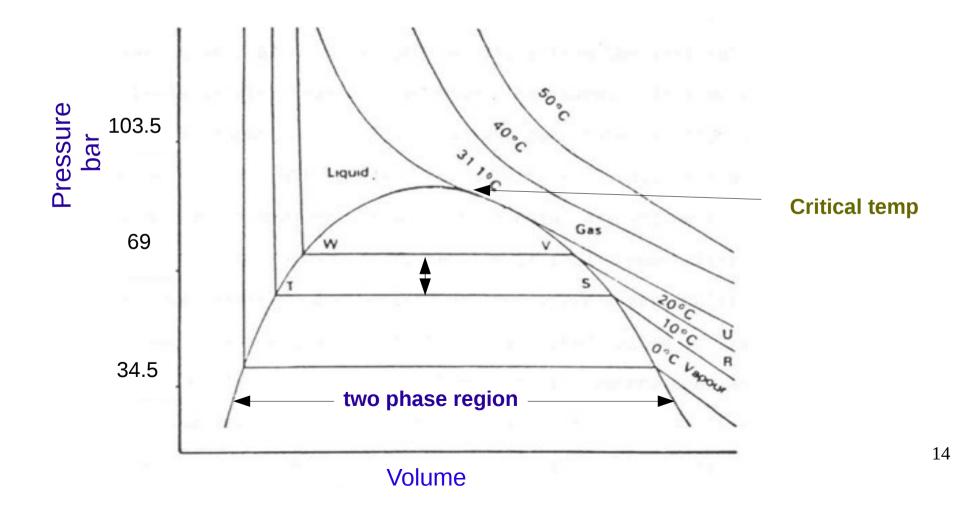


#### Why two phase cooling ?

Liquid CO2 has large latent heat → low flow rate

Low viscosity  $\rightarrow$  uniform flow

Evaporative CO2 cooling  $\rightarrow$  allows small tubing

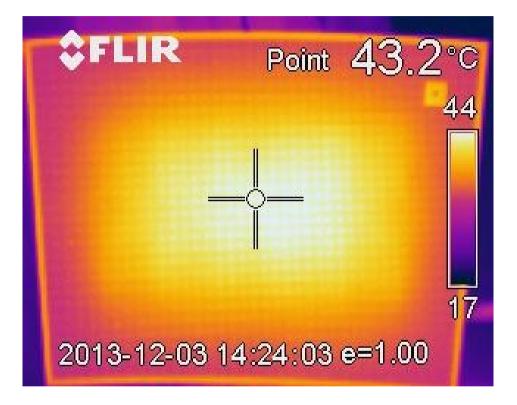


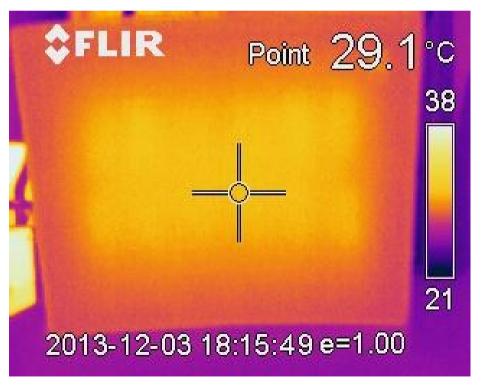
## Temperature difference with and without cooling\*

FECs and FEMI	Temp ( °C) Without cooling	Temp (°C) With cooling
FEC 0	55	35
FEC 1	58	34
FEC 2	<b>61</b>	33.5
FEC 3	62	34.5
FEC 4	60	34
FEC 5	55.5	33.5
FEMI	54.5	37

\*At Out-Pressure=45 bar, In-Pressure=55.2 bar, Temp 10.0 °C, Flow rate=2.0 gm/sec

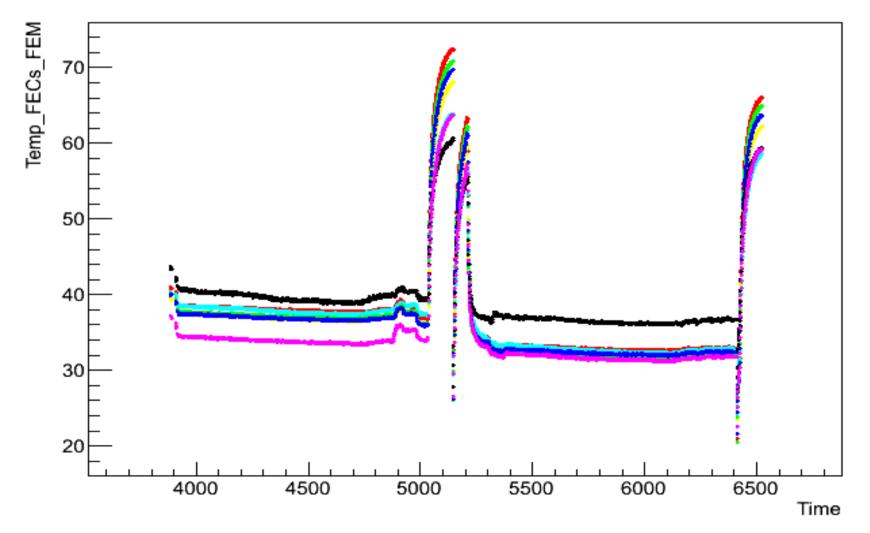
### Infrared images of the module without and without cooling





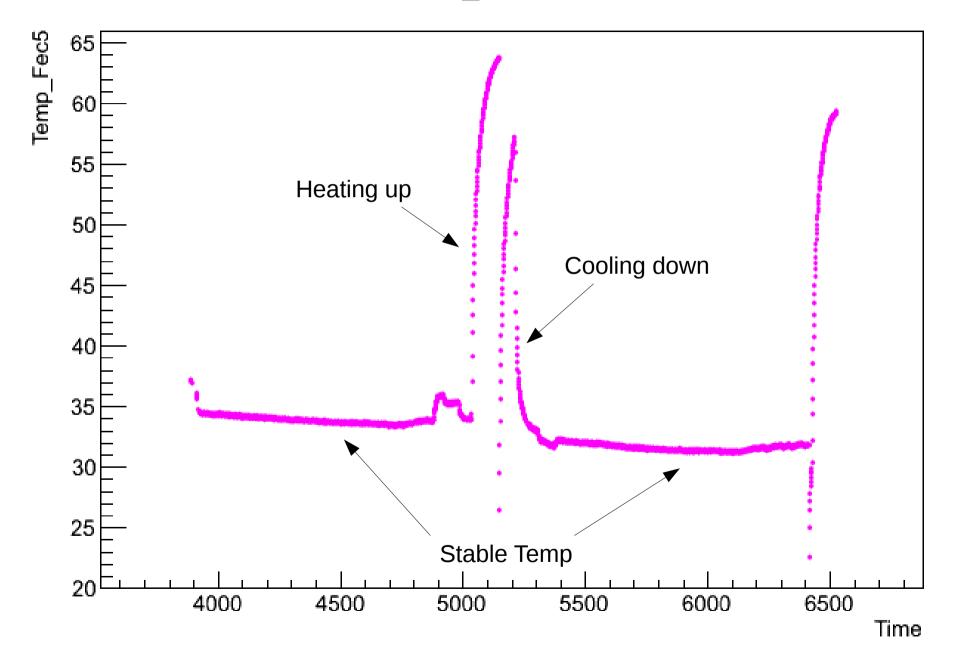
### Temperature profile for all the FECs and the FEMI

Temp\_FECs\_FEM:Time



7

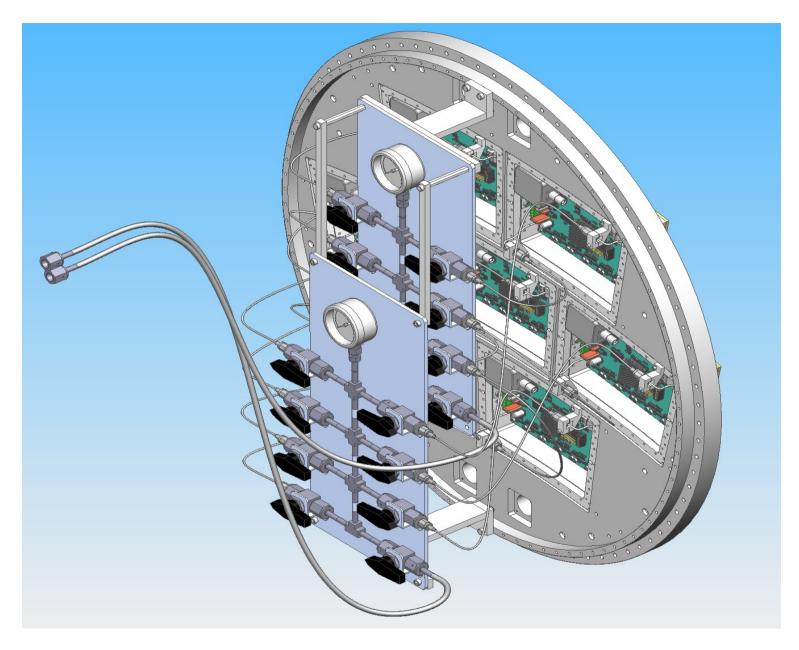
### Temp\_Fec5:Time



## **Conclusions and future plan**

- the temperature profiles show that cooling can stabilize temperature at 31 to 33 degree.
- better thermal contact with the cooling tubes and heat radiators will increase the performance.
- we are planing to apply this cooling model to all 7 modules in parallel for our next data taking at DESY starting from 17 Feb 2014 .

## **Future Plan**



# Thank You

## **Back Up Slides**

### Temperature vs Drift velocity

