



Update on J-PARC Project

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Collaborators

KEK

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YAMAMOTO**



Contents

- Introduction
- System overview and Design of the Cryogenics
- Status on cryostat components
- Recent status on installation work in the tunnel
- Summary



Contents

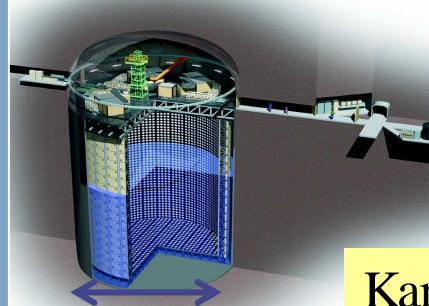
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Introduction

Neutrino physics at J-PARC
Tokai -to-Kamioka (T2K) Long Baseline
 ν Oscillation Experiment

Super-Kamiokande



objective: study the nature of Neutrino in detail

T2K (2009~)

295 km

Kamioka

K2K (1999~2005)
250km

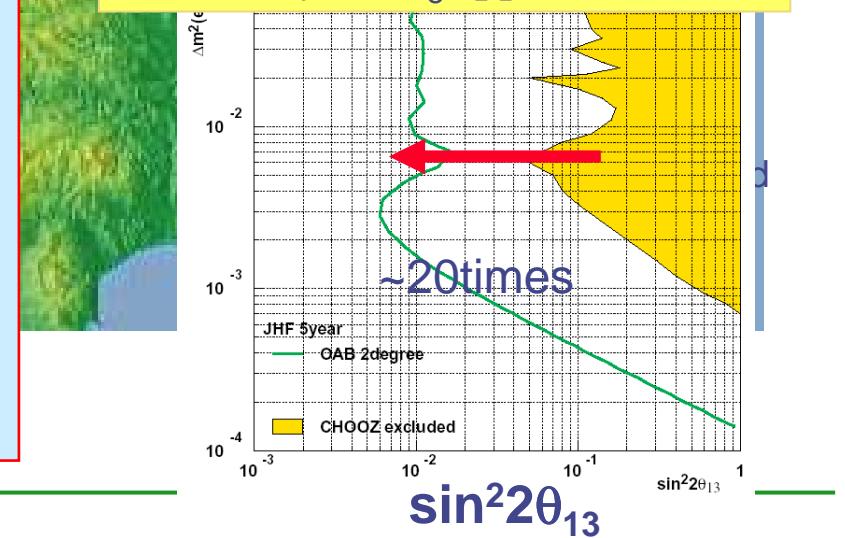
Tokai

KEK

J-PARC
@JAEA

Sensitivity on ν_e appearance

- Off-axis sub-GeV ν_μ beam from J-PARC 50GeV-PS
- ~3000 ν_μ CC int./yr (w/o osc.)
- ν_e appearance discovery
- ν_μ disapp. precise meas.
- 5 year construction
- Start experiment in 2009.





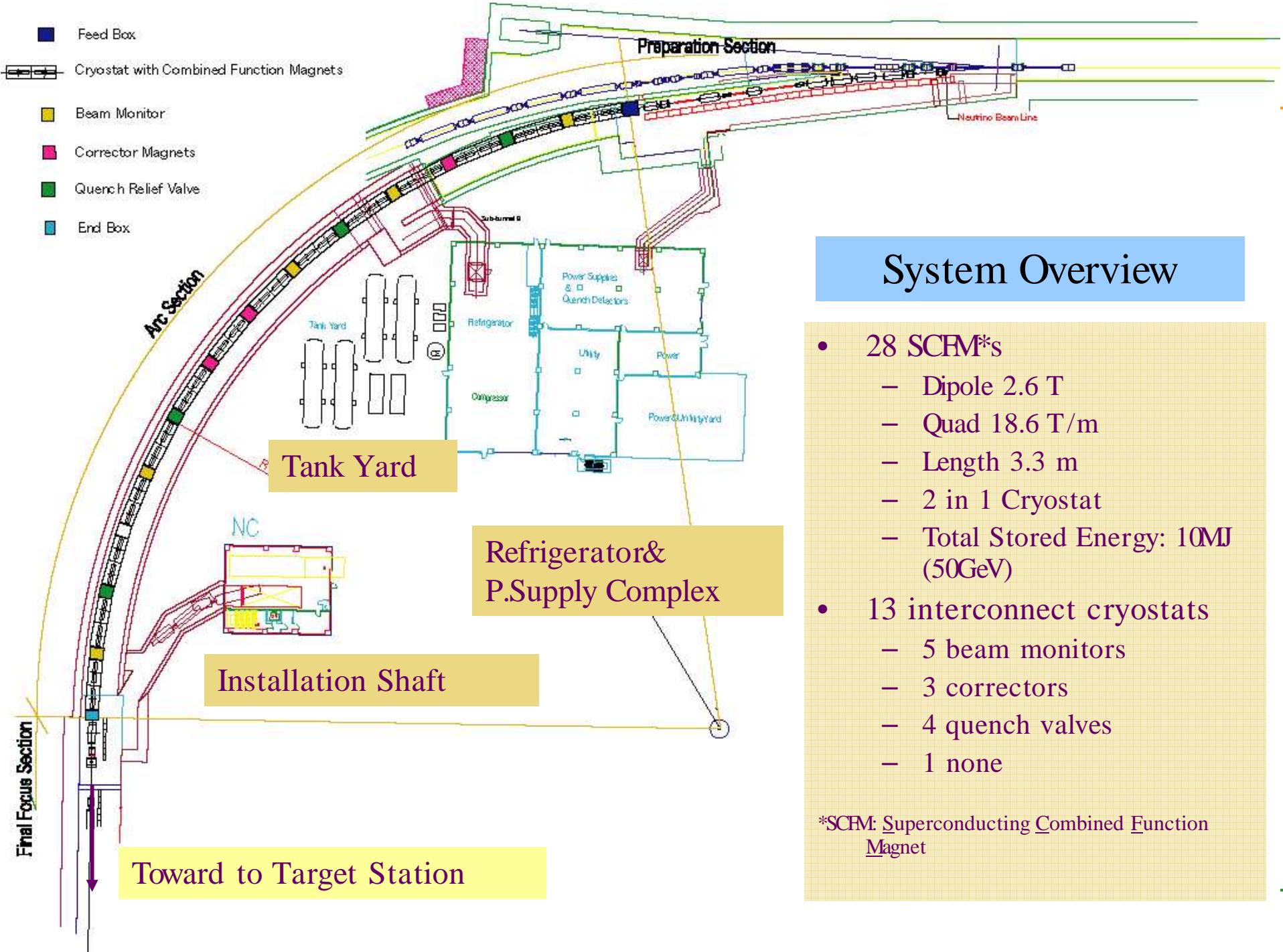
Overview of Project History

- 2001FY
 - Design of Neutrino beam line project (T2K) was started with 10 FODO = 40 Superconducting(SC) Magnets
- 2002FY
 - Superconducting Combined Function Magnets system, which consists of 14 doublet: 28 SCFM, was proposed as bending magnet system.
- 2003FY
 - SCFM was approved by internal review in KEK
 - The first production of prototype magnet was started in-house at the KEK
- 2004FY
 - Civil construction work of Neutrino beam line was started at Tokai
 - A prototype magnet was completed and tested in KEK
 - Design of cryostat and cooling system were started
 - Production of the magnets and cryostats were started at Mitsubishi Electric
- 2006FY
 - Continuing magnet production and cryostats
- 2007FY
 - Continuing magnet production and cryostats and started installation work in the tunnel
 - Production of the Transfer line was started
- 2008FY
 - Installation work should be completed and started commissioning on end of 2008.



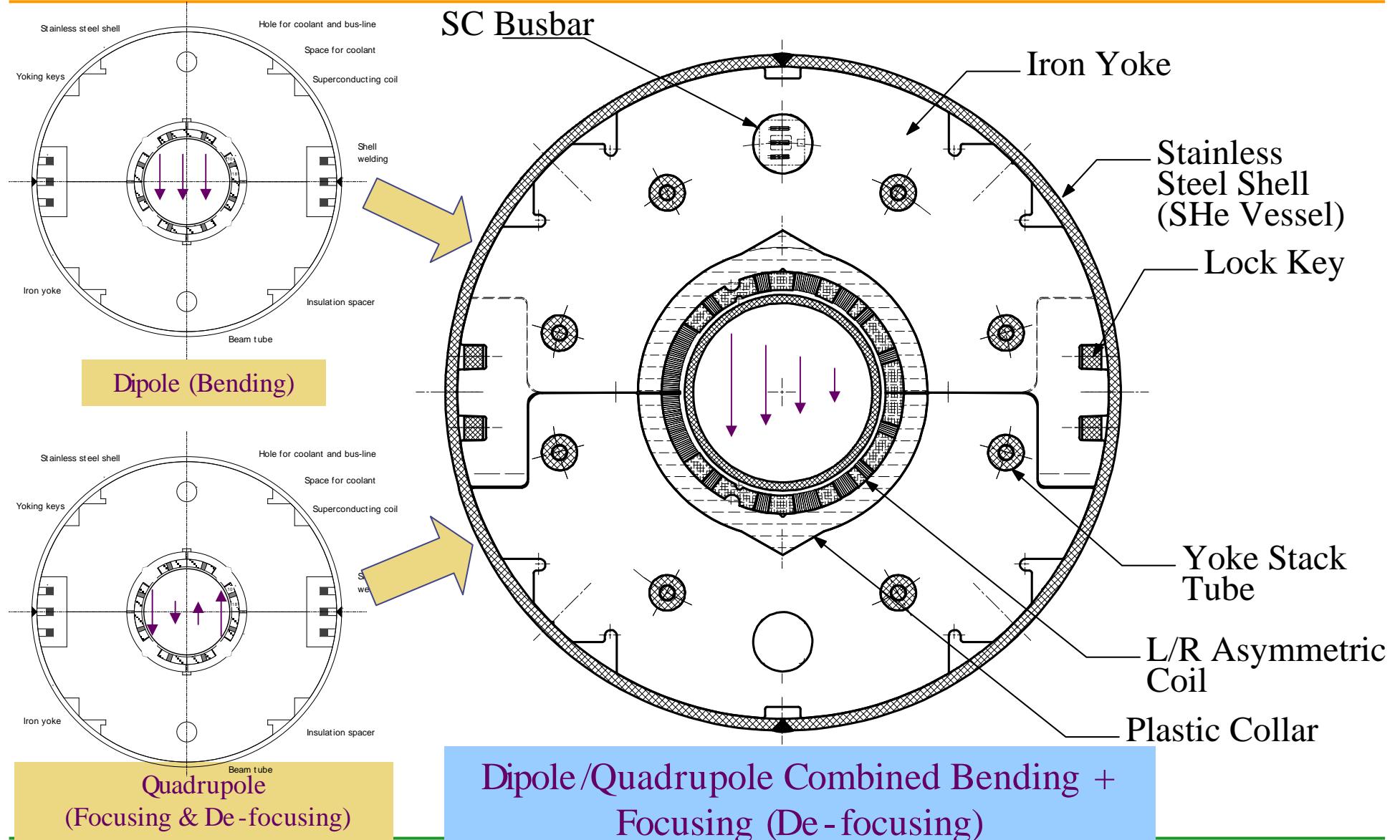
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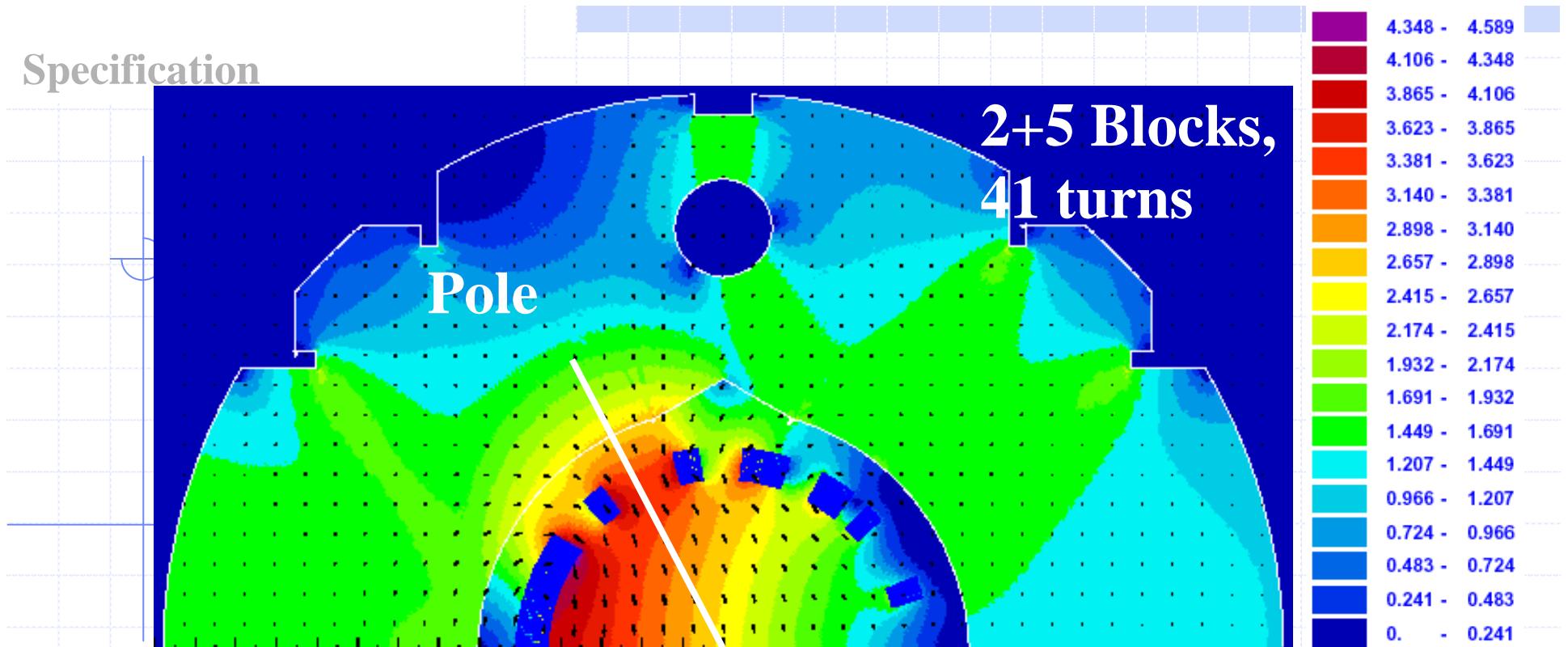




Superconducting Combined Function Magnet



Specification



Coil ID.: 173.4mm

Mag. Length: 3300 mm

Mech. Length: 3630 mm @RT

Tmax: < 5.0K
(Supercritical Helium)

Dipole Field: 2.59 T

Quad. Field: 18.6 T/m

Field Error: $< 10^{-3}$

Op. Current: 7345 A

Op. Margin: 72%

Inductance: 14.3 mH

Stored Energy: 386 kJ

of Magnet: 28

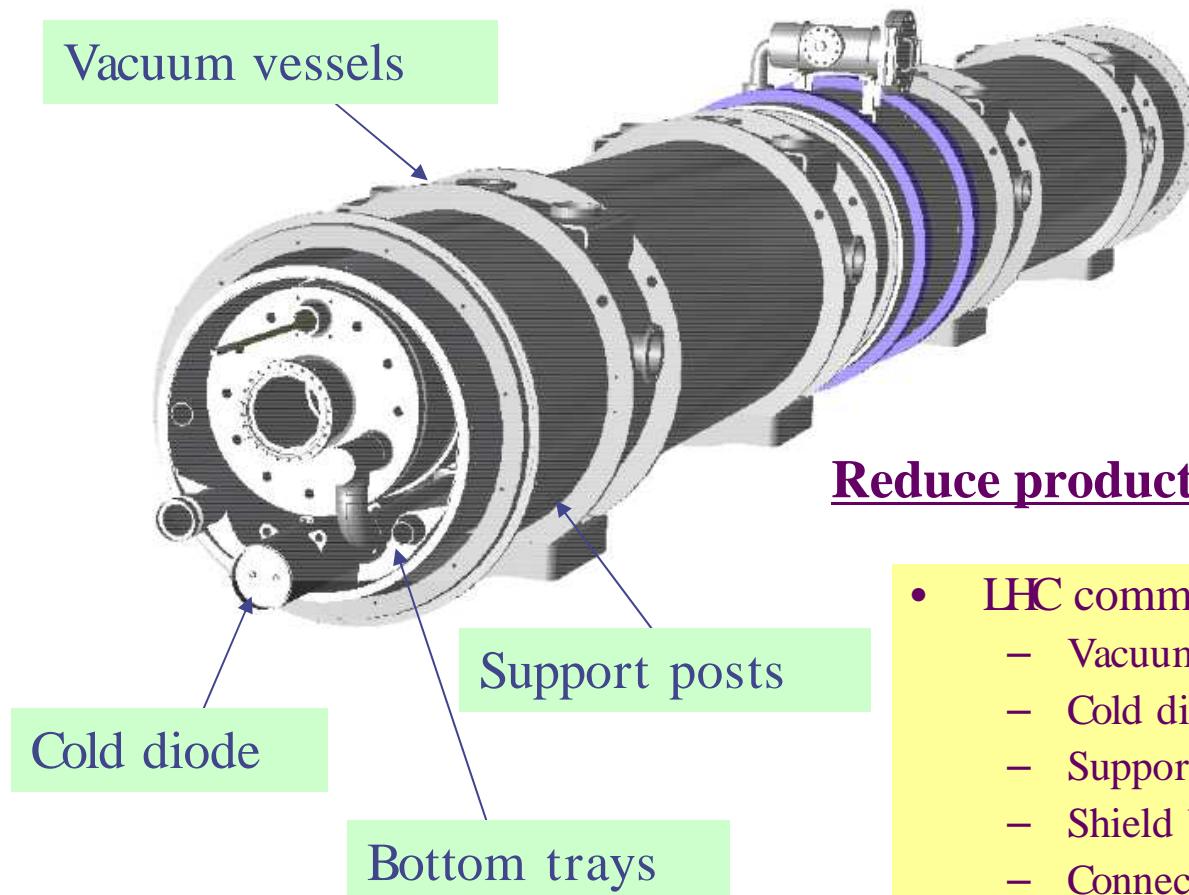
SC Cable: NbTi/Cu for LHC

Dipole Outer - L

10



Structure design of the Cryostat for SCFM

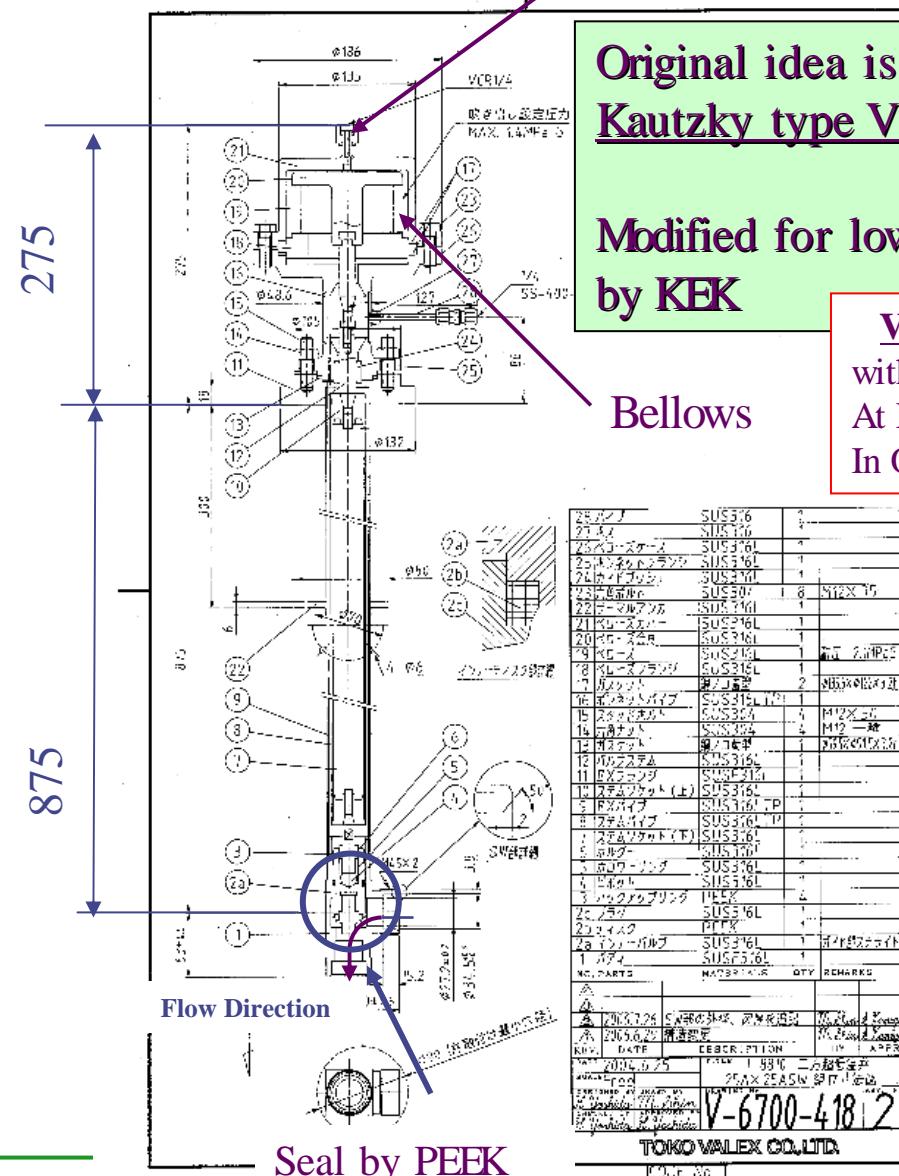


Reduce production cost of the cryostat

- LHC common parts
 - Vacuum vessel (modified)
 - Cold diode for protection
 - Support post
 - Shield bottom tray (modified)
 - Connecting Sleeve
- Two magnets assemble into one cryostat
 - F & D magnets (doublet optics)



Quench Relief Valve



Reference pressure input

Original idea is came from Kautzky type Valve @FNL

Modified for low temperature use by KEK

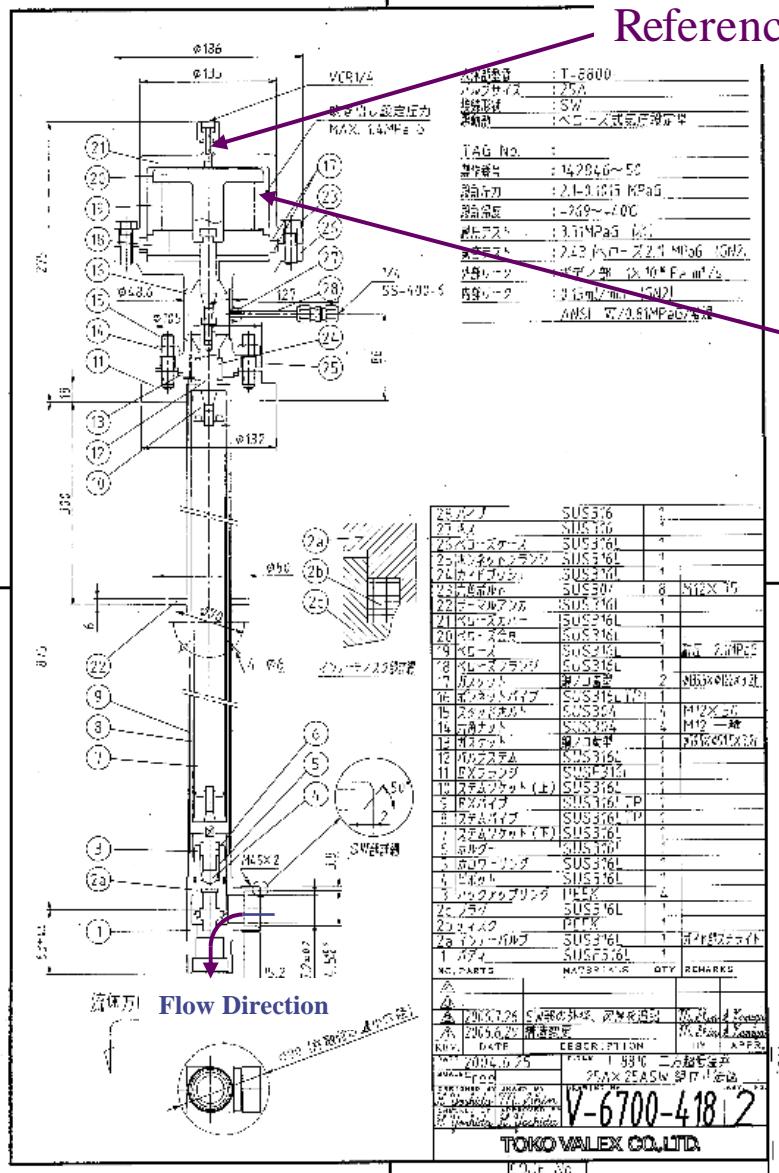
Variable Relief Pressure
with reference pressure
At Pre-cooling: 1.2 MPa
In Operation: 0.45 MPa



Assembling of Inter-connect Cryostat For Quench Valve



Quench Relief Valve



Reference pressure input

Bellows

Variable Relief Pressure

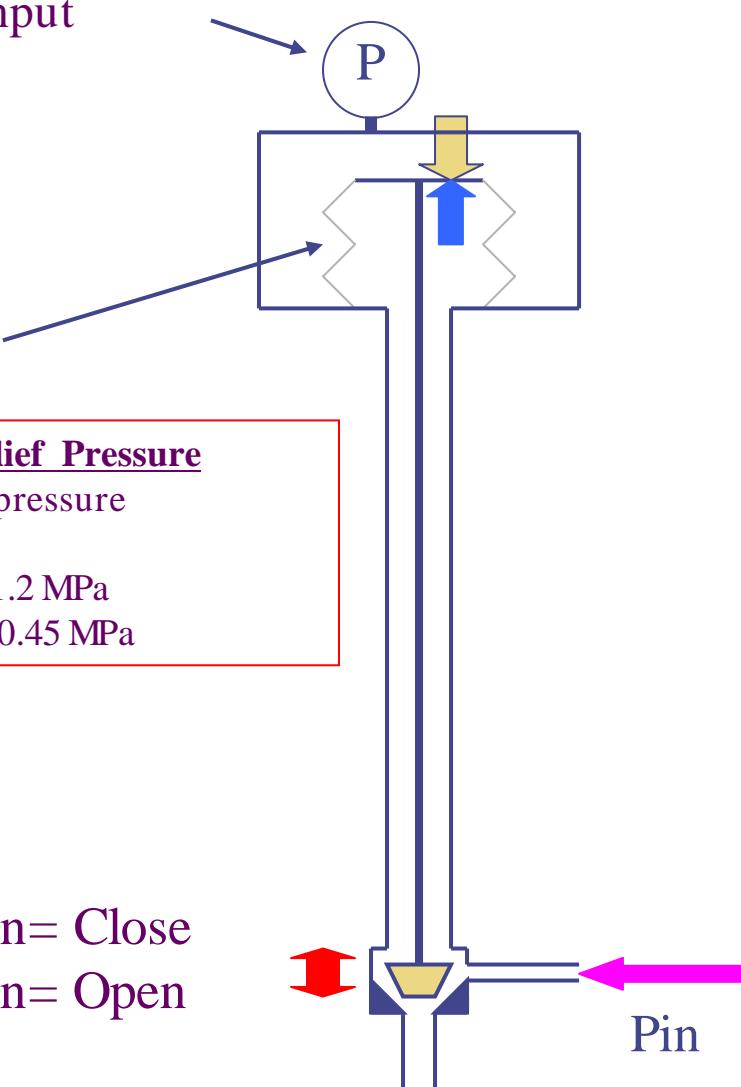
by reference pressure

Pre-cooling: 1.2 MPa

In Operation: 0.45 MPa

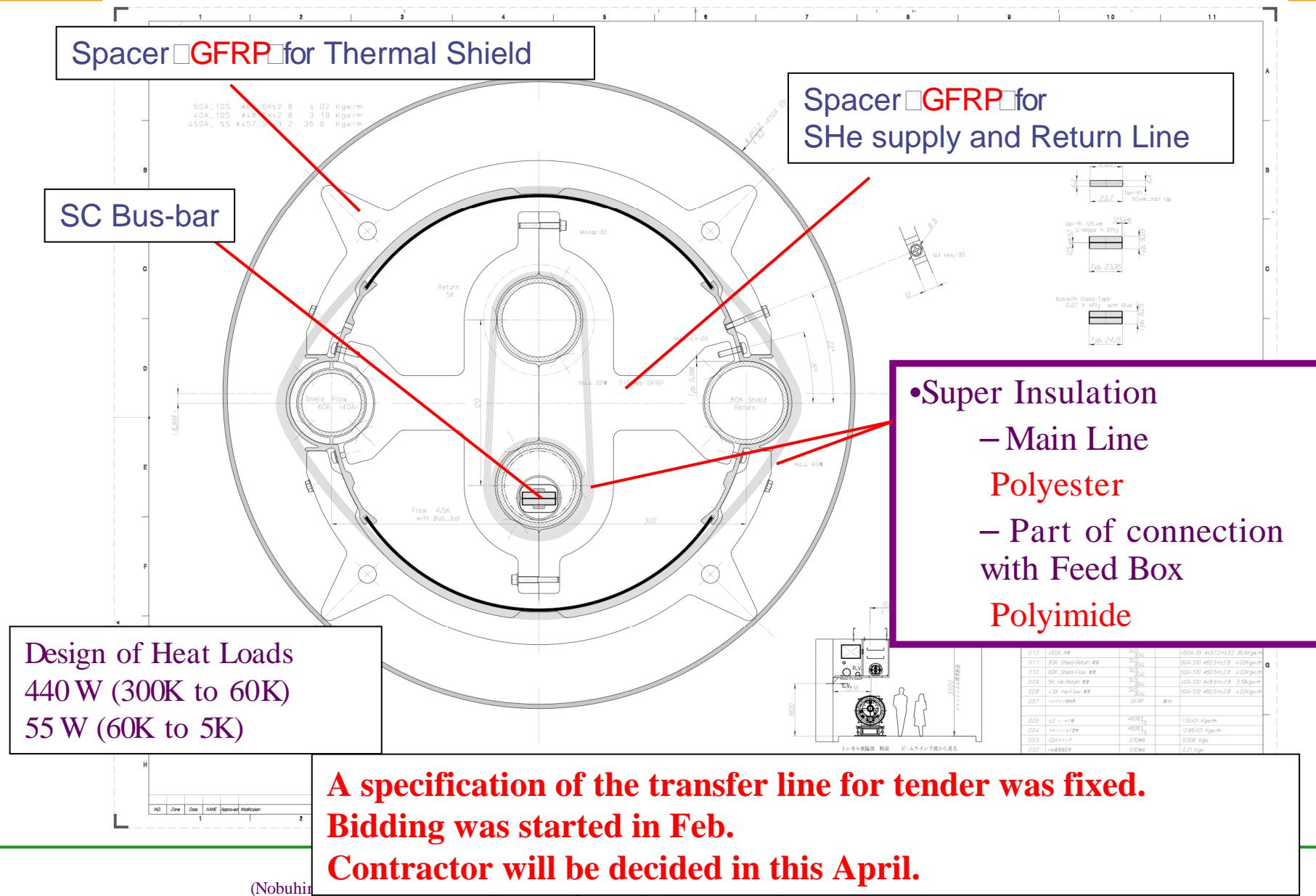
P >Pin= Close
P <Pin= Open

Conceptual Idea





Cross section of the Transfer Line





Transfer Line with SC bus-bar

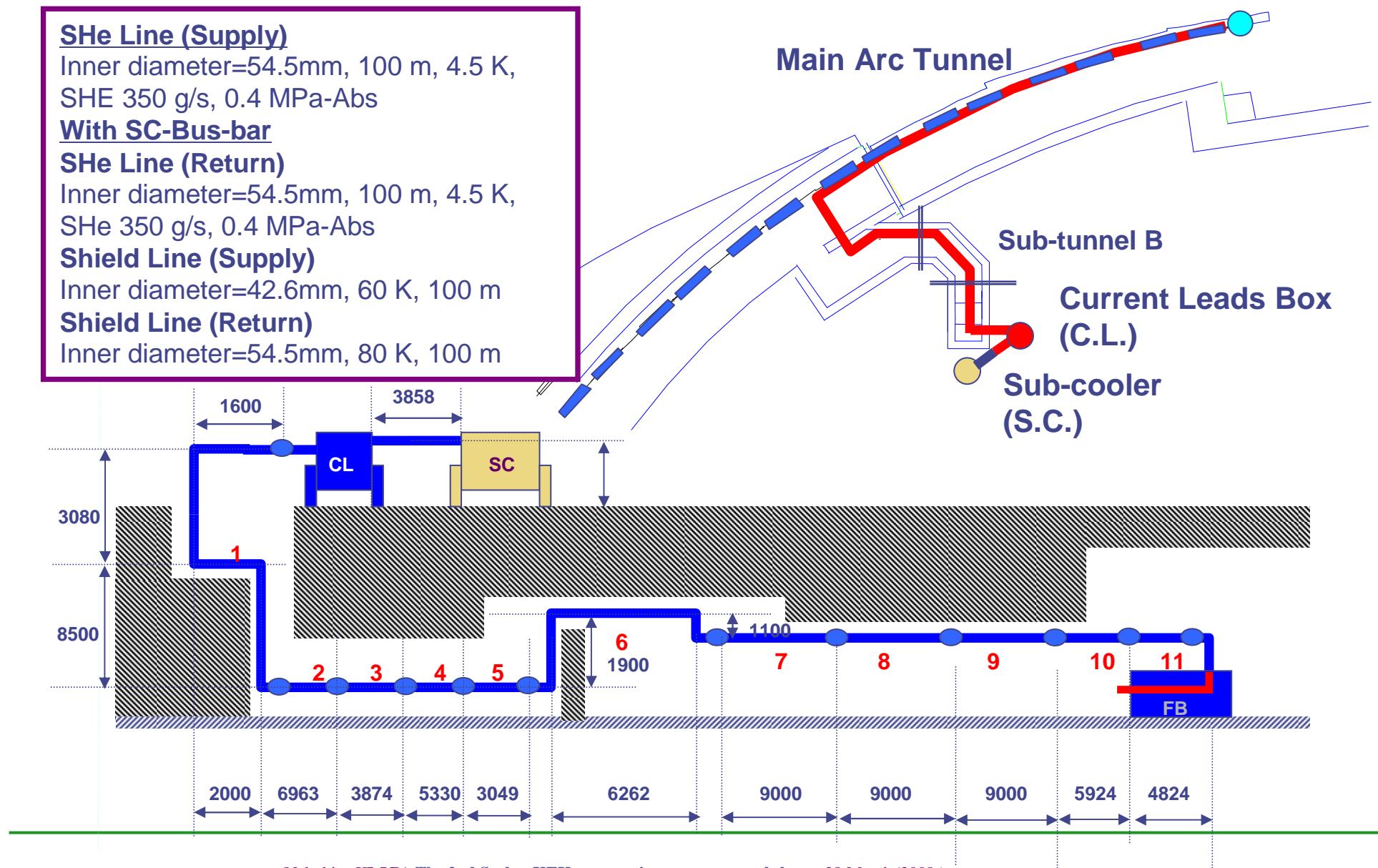
Feed Box (F.B.)

SHe Line (Supply)
Inner diameter=54.5mm, 100 m, 4.5 K,
SHE 350 g/s, 0.4 MPa-Abs
With SC-Bus-bar

SHe Line (Return)
Inner diameter=54.5mm, 100 m, 4.5 K,
SHE 350 g/s, 0.4 MPa-Abs

Shield Line (Supply)
Inner diameter=42.6mm, 60 K, 100 m

Shield Line (Return)
Inner diameter=54.5mm, 80 K, 100 m





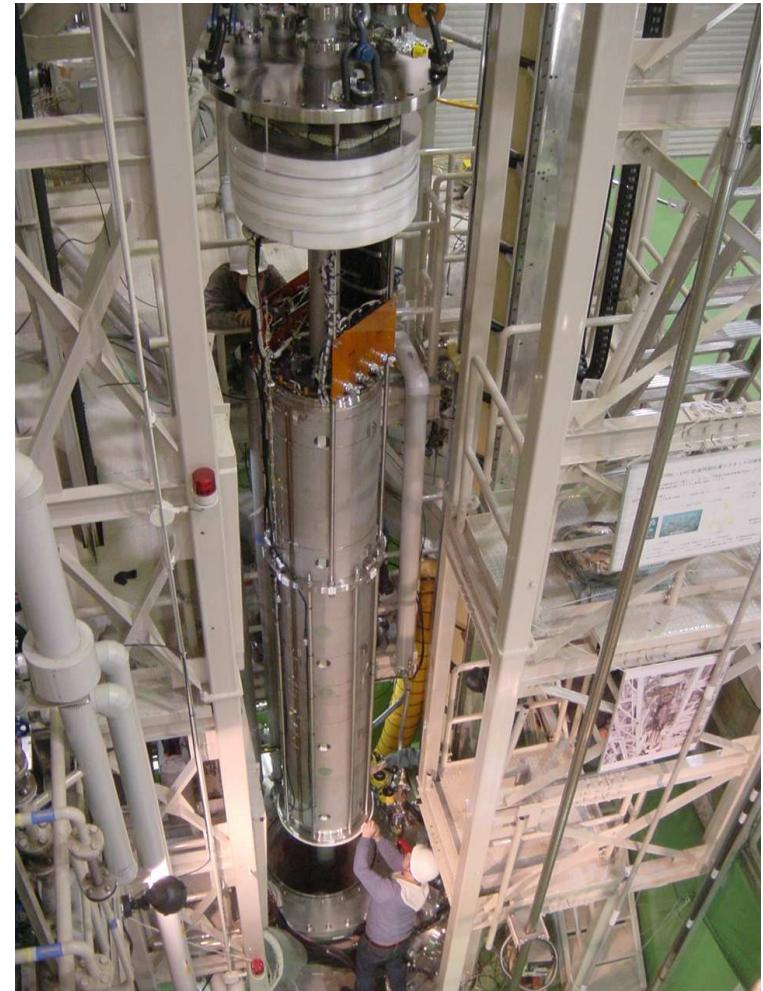
Contents

- Introduction
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Status on SCFM

- In our project plan, thirty two SCFMs products for T2K.
- All of the SCFMs are tested using with horizontal cryostat in KEK.
- In this time, twenty eight SCFMs have finished performance test without serious troubles.
- These magnets install in the tunnel with cryostats.
- Left four SCFMs will be done performance test until mid of 2008.
- These four SCFMs will use as reserve magnet.





Cryostat Assembly





Status on Dublet Cryostat



Twelve cryostats are completed, and installed in the tunnel.
Left two cryostats and two reserved cryostats are
now under production at Mitsubishi Electric Co..



Status of Transfer Line



Production of Transfer line
was started on end of 2007



First of four units of transfer line
were delivered for KEK.
These units will be installed
in the tunnel on this July.



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Status on Doublet Cryostat (1)



On 8th Feb. 2008, installation work of the doublet cryostats were started in the tunnel at Tokai.
The above photograph shows the cryostat into the tunnel by crane.
Left three photographs show the first cryostat in the tunnel.



Status on Doublet Cryostat (2)



Doublet cryostats in the tunnel.

eleven doublet cryostats (11/14) were set on the beam line.

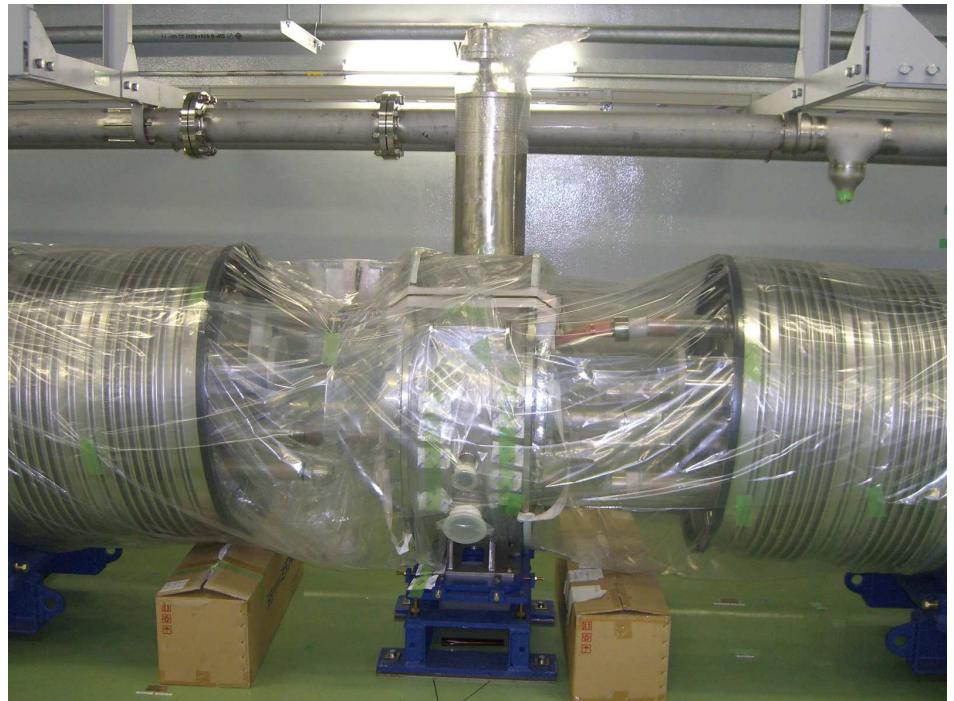
The cryostats were tighten by bolts.



Status on Quench Valve Cryostat



Four Quench Valve inter-connect cryostats were transferred in to the tunnel



Three Quench Valve inter-connect cryostats were set on right position in the system.



Status on other component



Quench pressure relief lines were installed in the tunnel.
The lines will be connected with quench relief valve cryostat
and stack.



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Summary

- Twenty eight SCMs have finished performance test without serious troubles.
- Twelve doublet cryostats have been completed, and installed in the tunnel.

Left two doublet cryostats and two reserved cryostats are now under production at Mitsubishi Electric Co..

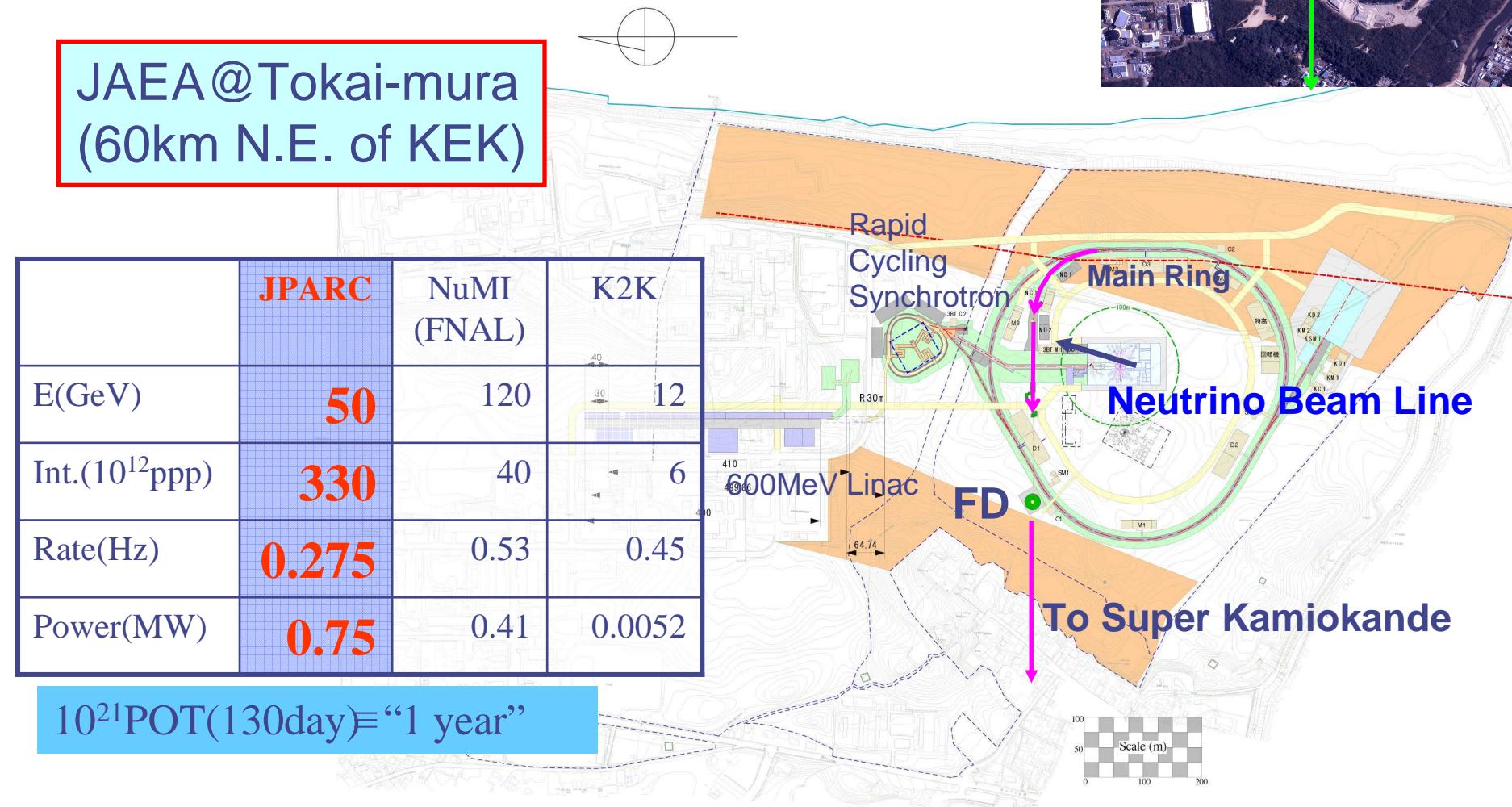
- Three quench relief valve cryostats and two corrector magnet cryostats have installed in the tunnel.
- First of four units of transfer line were delivered for KEK. These units will be installed in the tunnel on this July.
- A cold box manufactured by LINDE and components such as a Helium compressor will be delivered Tokai at mid of May 2008.

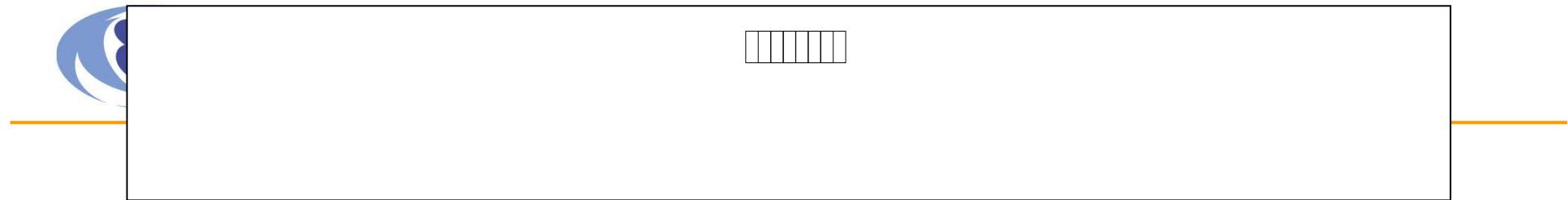




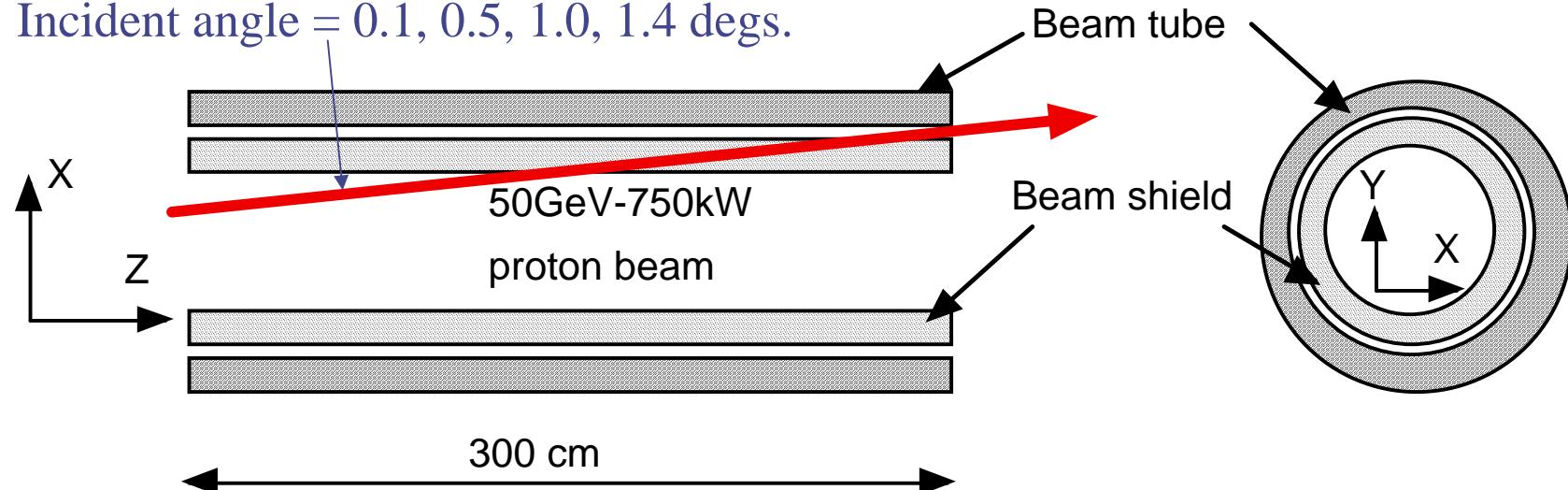
Introduction

J-PARC project and Neutrino beam line





Incident angle = 0.1, 0.5, 1.0, 1.4 degs.



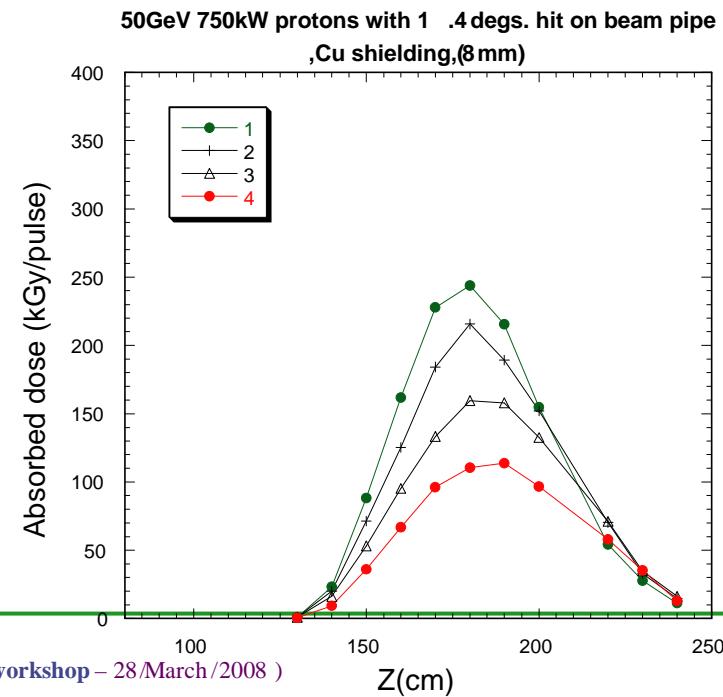
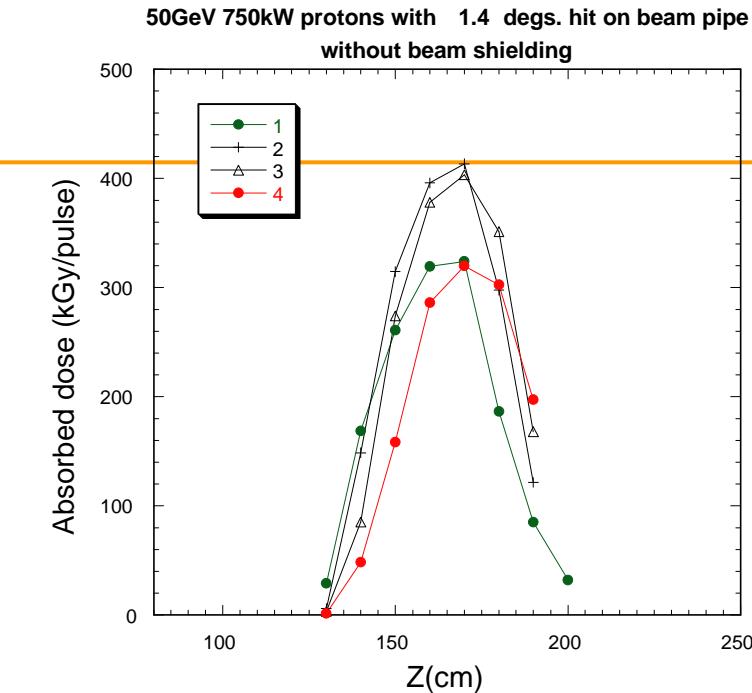
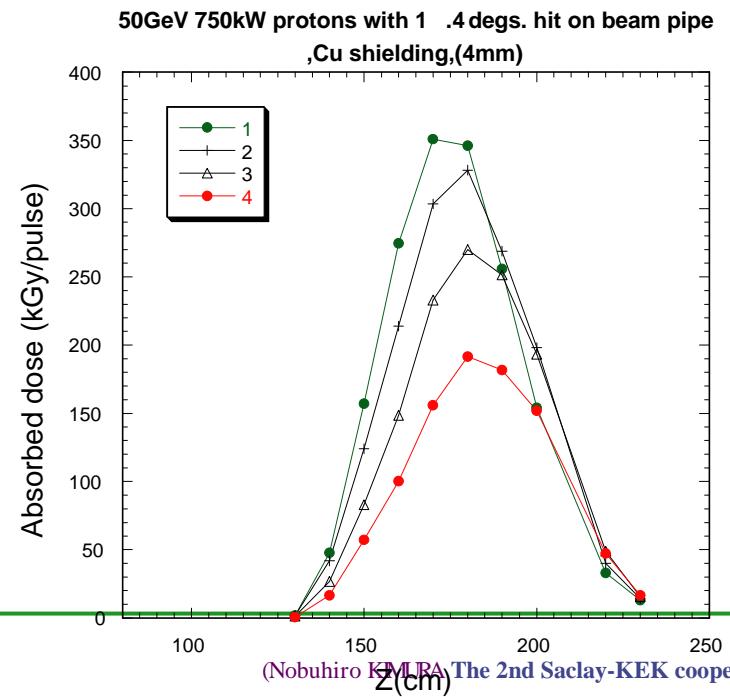
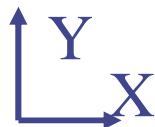
The beam is distributed uniformly rectangular area. $dX=1\text{cm}$, $dY=2\text{cm}$

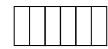
-	Beam shield	Beam tube
Material	Copper	Stainless
Radius (mm)	~74	76~84
Thickness(mm)	0, 4, 8, 10	8



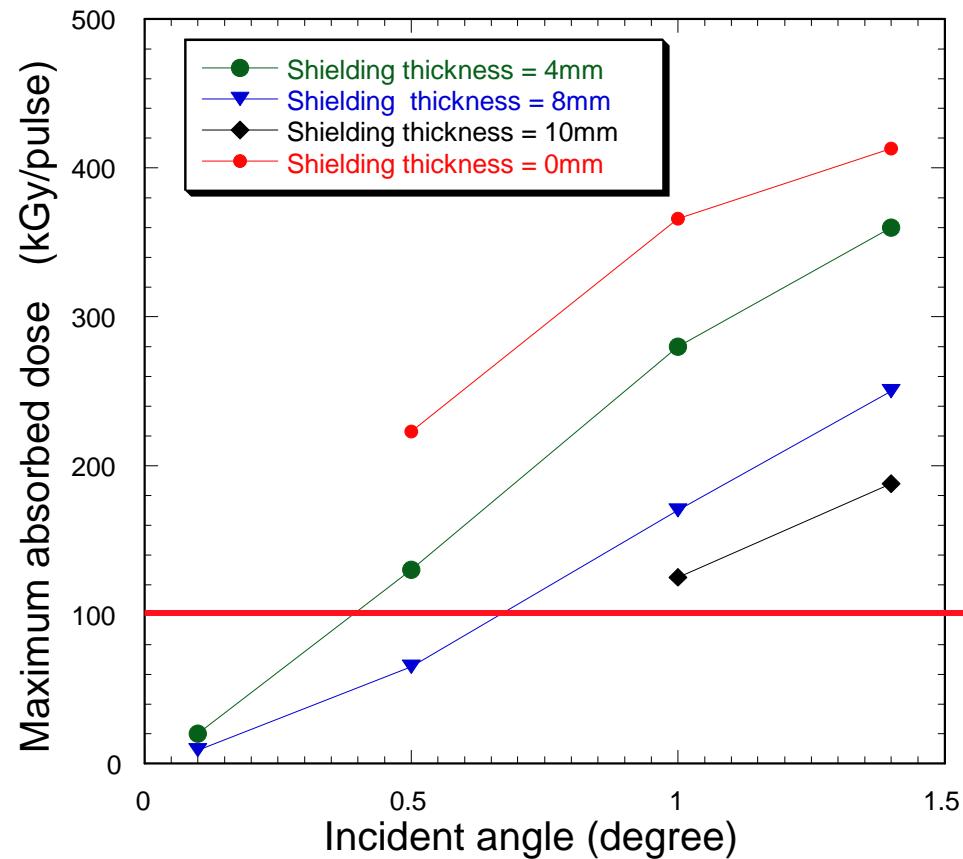
Heat input

QuickTime[®] C²
TIFFÄILZWÄj èLiçÈvÈcÈOÈaÈÄ
C™Ç±ÇÄsÈNE'EEÇ%@ÇÈÇzÇ%Ç...ÇÖiKovÇ-ÇÅAB



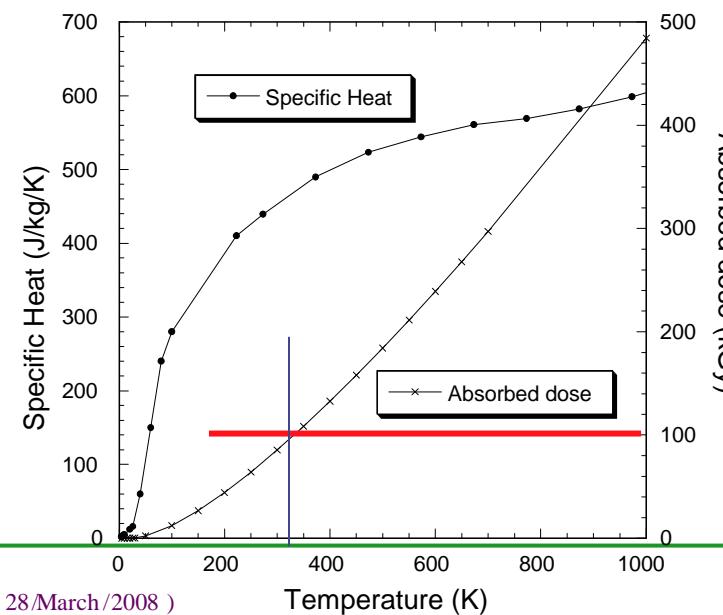


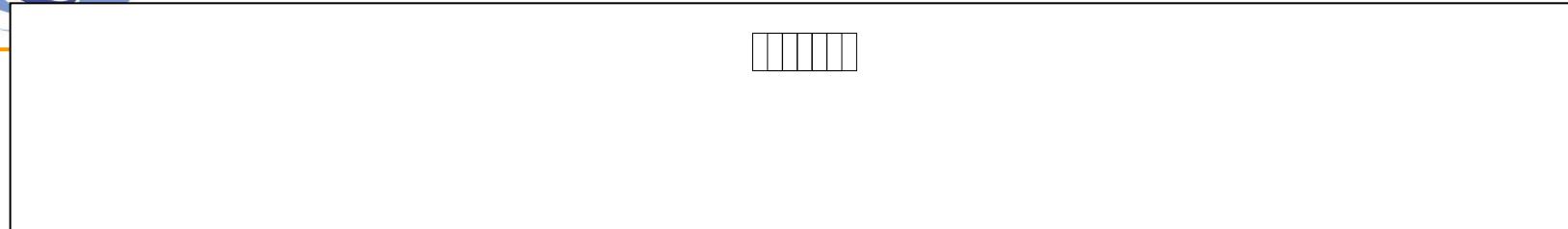
Simple estimation: $\sigma_{th} = \frac{2-\nu}{3(1-\nu)} \cdot E \cdot \int_0^T \alpha \cdot dT$ 100kGy/pulse



$$\sigma_{th} = 0.52 \text{ GPa}$$

~ Tensile Strength @ 300K

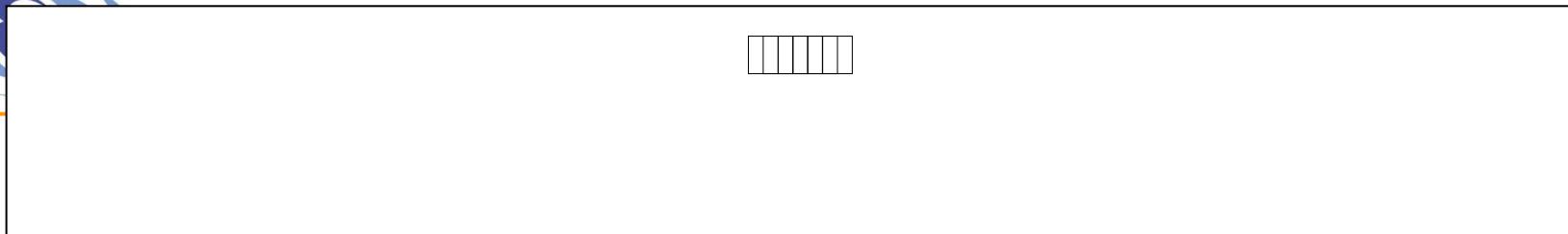




- 
 -  OFF 1.4
 - 10mm 

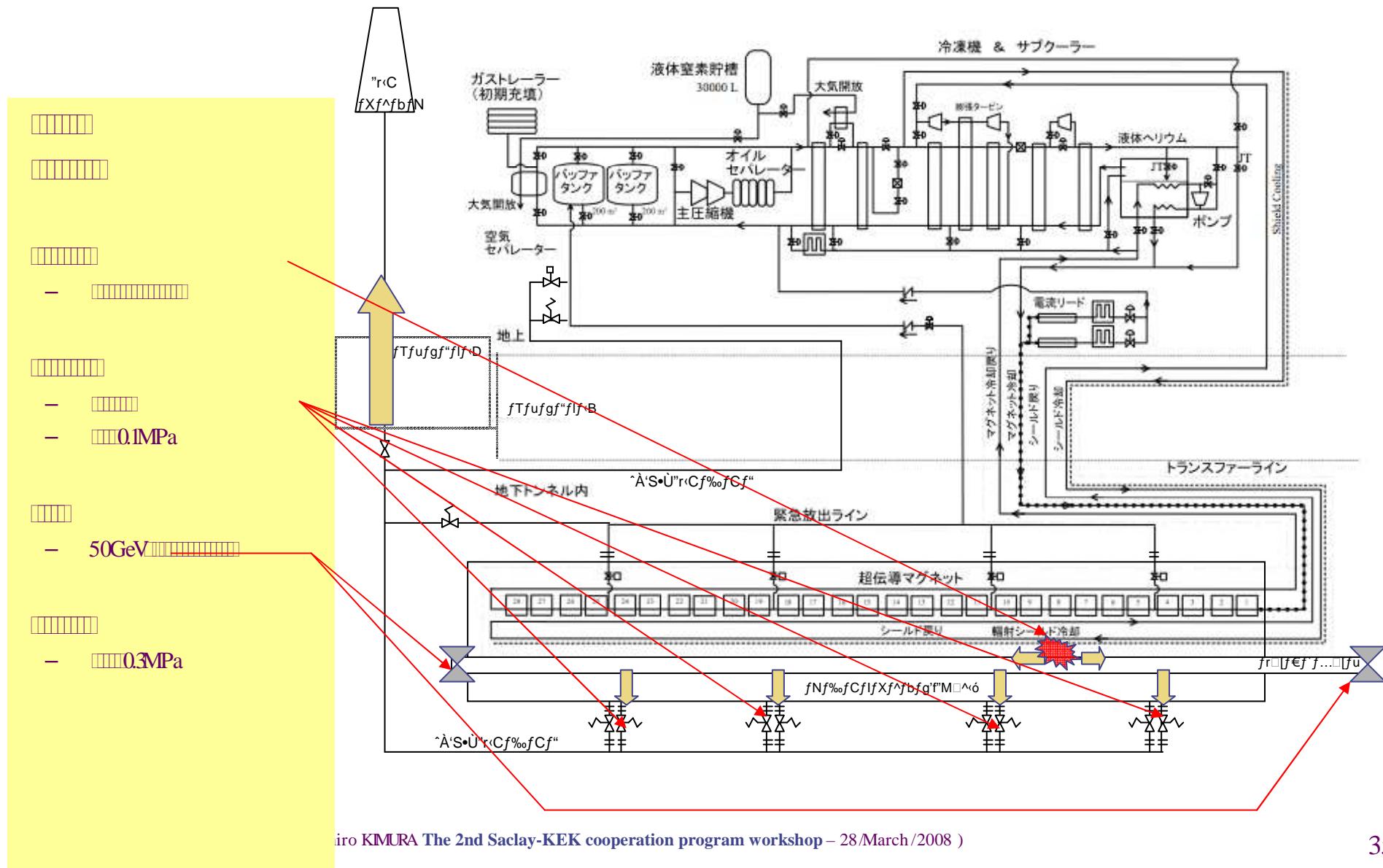
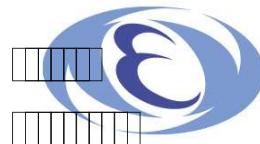
- 
 -  π
 -  130mm
 -  10mm

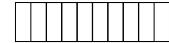
- 
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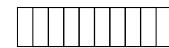
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- ◆
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-



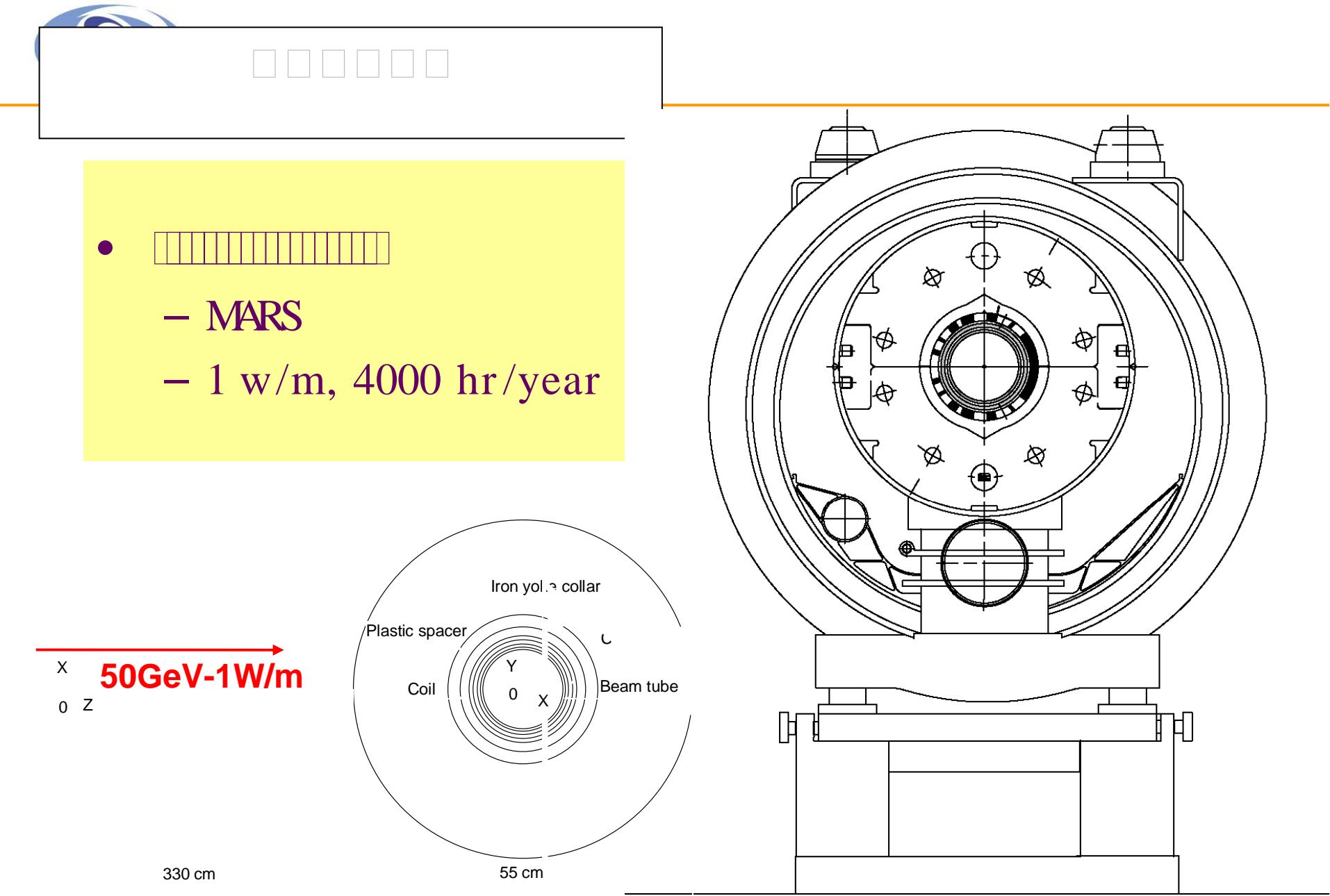




- - mm OFF
 -
 -
- - 35 msec
 - →
 - 1 msec
- - 0.1 MPa
 - 0.3 MPa
 -

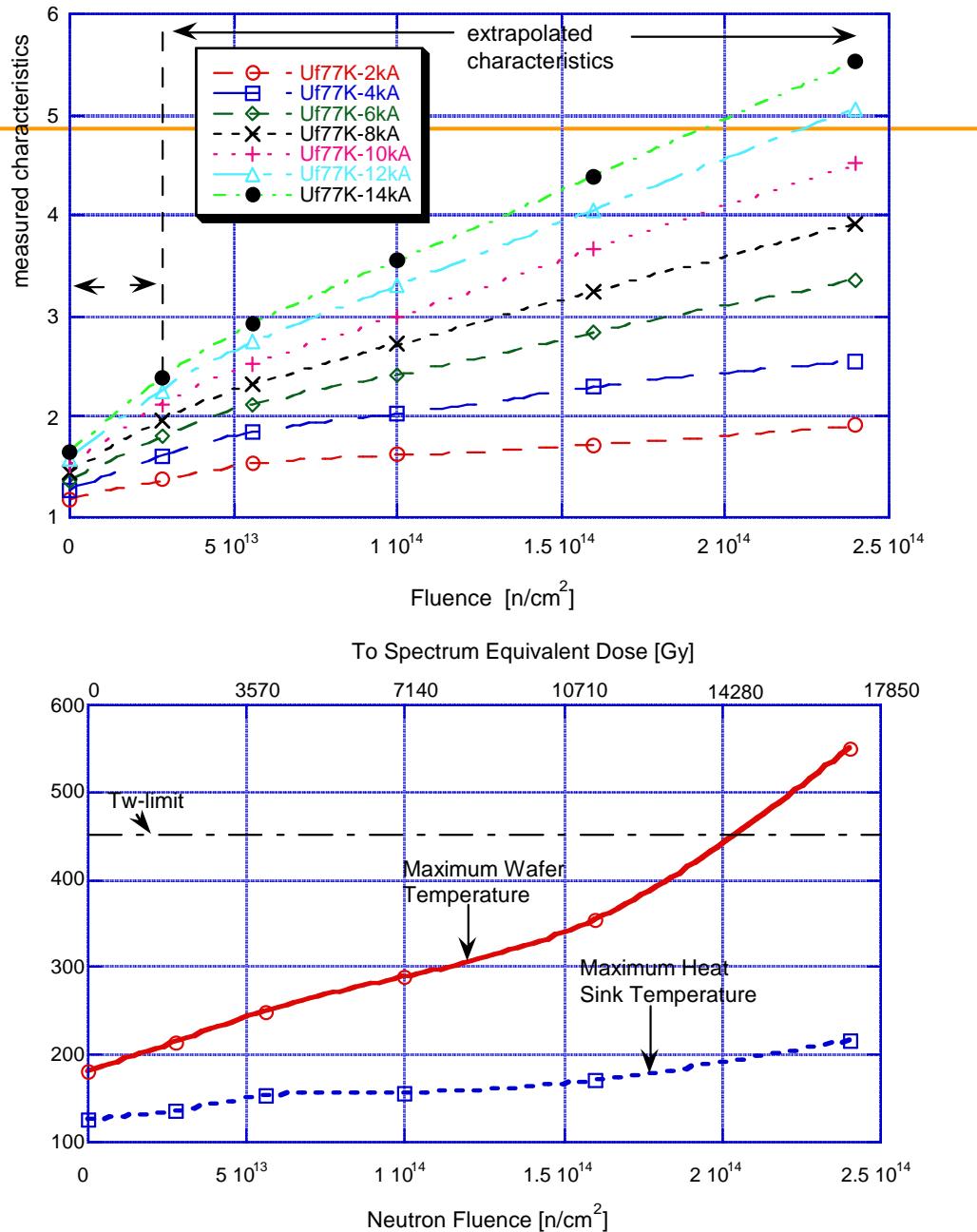


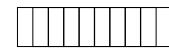
- 
 - 
 - 
 - ◆ 
 - ◆ 
- 
 - 
 - ◆ 
 - ◆ 



Radiation to Cold Diode

- Influence of Neutron to Cold Diode
 - Intensively studied at CERN by D. Hagedorn
 - ◆ Change Forward Voltage
 - Using LHC Arc Quad Assembly
 - ◆ 7.5kA Operation
 - ◆ Limit; $2 \cdot 10^{14} \text{ n/cm}^2$

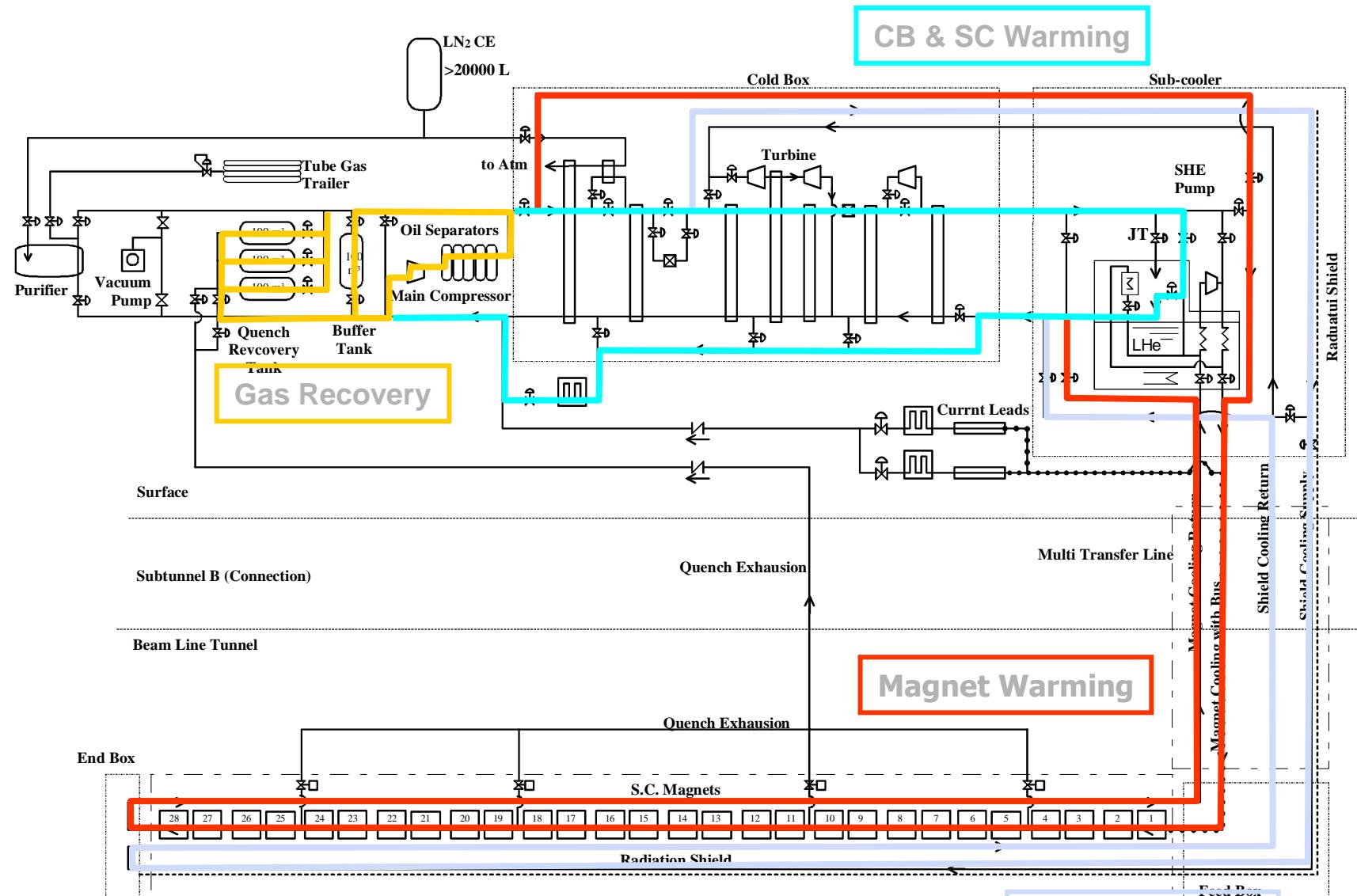




- 
- 
- 
 - ◆ 
 - ◆ 
- 
 - 
 - ◆ 
 - ◆ 

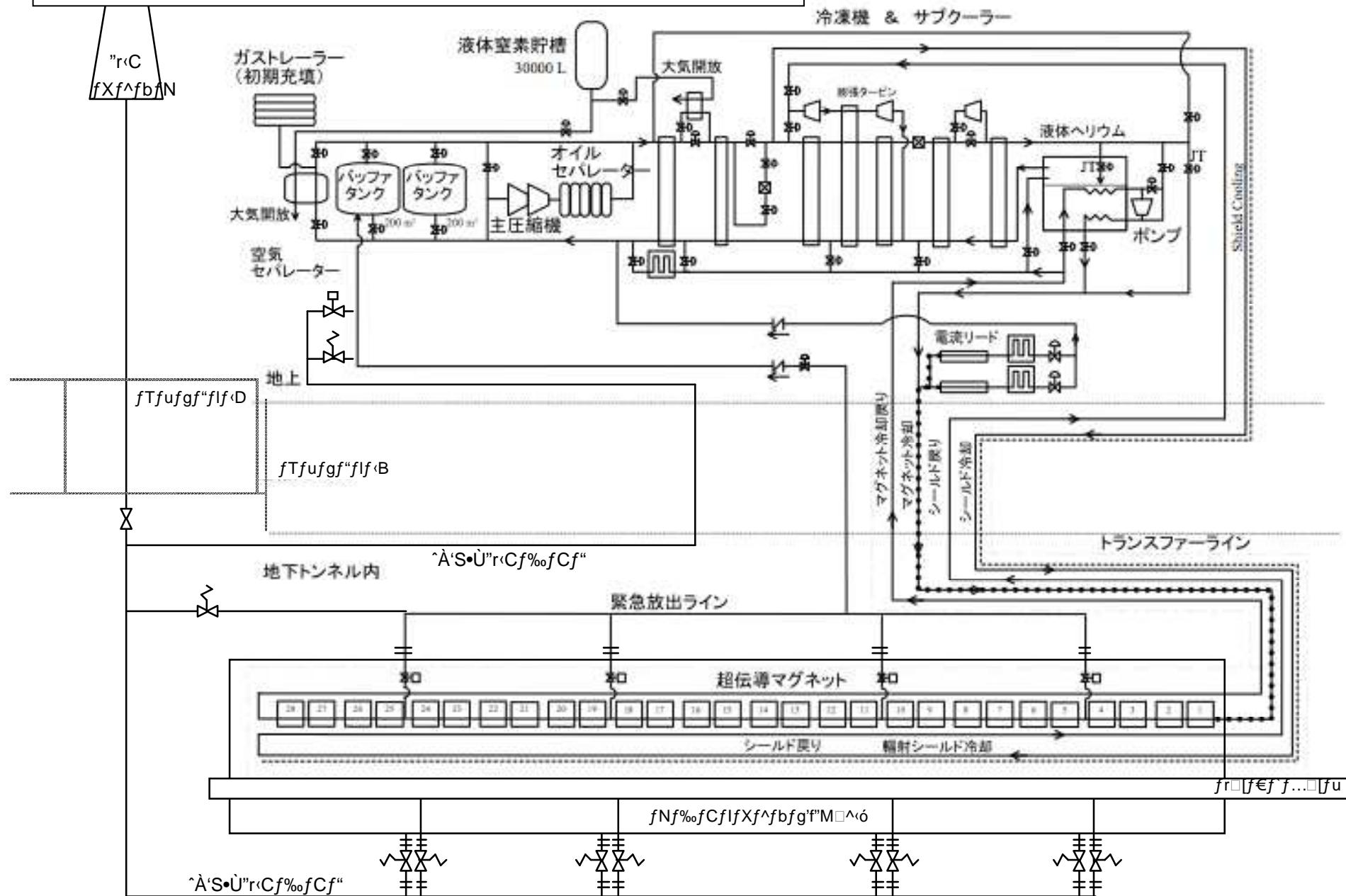


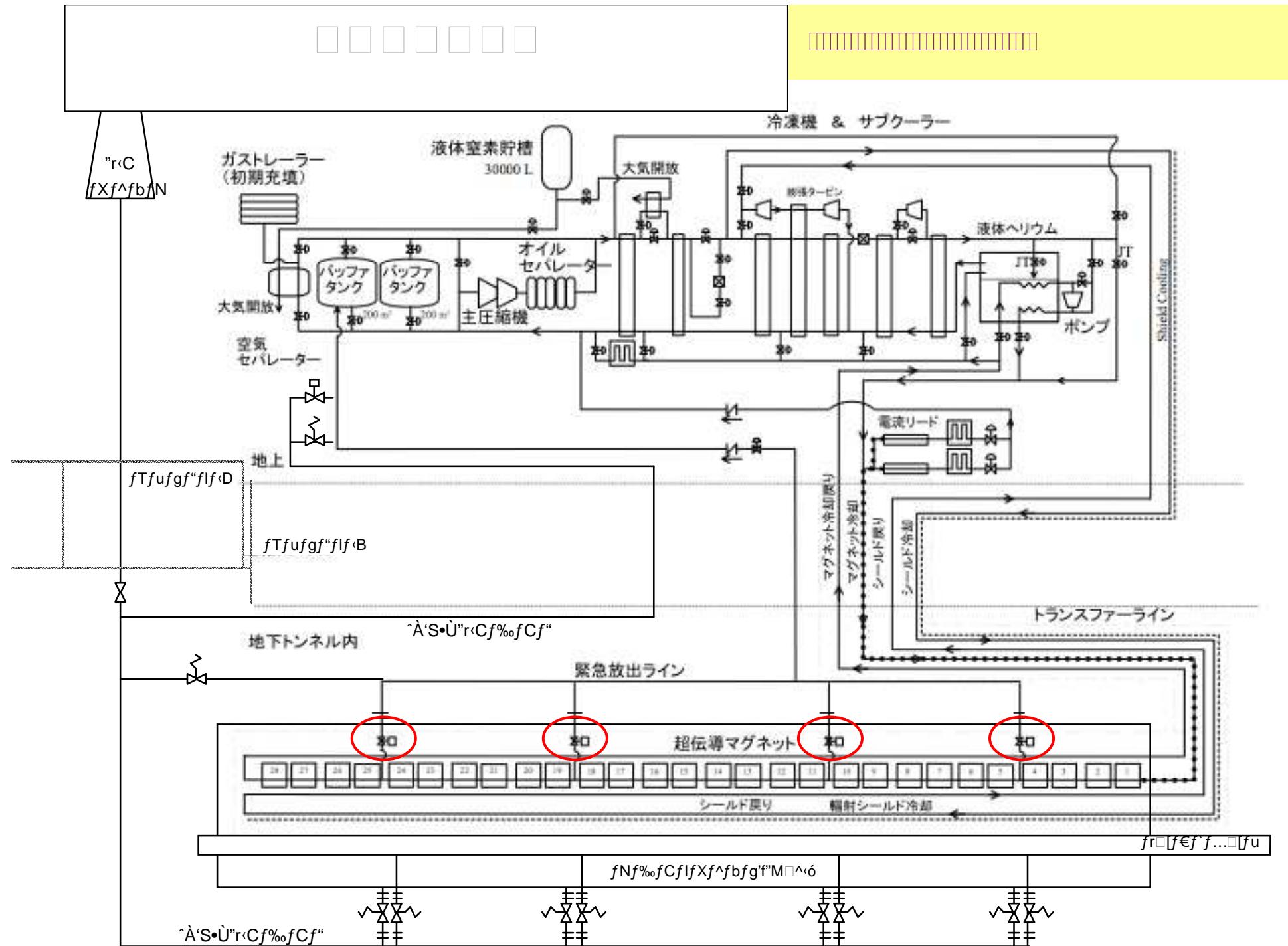
5. Operation - Warming



□ □ □ □ □

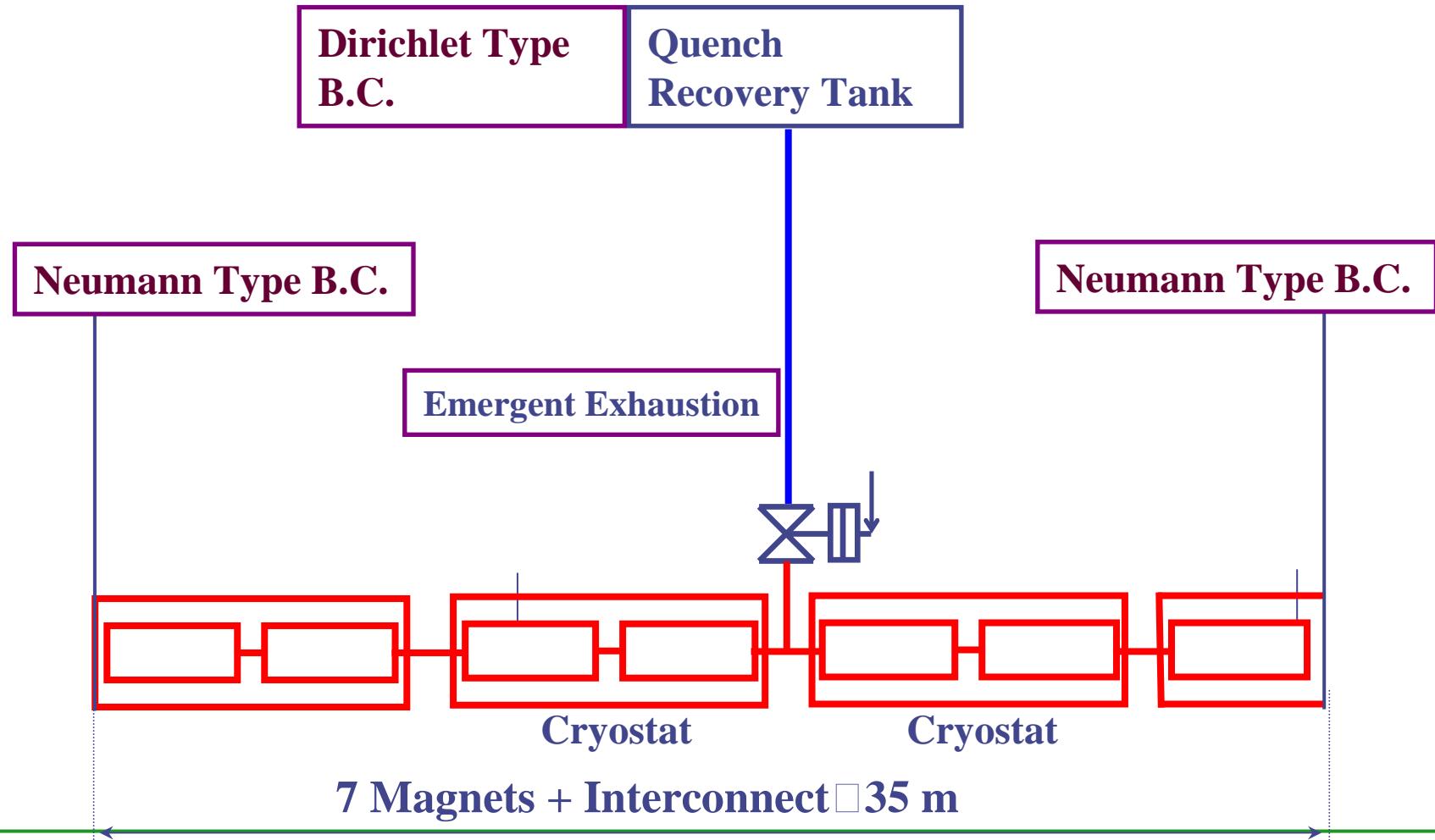
Motor Power □ 1.2kW@4.5K □ 2.5kW@80K
Vertical Helium Pump, 300~400 g/s 400kPa

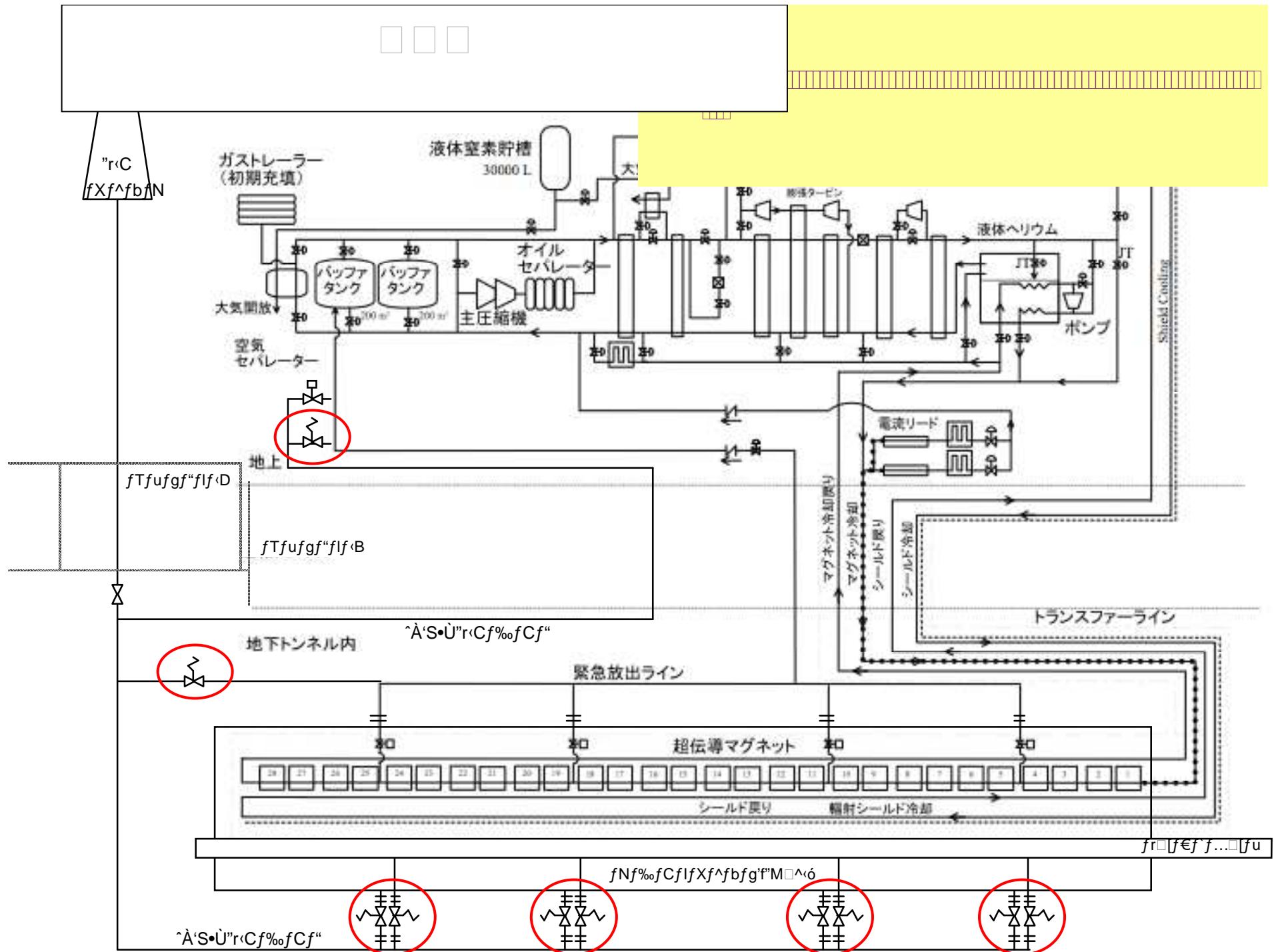


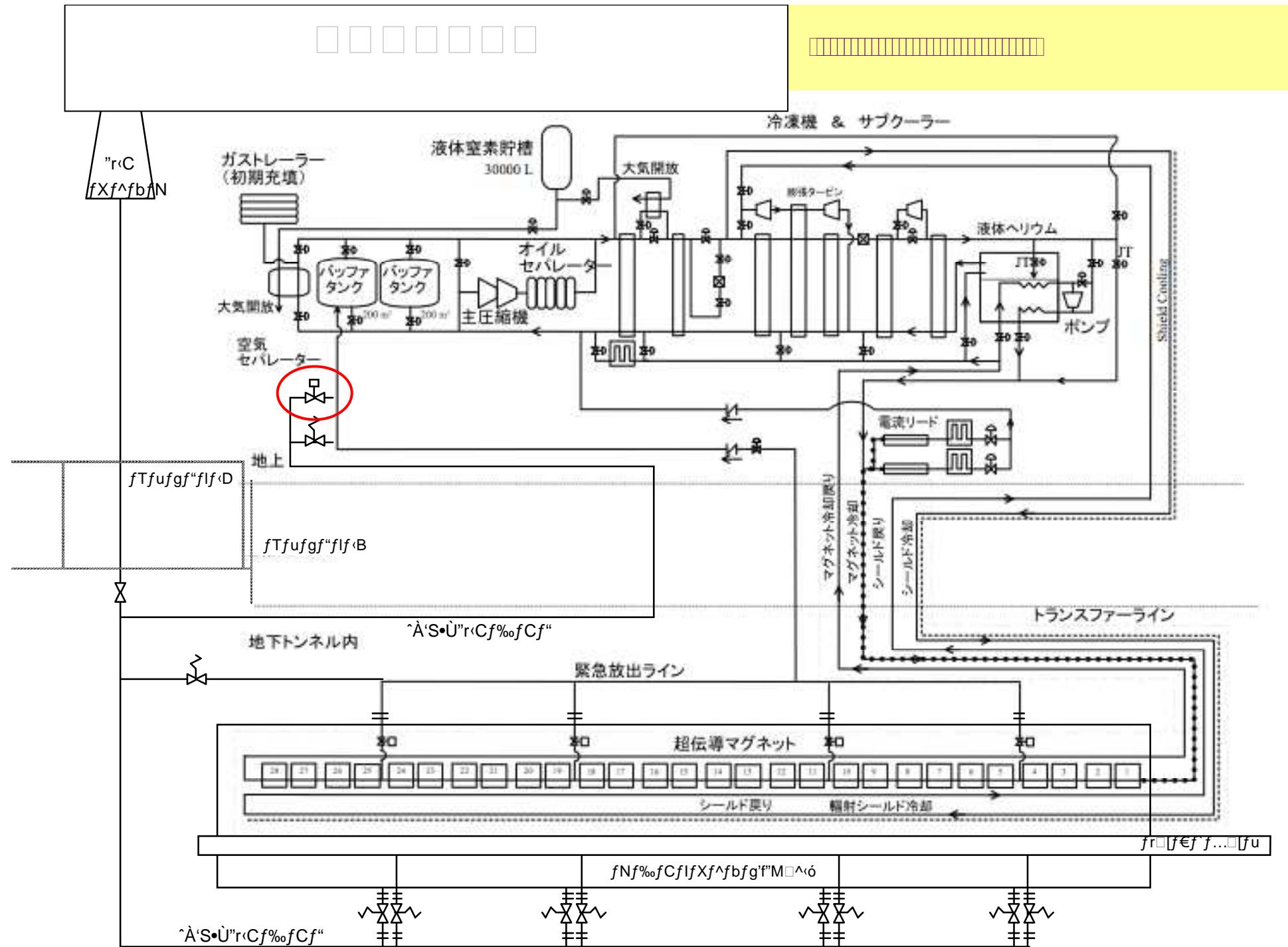


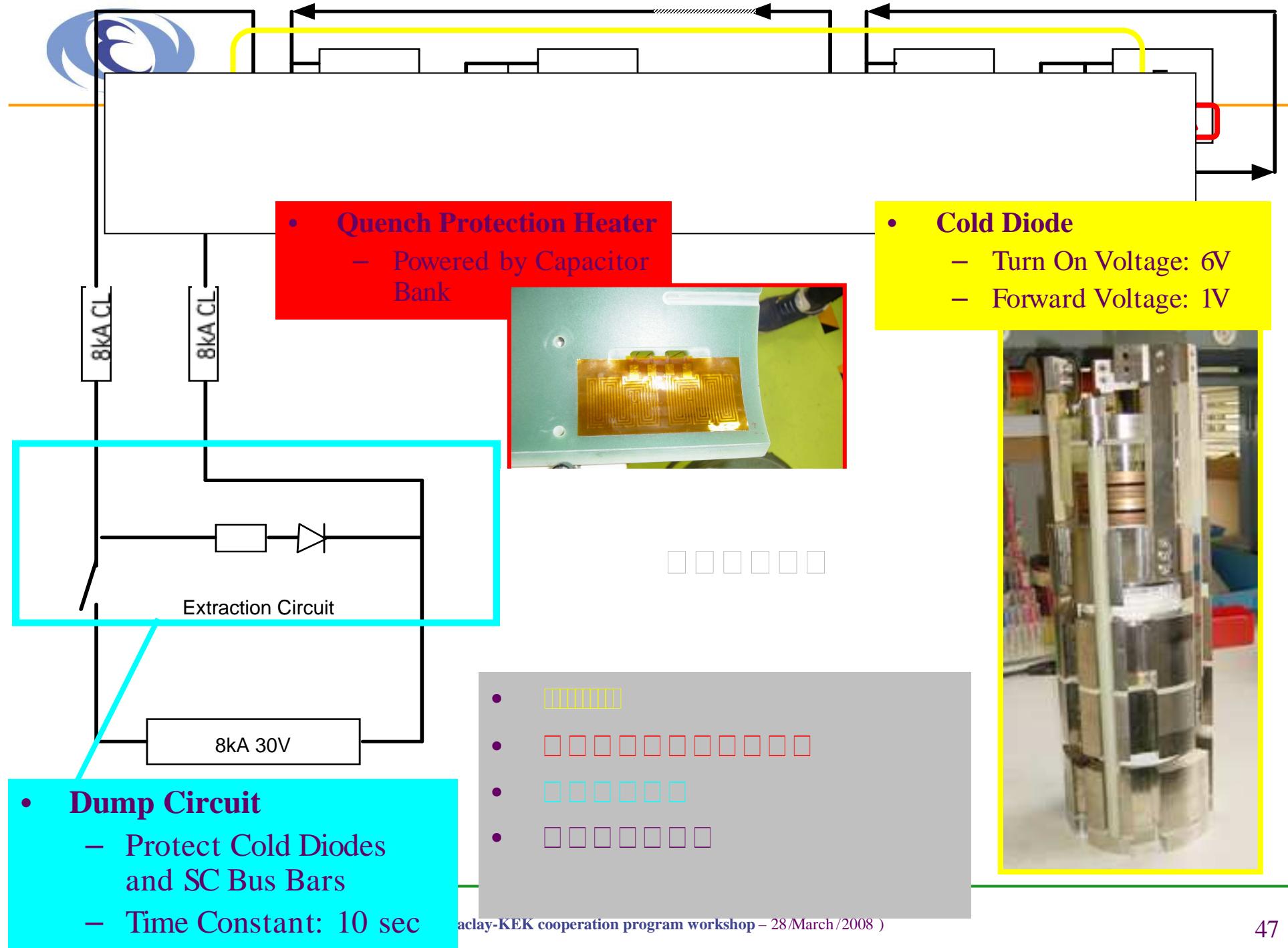


Strict analytical model









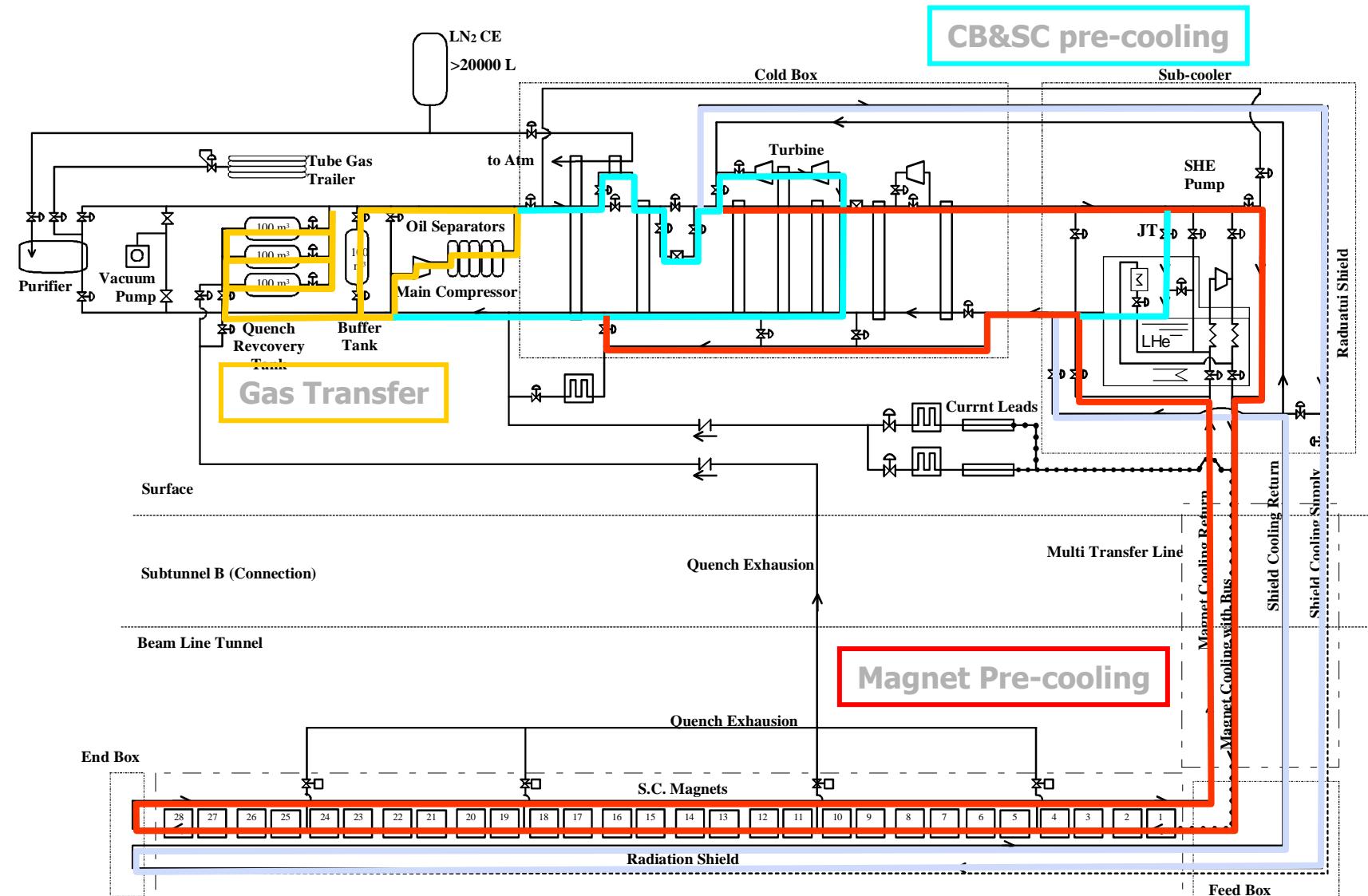


Summary of Load (Magnet & Transfer Lines) to Cryogenic System

	4.5 K Level	Remarks	80 K Level	Remarks
Coolant	SHe	4.5 K \square 0.4 MPa	He Gas	60 \square 100 K, 1.2 MPa
Heat Load Estimation	336 W	Including beam loss of 150 W transfer line of 55 W	1419 W	Including transfer line of 440 W
Current Lead	1.0 g/s	8000A	-	
+ 20 % Contingency	403 W + 1.1 g/s		1703 W	
Cold Mass	204 ton	Iron basis	6.8 ton 2.5 ton	Aluminum basis Iron basis
+ 10 % Contingency	225 ton	Iron basis	7.5 ton 2.8 ton	Aluminum basis Iron basis
Inventory	3550 t		1620 t	
+ 10 % Contingency	3900 t		1780 t	
Pressure Drop	84 kPa	300 g/sec, 4.5 K, 0.4 MPaAbs	36 kPa	40 g/sec, 80 K, 1.35 MPa
Design Pressure	>1.4 MPa(G)		>1.4 MPa(G)	

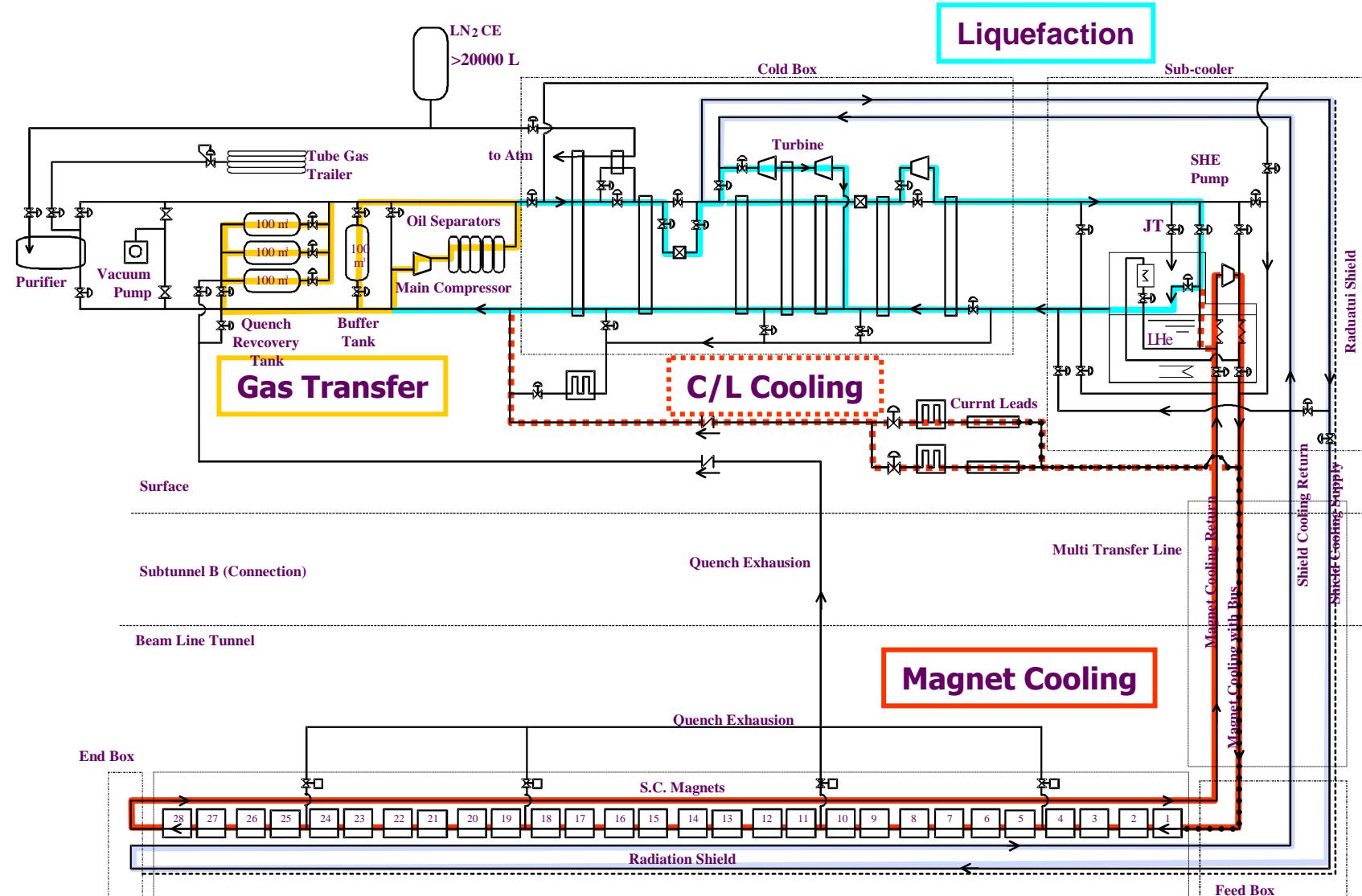


Operation – Pre-cooling



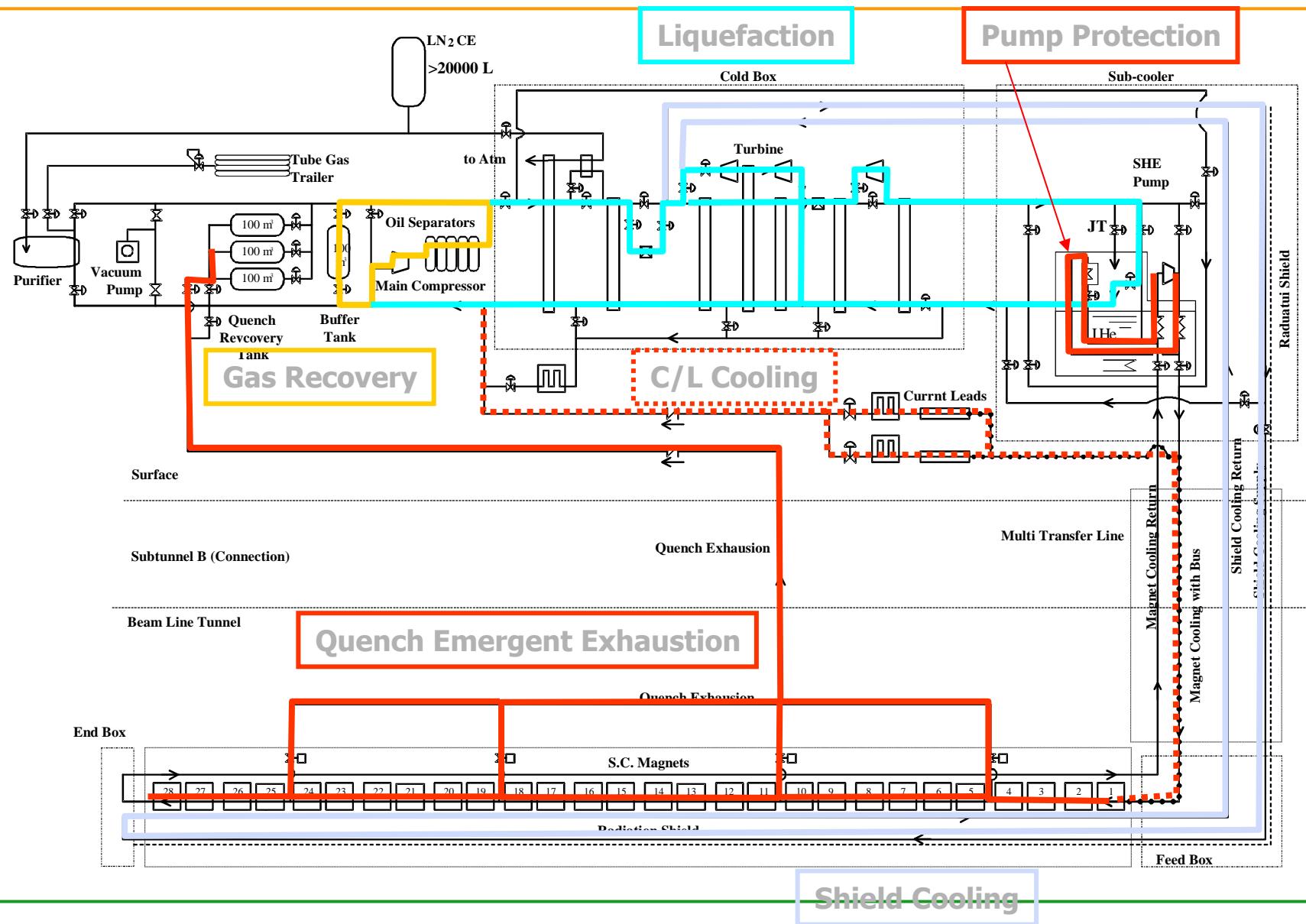


Operation – Magnet Excitation (Steady state)





Operation - Quench



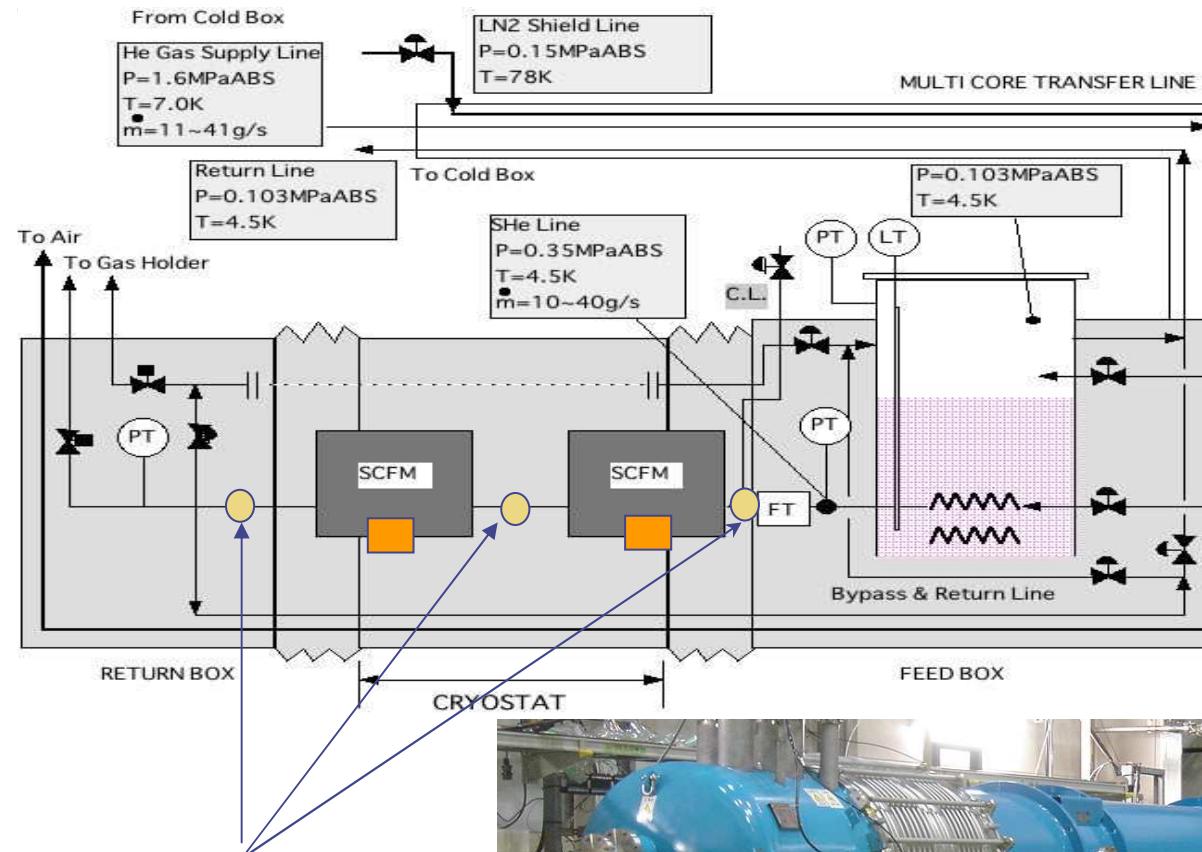


Schedule

	2005	2006	2007	2008
Cryostat w/ 2-SCFM	1 (proto)	6 (12 Mag.)	6 (12 Mag.)	2 & Install
Transfer Line				Install
Refrig.				Install
PS				Install
Corrector Magnet				Install
Quench Detector				Install



Horizontal Test Bench with the Cryostat



Feed Box

Mass flow rate: 20 ~ 40 g/sec
Coolant: SHe
Supply temperature : 4.5 ~4.8K
Supply pressure :
0.35~0.4 MPa Abs

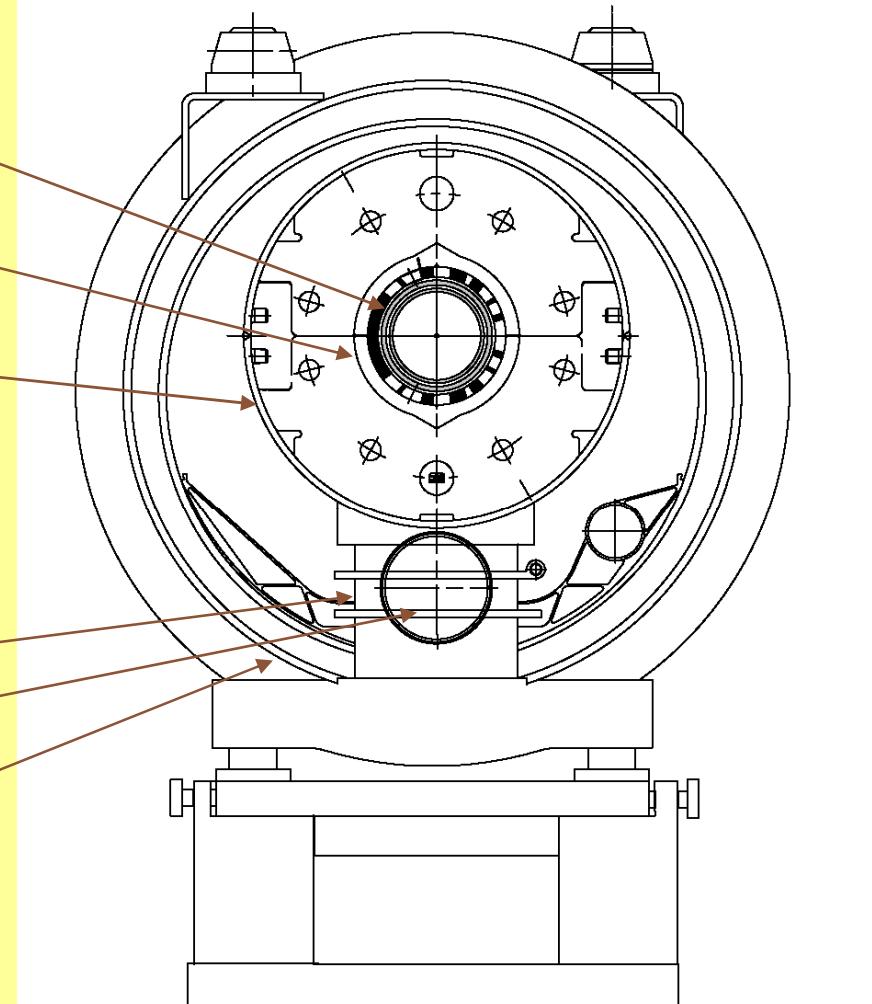




Summary of Organic Materials in Magnet and Radiation Resistance

1 w/m, 4000 hr/year

- Coil (~30kGy/y)
 - GFRP (10^7 Gy)
 - Polyimide (10^7 Gy)
- Plastic Collar (~10kGy/y)
 - Glass Filled Phenol (10^7 Gy)
- Super Insulator
 - Body (~200Gy/y)
 - ◆ Polyester (10^6 Gy)
 - End (~30kGy/y)
 - ◆ Polyimide ($6 \cdot 10^7$ Gy)
- Support Post (~200Gy/y)
 - GFRP (10^7 Gy)
- Cold Diode
 - (~200Gy/y, $1.2 \cdot 10^{13}$ n/cm²)
- Elastomer Seals
 - (~30kGy/y)





New Elastomer Seal for the Cryostat

New EPDM type Elastomer seal were developed with collaboration of KEK, JAEA and Hayakawa Rubber Co..

New Elastomer seal have been tested up to Dose=9.1 MGy with γ ray source.



Dose=9.1 MGy
1.00mm/div



Dose=0.34 MGy
1.00mm/div



Dose=2.6 MGy
1.00mm/div



Dose=0 MGy
1.00mm/div



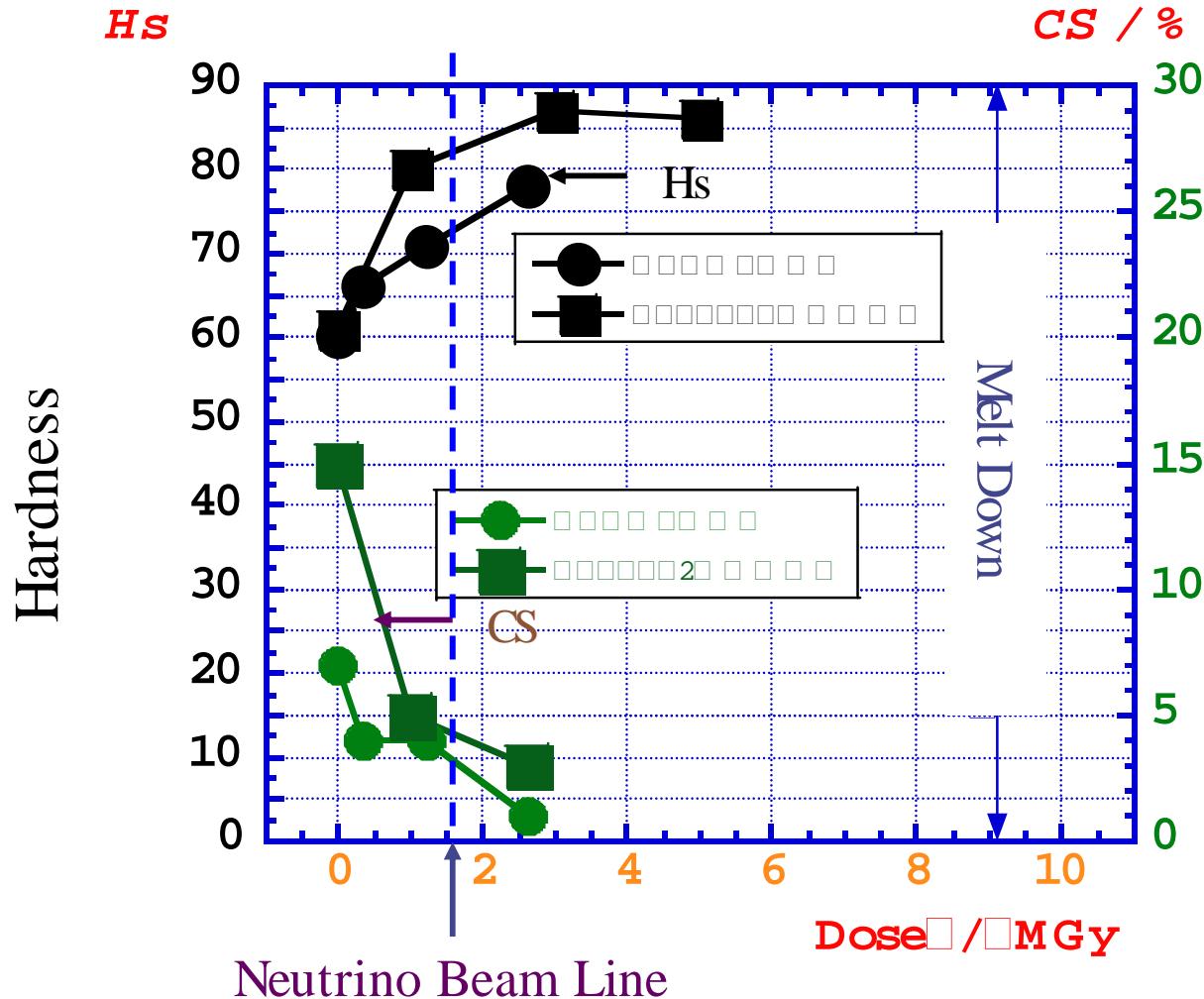
Dose=1.2 MGy
1.00mm/div



Blank
Dose=0 MGy
1.00mm/div



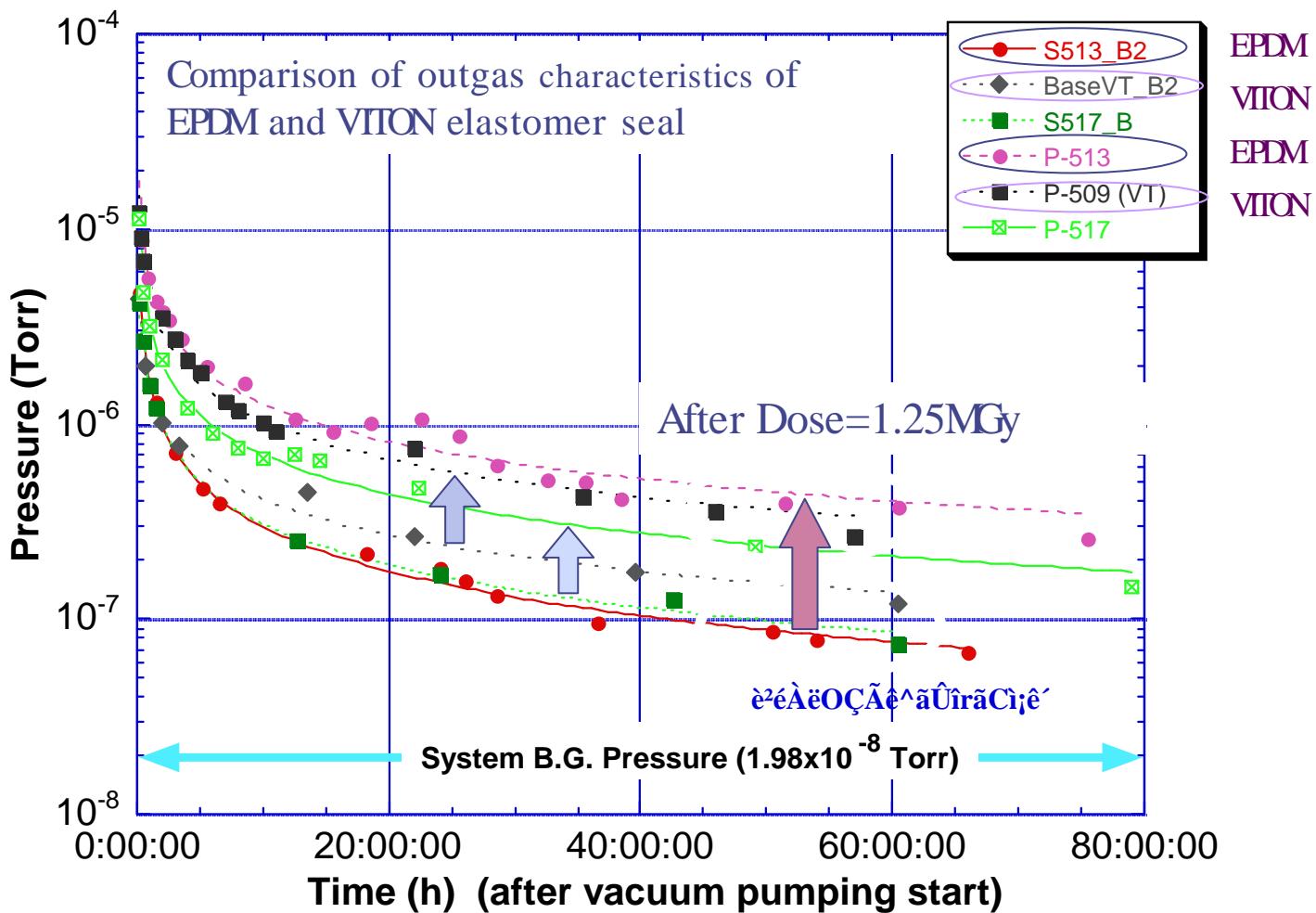
Hardness and Compression set



Compression Set



Outgas Characteristics of EPDM Elastomer Seal



It is confirmed that new EPDM type elastomer seal can be used up to Dose=1.2 MGy.



Layout of Cryogenic Components

Magnet String & Transfer Line

**Inventory: 3900 l,
Cold mass: 225 ton(Fe)**

**3 Recovery Tanks
(for Quench)
Volume 100 m³×3**

Cold Box & Sub-cooler

**SHe Max 300 g/sec at 4.5 K
LHe reserver : 800 l**

**Main compressor(MCP)
550 kW
Discharge pressure:1.5 MPaAbs**

**Buffer Tank for Main Compressor
(steady state)
Volume 100m³×1**

**LN2 > 20000 l
Only precooling and
recooling after quench
18000 l/day
For first heat exchanger
(cold box)**

Quench Release Analysis

OKAMURA Takahiro
takahiro.okamura@kek.jp

Contents

0. Motivation of this work
1. Conceptual diagram of quench release
2. Analytical model & Method
3. Highlight numerical result
4. Summary

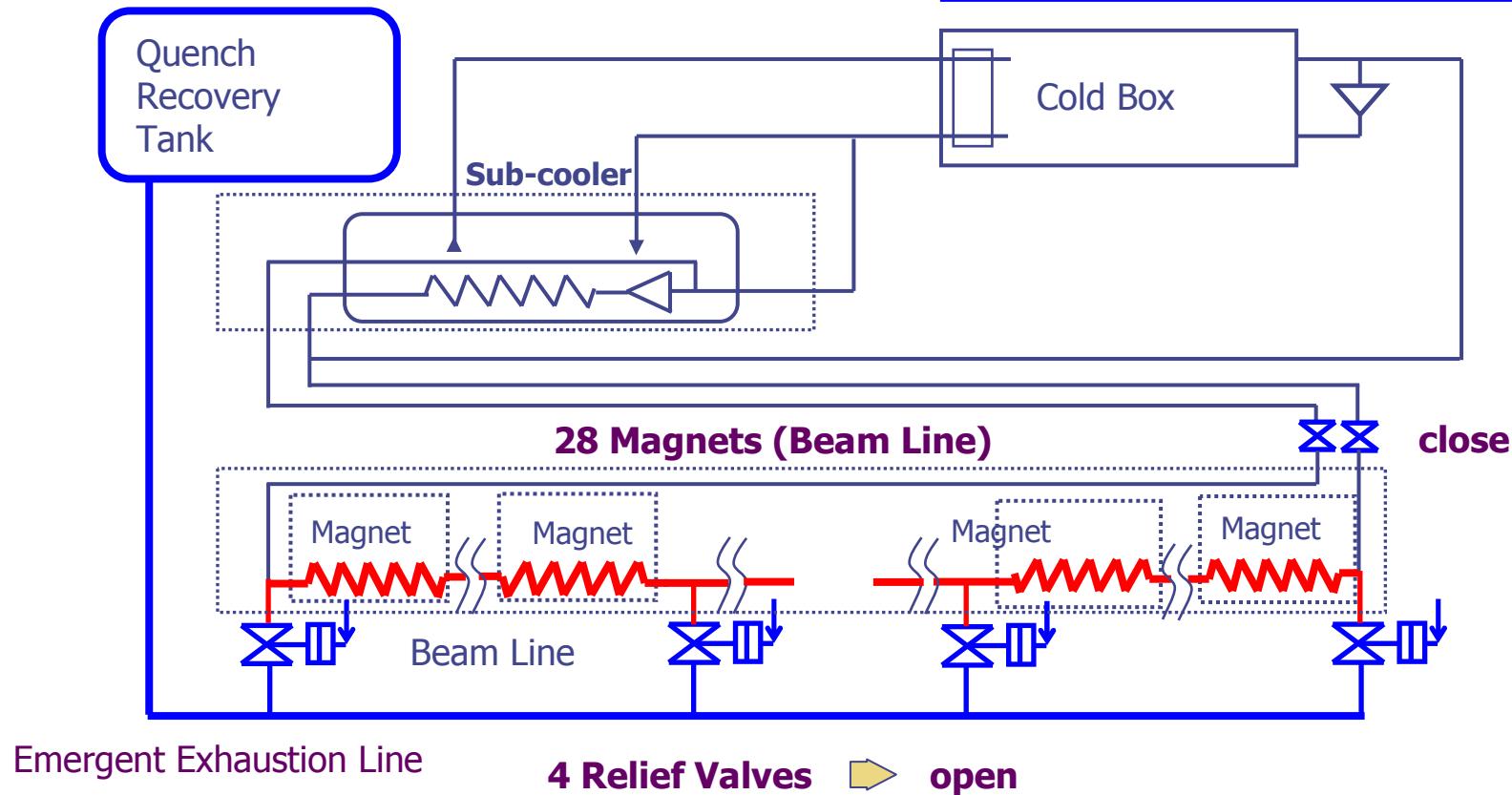


Conceptual diagram of quench release

Allowable pressure of SC magnet system: 2.0 MPa

**Essential qualification of He gas
Max Pressure of He gas < 2.0 MPa**

**Determination
of Geometric Parameter
I.D. of Relief Valve**





Simulation Model

Emergent Exhaustion (a)

I.D: 100 mm, Length: 1.5 m

Emergent Exhaustion (b)

I.D: 250A, Length: 175 m

**Dirichlet Type
B.C.**

**Quench
Recovery Tank**

25A 28.0 mm
50A 53.5 mm
90A 101.6 mm
250A 255.4 mm

Relief Valve
I.D: 25 A, L:120 mm

Interconnect (a)
I.D: 50A
L: 770 mm

**Interconnect
(b)**
I.D: 50A, 90A,
L: 1910 mm

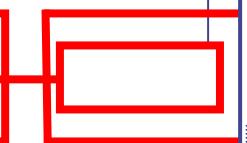
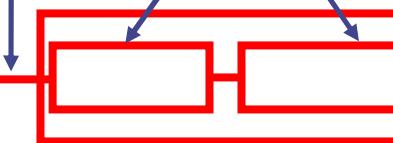
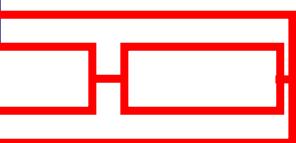
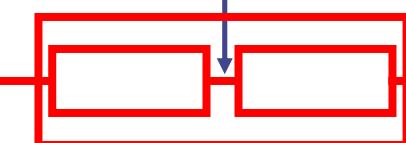
Neumann Type B.C.

Magnets

(1)

(2)

(3)



Cryostat

Cryostat

7 Magnets + Interconnect □ 35 m



Governing Equations & Method

Continuity Equation □

$$\frac{\partial}{\partial t}(\rho A) + \frac{\partial}{\partial x}(\rho u A) = 0$$

Momentum Equation □

$$\frac{\partial}{\partial t}(\rho u A) + \frac{\partial}{\partial x}(\rho u u A) = -\frac{\partial}{\partial x}(pA) + G(u)$$

Energy Equation □

$$\frac{\partial}{\partial t}(\rho e A) + \frac{\partial}{\partial x}(\rho e u A) = -p A \frac{\partial u}{\partial x} - \frac{\partial}{\partial x}(q A) + \dot{Q}$$

Thermal Equation of State □

$$de = \left(\frac{\partial e}{\partial p} \right)_\rho dp + \left(\frac{\partial e}{\partial \rho} \right)_p d\rho$$

Mass, Momentum, Energy are directly coupled.
(Semi-Implicit Pressure Based Scheme)

Solving Pressure Poisson equation

Finite Difference Method (FDM)+FCT Algorithm

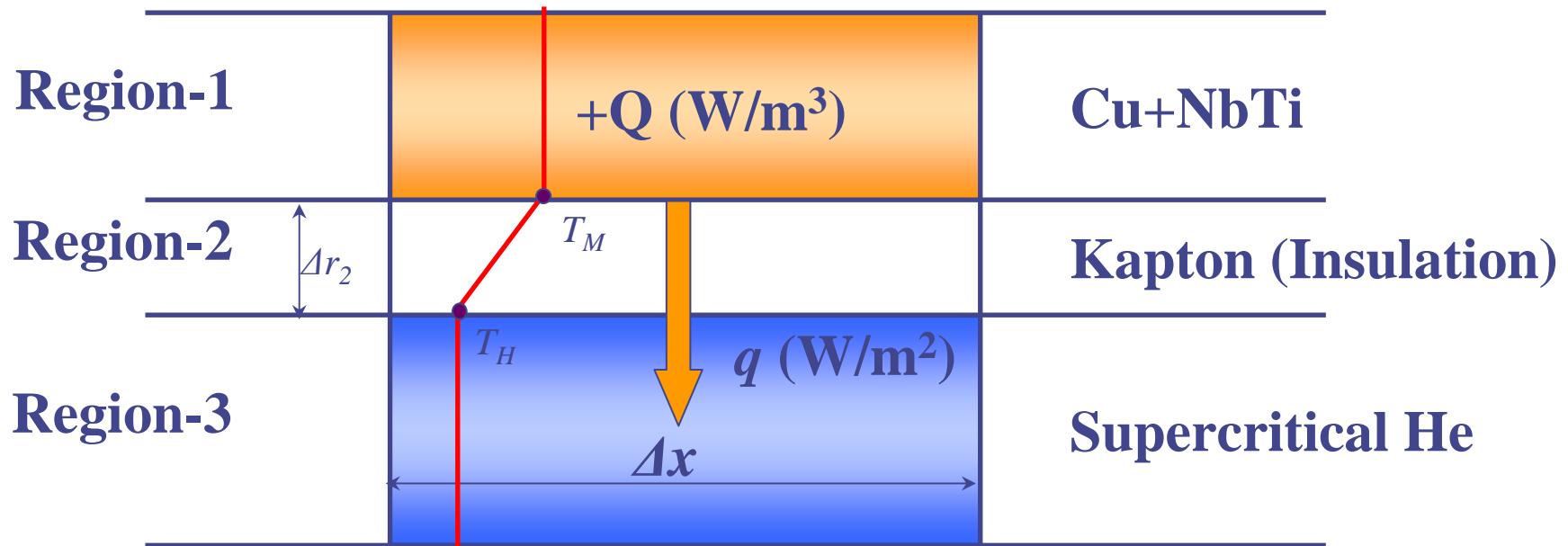
Heat generation term from Magnet.

Most Important Term !!?

1-Dimensional Heat Transfer Model



Heat Transfer Model



Region-1: Energy Balance Equation

$$\rho c_p \frac{\partial T_M}{\partial t} \Delta V = Q \Delta V - q \Delta S , \quad q = \lambda_K \frac{T_M - T_H}{\Delta r_2}$$

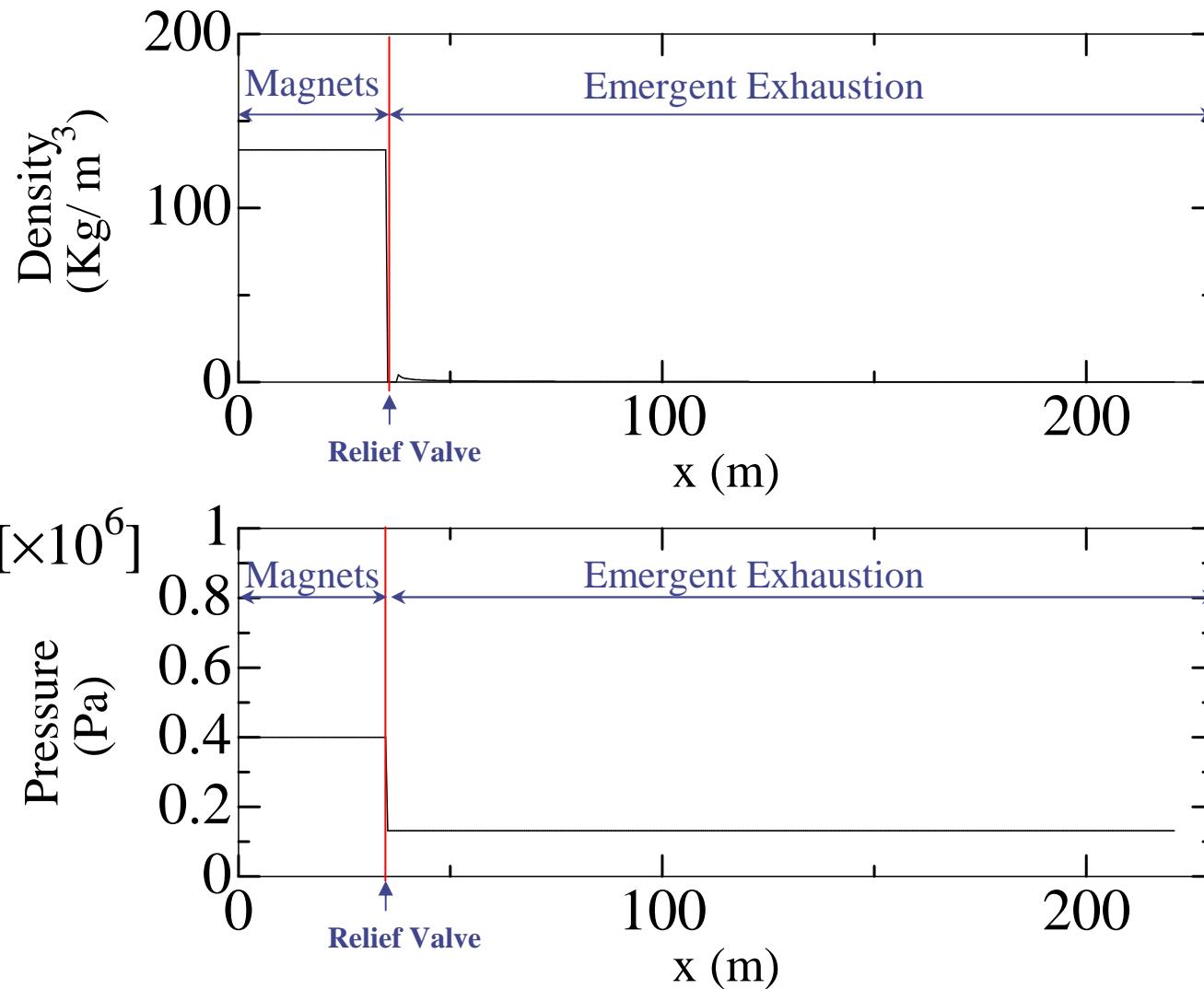


Heat Generation term
from Magnet to SHE

$$\dot{Q} = \frac{q \Delta S}{\Delta x}$$

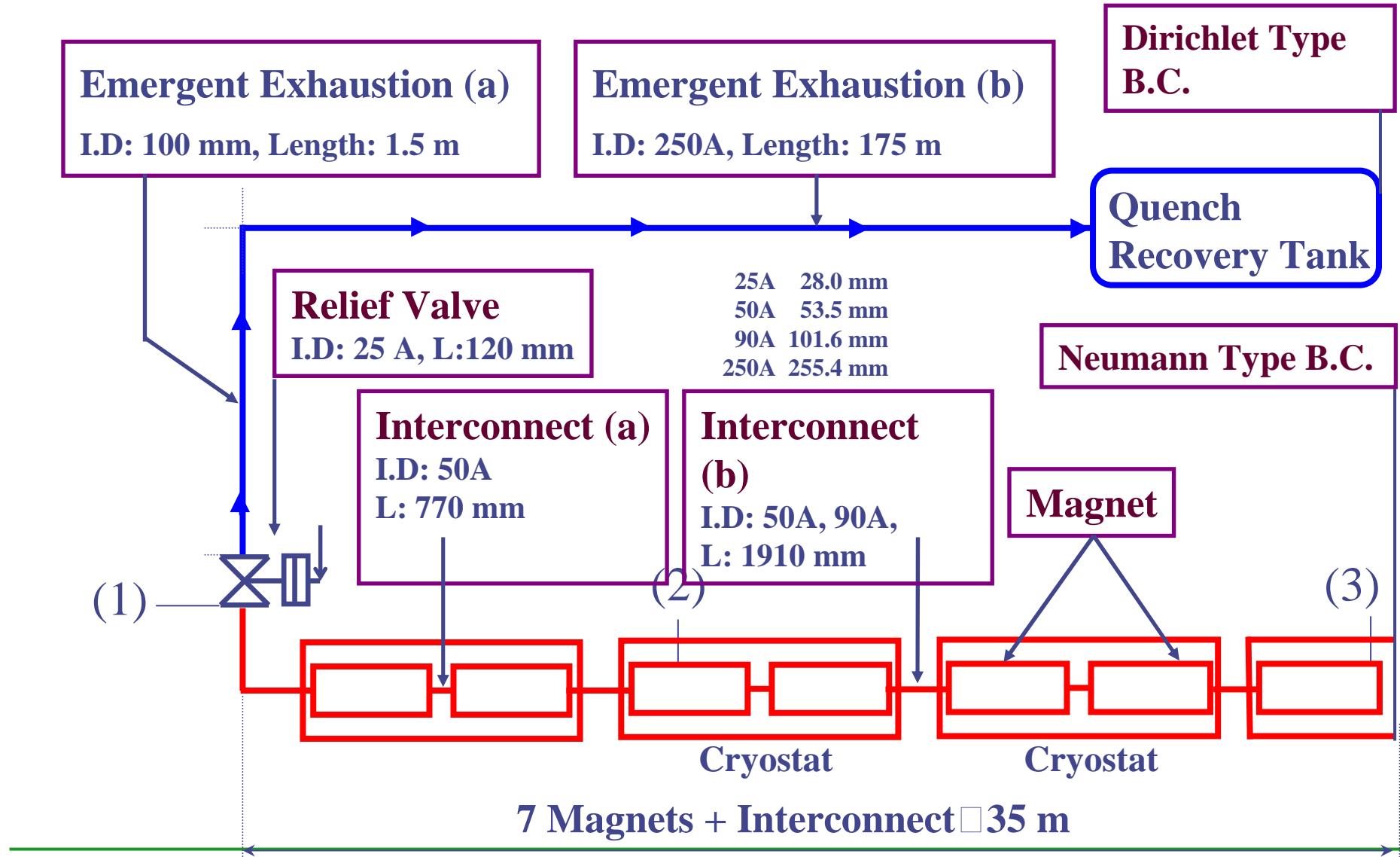


Initial Conditions



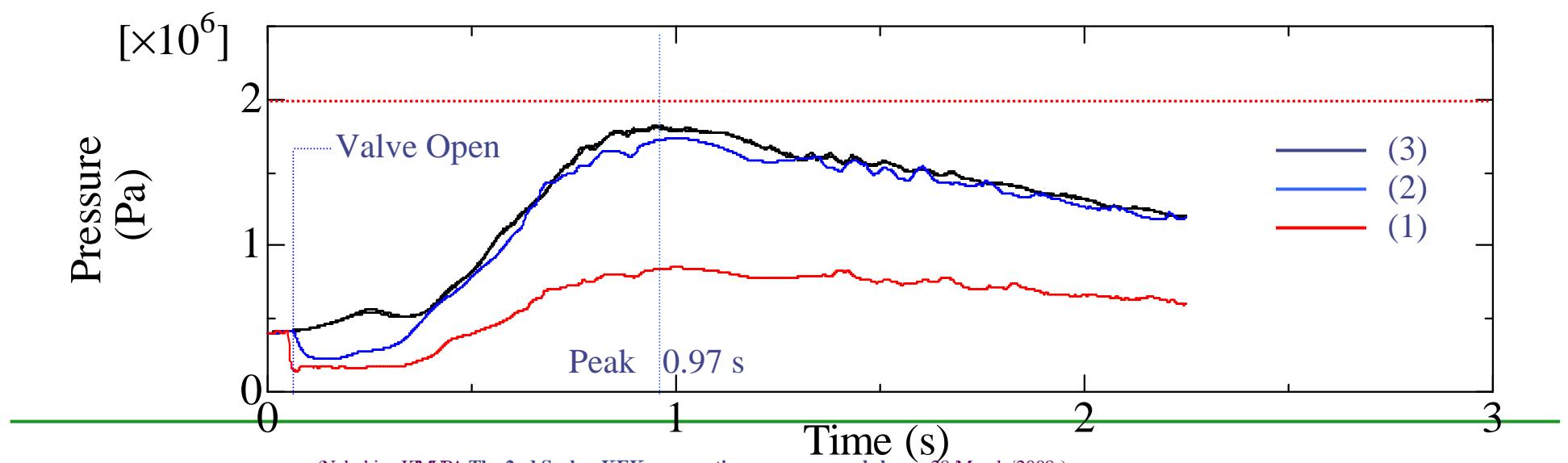
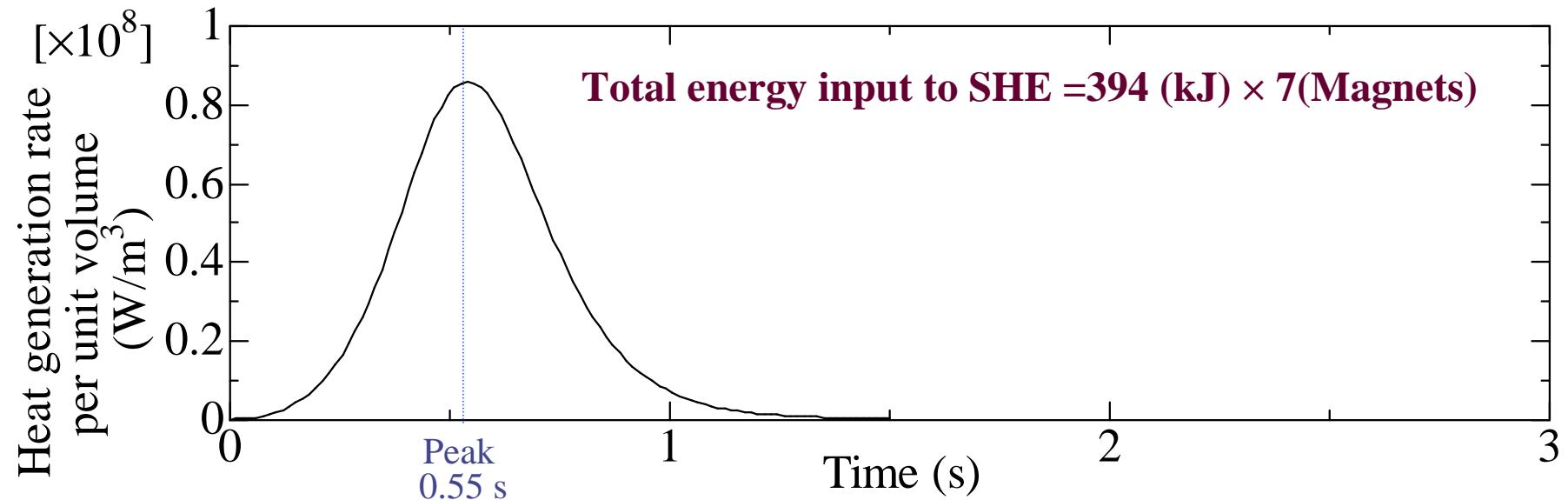


Simulation Model



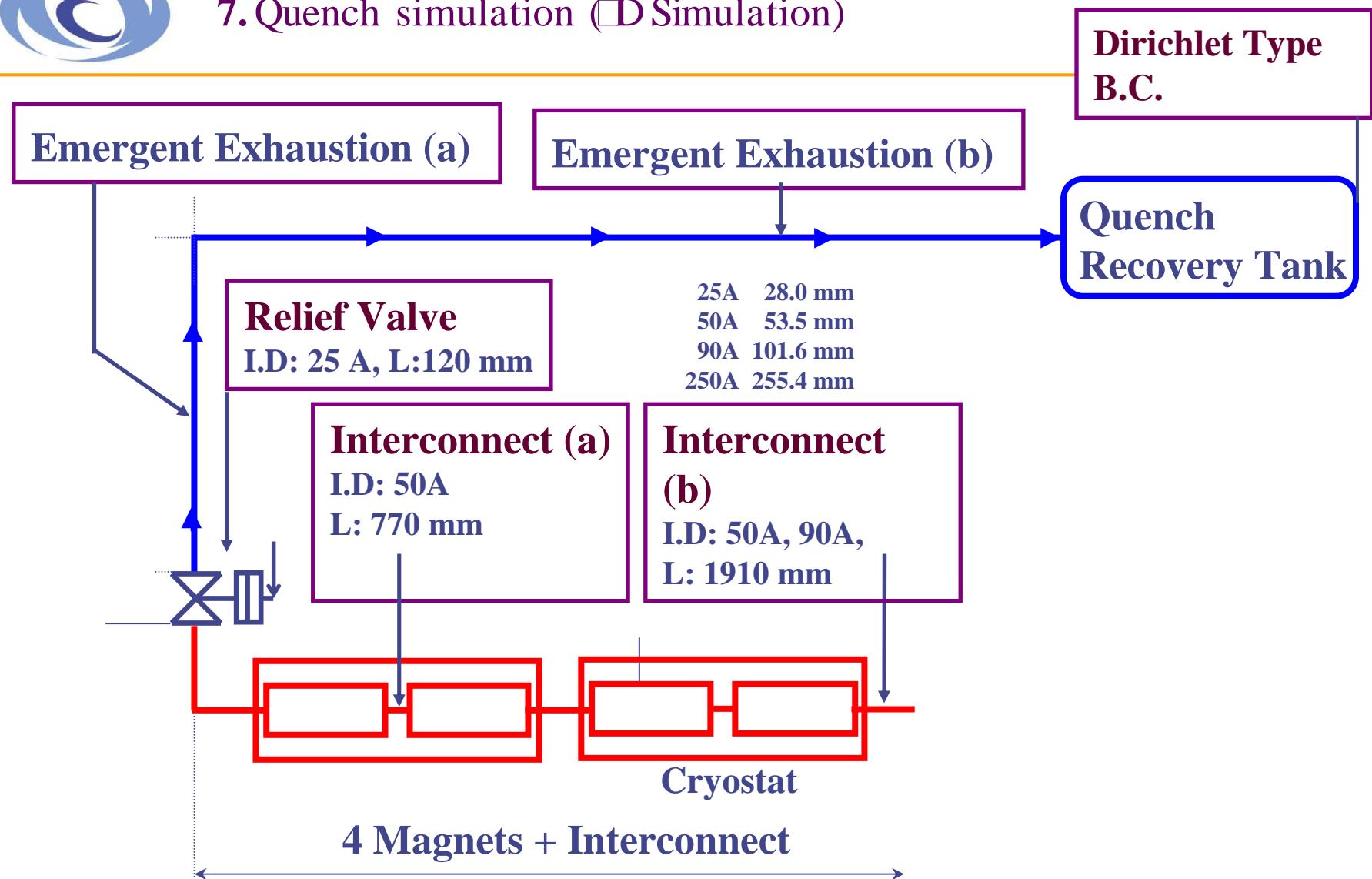


Highlight Result





7. Quench simulation (2D Simulation)



This model is based on an assumption that flow is Two-dimensional
Numerical simulation is carried out involving four magnets, one relief valve, venting line and buffer tank.



Simulation Method

- **E.Q.: NSE+Equation of State**

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

$$\frac{\partial}{\partial t}(\rho \mathbf{v}) + \nabla \cdot (\rho \mathbf{v} \mathbf{v}) = -\nabla p + \nabla \cdot \boldsymbol{\tau} + \rho \mathbf{g}$$

$$\frac{\partial}{\partial t}(\rho e) + \nabla \cdot (\rho e \mathbf{v}) = -\nabla \cdot \mathbf{q} - p(\nabla \cdot \mathbf{v}) + \boldsymbol{\tau} : \nabla \mathbf{v}$$

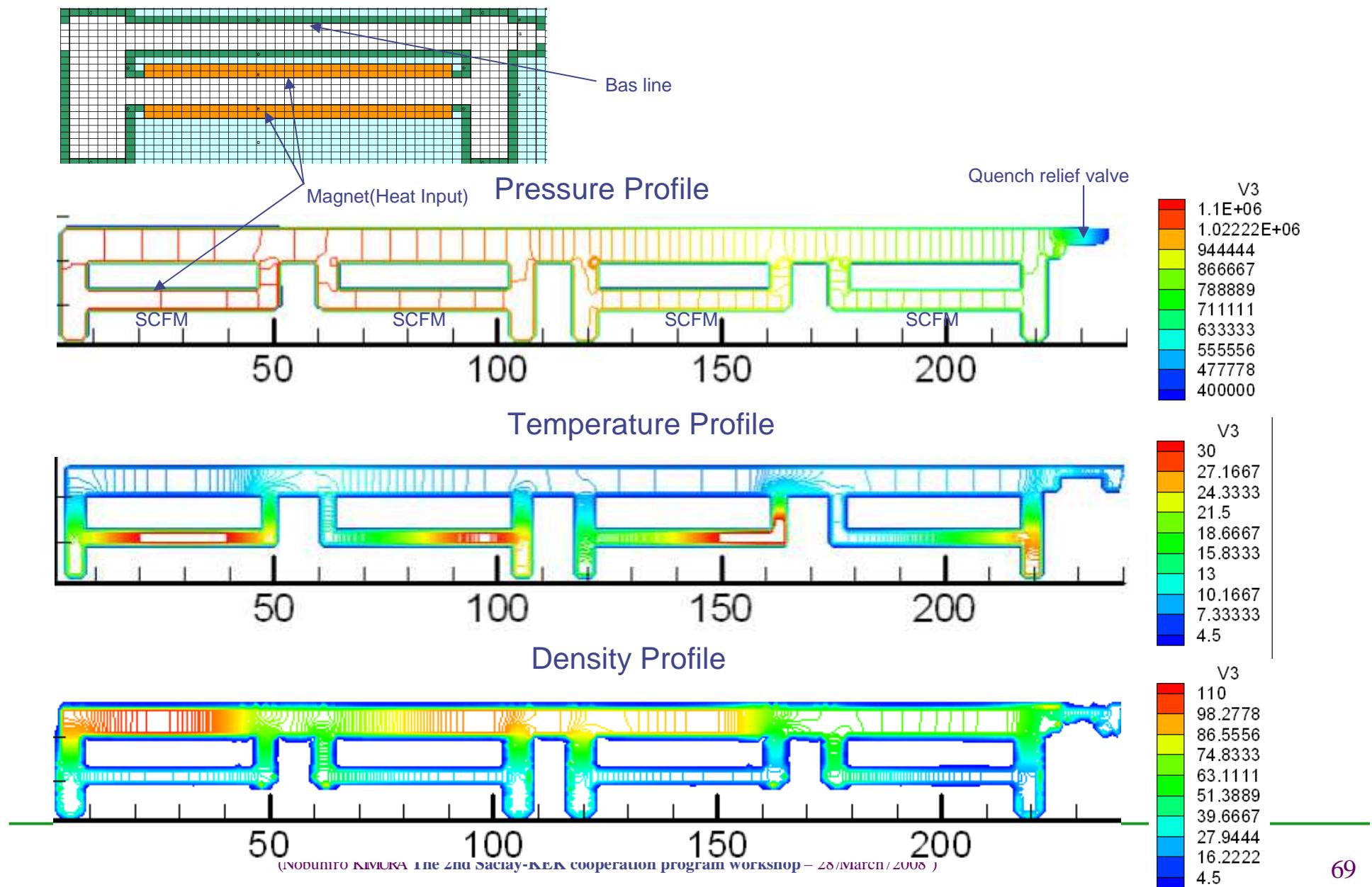
$$de = \left(\frac{1}{\varphi \rho} \right) dp - \left(\frac{c^2}{\varphi \rho} - \frac{p}{\rho^2} \right) d\rho$$

where
$$\begin{cases} \boldsymbol{\tau} = \mu \left\{ \nabla \mathbf{v} + (\nabla \mathbf{v})^T \right\} - \frac{2}{3} \mu (\nabla \cdot \mathbf{v}) \mathbf{I} \\ \mathbf{q} = -\lambda \nabla T \end{cases}$$

- **Method: FVM+P1 element based scheme**
- **Coordinate: BFC**



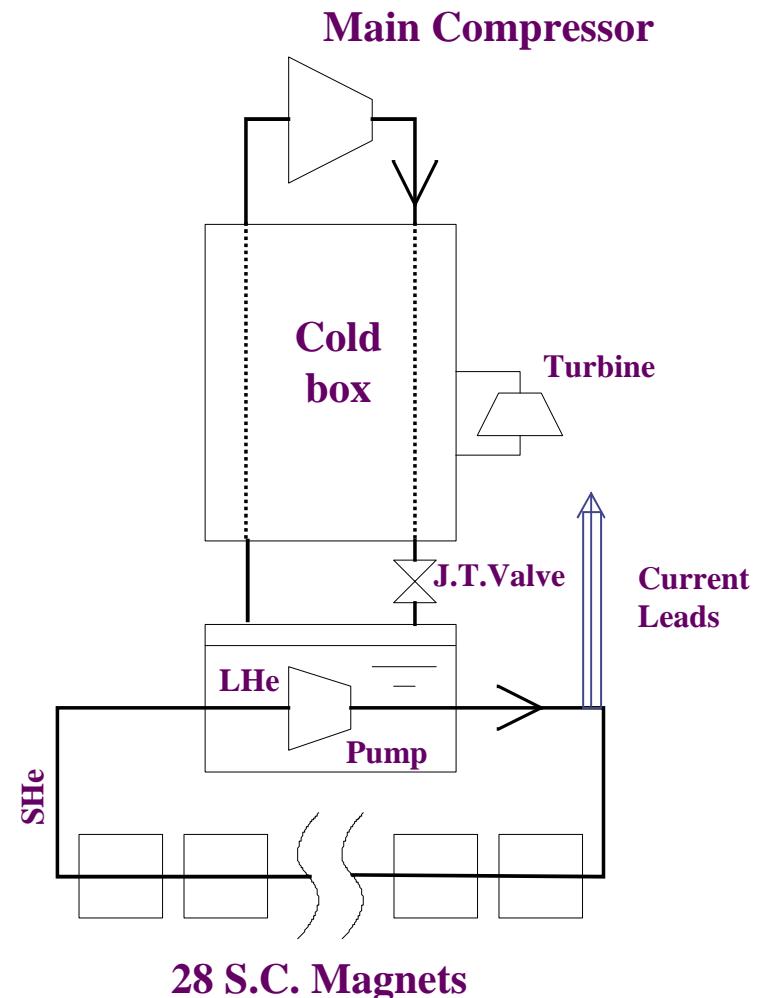
7. Simulation Results at 4/4 Magnets Quench





Required Cooling Capacity

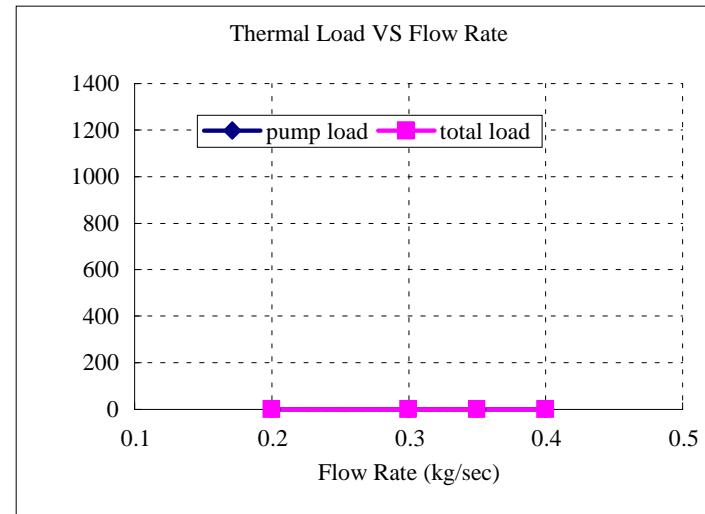
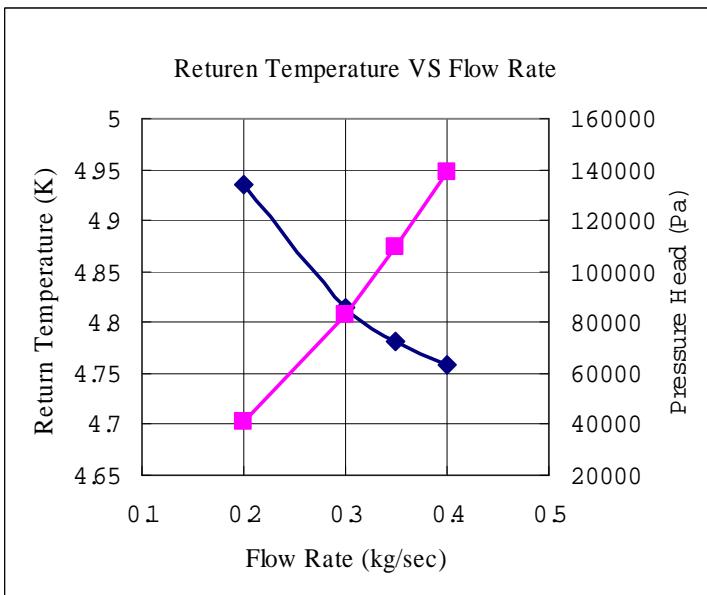
SHe Flow Rate	max 300 g/s
SHe Condition	0.4 MPa(A), 4.5 K
SHe Return	4.9 K
Thermal Load to SHe Flow	410 W
Pressure Head of SHe	85 kPa
Current Lead cooling gas	1.1 g/s (1 pair)
Shield Temperature	60–100 K
Shield Cooling	Cold Helium Gas
Thermal Load to Shield Line	1710 W
Shield Cooling Gas Condition	Not specified
LN2 usage	Only Pre-cooling and re-cooling after quench
Pre-cooling duration	< 20 days
Re-cooling duration	<6 hours (30GeV operation)



Schematic diagram of SHe circulation system



Required Cooling Capacity - Estimation



$$\text{PressureHead} = f \frac{L}{D_h} \frac{\rho u^2}{2}$$

f : Friction Coefficient, L : Length,
 D_h : Hydraulic Diameter, u : Flow Velocity

Expected Operational Flow Rate :
300 g/s Pump Load : < 300 W
Mag. Temp. : ~ 4.8 K

$$\begin{aligned} \text{TotalLoad} = \\ [\text{Mag \& Trans.T} + 20\%] \\ + [\text{PumpLoad}] \\ + [\text{Sub-coolerLoad} : 143W] \end{aligned}$$

$$\text{PumpLoad} = \frac{\Delta P M}{\rho \eta}$$

Mass-flow rate is controlled to be 300 g/s at the maximum.



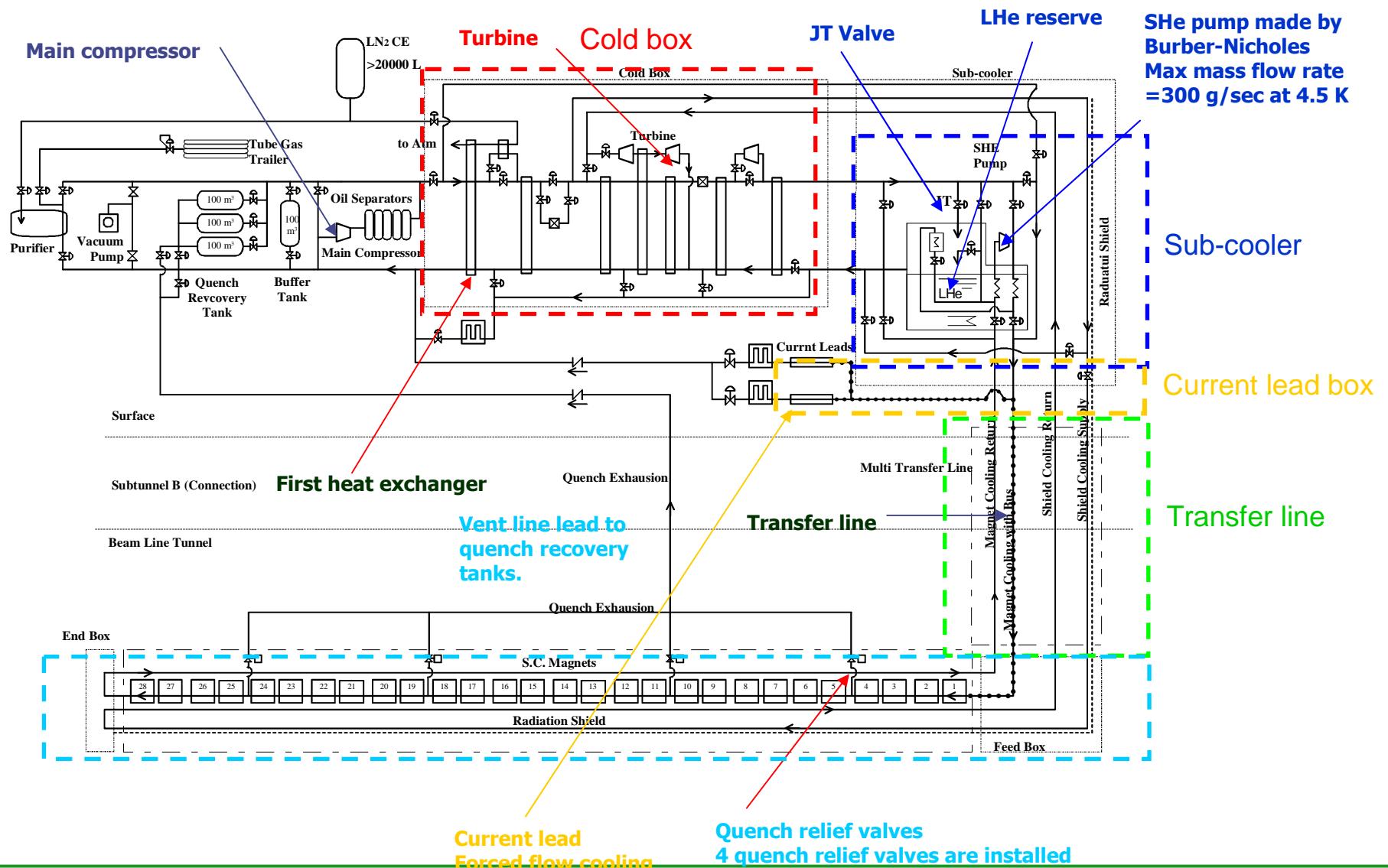
Required Refrigeration Capacity – Design by Contractor

		Thermal Load @4.5 K Level	Thermal Load @shield Level
KEK Requirement	Magnet & Transfer Line	410 W + 1.1 g/s	1710 W
	SHe Flow conditions	Max 300 g/s, 4.5 K, 0.4 MPa Pressure Head 85 kPa	
Contractor Design	SHe Pump Load	330 W	
	Sub-cooler, Transfer Line b/w CB	150 W	250 W
	Required Refrigeration	890 W + 1.1 g/s → 1.0 kW	1960 W → 2 kW
	+ 20 % Margin	1.2 kW	2.4 kW

Taiyo-Nissan Co. in the business collaboration with LINDE won the bid.



Conceptual Flow Diagram





Summary

- A new Implicit Continuous-fluid Eulerian code for SHe venting simulation has been developed by means of 1 & 2 Dimensional Heat transfer model.
- Maximum Pressure is about 1.8 MPa and lower than allowable pressure of the magnet under the present relief valve and emergent exhaustion line design conditions.



Heat Load of the Cryostat

